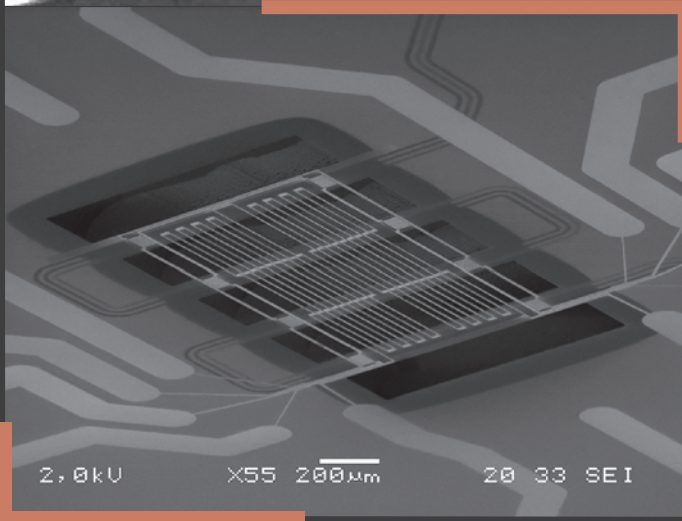
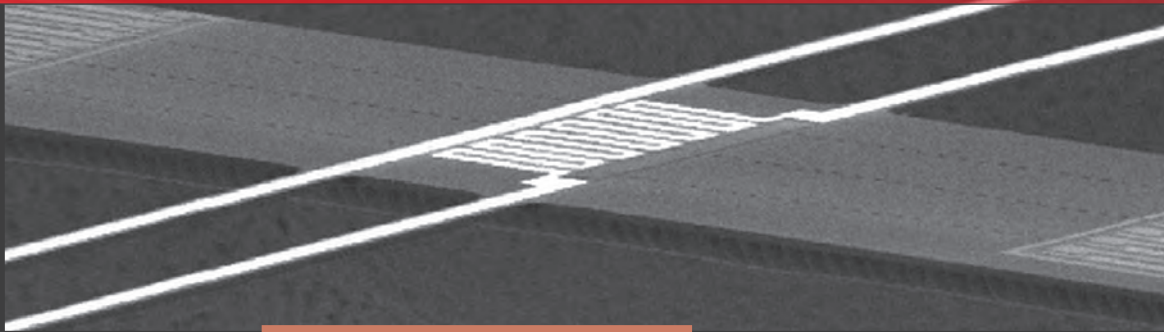


Low-Drift Micro Flow Sensors



MARCEL DIJKSTRA

LOW-DRIFT MICRO FLOW SENSORS

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The research described in this thesis was carried out at the Transducers Science & Technology group at the MESA⁺ Institute for Nanotechnology at the University of Twente, Enschede, The Netherlands. The project was financially supported by the Dutch Technology Foundation (STW).

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Micrographs of Ambient Temperature-Gradient Compensated Flow Sensors (Chap. 7).

Power Spectral Density of Low-Drift Flow Sensor Response (Fig. 5.11).

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LOW-DRIFT MICRO FLOW SENSORS

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1

Introduction

The research described in this thesis concerns the development of low-drift micro flow sensors for the accurate measurement of minute amounts of liquid flow. In this chapter an introduction is given on micro flow sensors, with emphasis on thermal flow sensors applicable to microfluidic systems. The main aim of this work is to increase the accuracy of microfluidic thermal flow sensors. The chapter ends with an outline of the scientific work presented, which has led to the realisation of low-drift micro thermal flow sensors, with compensation for external temperature gradients.

1.1 General Introduction

Flow sensors find applications in many areas in industry. Applications range from motorcars, process industry, analysis and synthesis in chemistry, pharmacy, biology and medicine. The emerging fields of micro total-analysis systems (micro-TAS), micro reactors and bio-MEMS drives the need for further miniaturisation of flow sensors capable of measuring minute amounts of liquid flow. Miniaturisation has intrinsic advantages such as high speed, small amounts of fluid required for analysis and portability, but perhaps more importantly fluidic components are being integrated in whole systems. The control of these systems is only possible with sensors measuring quantities such as pressure, temperature and flow. The need for small and reliable sensors makes flow sensing an important application in micro systems technology.

Micro high-pressure liquid chromatography (micro-HPLC) and other micro-TAS applications require liquid flow-rate resolution down to the $\text{nl}\cdot\text{min}^{-1}$ range. Advancements in nanofluidics demand an even further increase in flow-rate resolution. This can only be achieved with the development of highly-sensitive micro flow sensors. These sensors should make optimum use of the liquid flow, while applying a transduction principle with maximum signal-to-noise ratio. Resolution can be increased further by proper electronic amplification of the sensor signal, while reducing noise influences. Ultimately, external disturbances and drift of transducer properties have to be compensated for, or otherwise drift will render the flow sensor useless for accurate determination of flow rates in the $\text{nl}\cdot\text{min}^{-1}$ range and below.

1.2 Micro Flow Sensors

The development of instrumentation for the measurement of fluid flow has a long history dating back to ancient times. A whole range of flow sensing principles has since been studied by a great many scientists in not more than the last four hundred years [1.1]. Many of these principles are applied in the miniaturisation of flow sensors

[1.2-1.4], but not all principles are equally suited for downscaling and some find difficulties in microfluidic system integration.

Mechanical Flow Sensors

One of the most direct ways in measuring fluid flow is by utilising the mechanical drag force imposed by a moving fluid on a structure. This principle has been adopted by nature, where by means of natural selection [1.5] highly-flow-sensitive mechanoceptive hairs have evolved in e.g. crickets, which are yet unrivalled by intelligent designs (Krijnen et al. [1.6-1.8]). Figure 1.1a shows biomimetic sensory hairs made from SU-8, where electrodes are integrated for the capacitive measurement of low frequency sound waves [1.6]. Other types of mechanical flow sensors include lift-force and

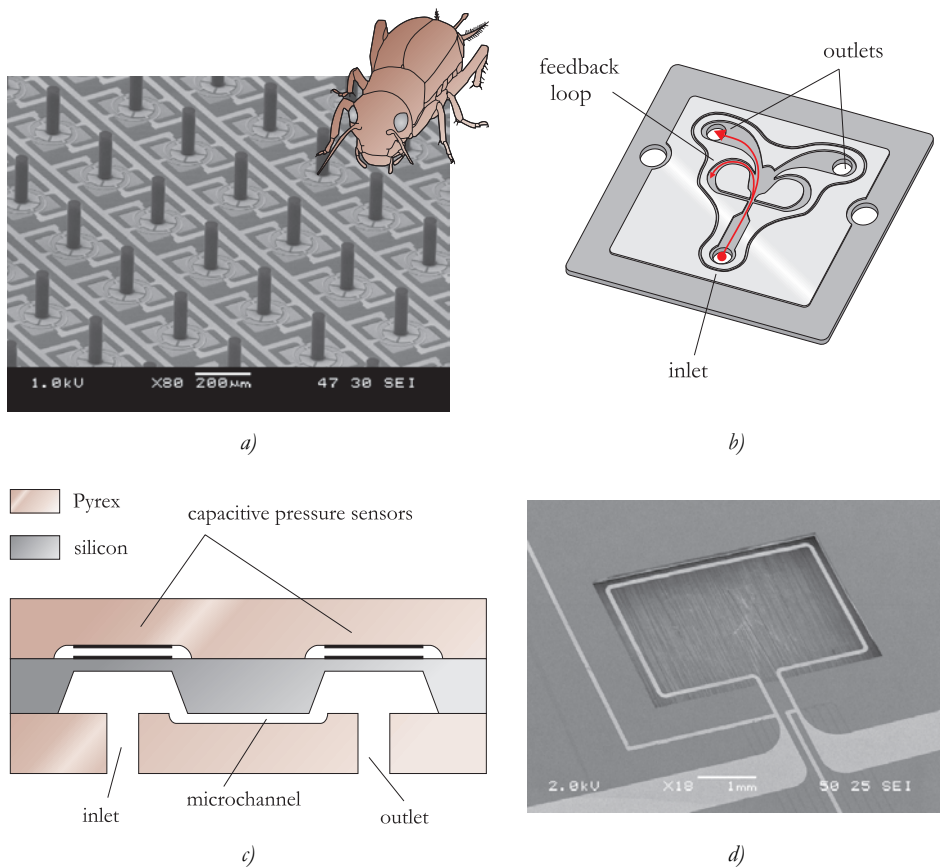


Fig. 1.1 Micro flow sensors based on a) drag-force by Dijkstra et al. [1.6], b) the Coanda effect by Gebbard et al. [1.10], c) differential-pressure by Oosterbroek et al. [1.15] and d) the Coriolis effect by Haneveld et al. [1.20].

skin-friction sensors [1.2]. Recently, Attia et al. [1.9] determined liquid flow rate by measuring the extension of a spring structure contained in a microfluidic channel. Flow rates down to $\text{nl}\cdot\text{min}^{-1}$ could be obtained through optical microscope inspection. A fully functional flow sensor has not yet been demonstrated.

Oscillatory Flow Sensors

Extremely accurate flow sensing can be obtained using fluidic oscillators [1.10-1.13]. Figure 1.1b shows a Coanda flow meter fabricated in the LIGA process by Gebhard et al. [1.10], where an oscillating fluid jet is generated between two outlets. The fluid flows to one of the outlets, due to the Coanda effect. Switching occurs by a feedback loop, which counteracts the Coanda effect. The frequency of oscillation ω is a measure of the flow velocity v and is governed [1.12] by the Strouhal number $S_r = \omega D_b / v$, being constant or linearly dependent on the Reynolds number $Re = v D_b / \nu$, with D_b the hydraulic diameter and ν the kinematic viscosity of the fluid. The oscillation frequency can be picked up by e.g. pressure or thermal transducers in the feedback loops. Coanda flow sensors are used as highly stable reference flow meters in the gas industry [1.13], because the oscillating digital output of the sensor allows transducer properties to drift over time, while retaining sensor accuracy. However, Coanda flow sensors are not suitable for miniaturisation as the Strouhal number becomes dependent on a decreasing Reynolds number and oscillations finally cease because of viscous damping [1.14].

Differential-Pressure Flow Sensors

Efficient integration with a microfluidic system can be obtained with differential-pressure flow sensors [1.2]. Oosterbroek et al. [1.15, 1.16] have demonstrated measurement of ethanol in the $\mu\text{l}\cdot\text{s}^{-1}$ range by measuring the pressure drop over a microchannel using capacitive pressure sensors (Fig. 1.1c). The resolution of the sensor can possibly be improved, applying more sensitive pressure sensors and by increasing the hydraulic resistance of the microchannel, which constitutes in the largest bottleneck for the flow.

Coriolis Flow Sensors

Coriolis flow sensors make use of the Coriolis force $\vec{F}'_c = 2\vec{\phi} \times \partial_t \vec{\theta}$, acting along a vibrating tube $\partial_t \vec{\theta}$, guiding a mass flow rate ϕ . Enokson et al. [1.17] were the first to develop a micromachined Coriolis flow sensor, followed by Sparks et al. [1.18]. The Coriolis effect is not directly suitable for downscaling. However, Mehendale [1.19] demonstrated that with an intelligent large-scale sensor design mass flow rates in the $\text{g}\cdot\text{h}^{-1}$ range ($3.3 \text{ ml}\cdot\text{min}^{-1}$ for water) can be measured.

Figure 1.1d shows the realisation of a Coriolis resonance tube by Haneveld et al. [1.20]. A fully-functional Coriolis flow sensor was created by Haneveld et al. [1.21], using integrated comb-like capacitive-readout electrodes and Lorentz-force actuation. Liquid flow rates up to $20 \mu\text{l}\cdot\text{min}^{-1}$ have been measured with $0.4 \mu\text{l}\cdot\text{min}^{-1}$ resolution.

1.3 Micro Thermal Flow Sensors

Most sensing principles described in Sec. 1.2 have gradually resulted in flow sensors with higher sensitivity. However, micro thermal flow sensors are among the most sensitive flow sensors for liquids to date. The most sensitive sensors are obtained with fully-heated freely-suspended microchannels. These micro thermal flow sensors can furthermore readily be integrated with other microfluidic components.

1.3.1 External Forced-Convection Sensors

Micromachined thermal flow sensors rely on the interaction of a fluid flow with heat produced by on-chip integrated heaters. In many cases the flow sensor chip is positioned directly in the flow or the chip is integrated in the wall of a conduit. Sensor characteristics depend in both cases on fluid boundary layers.

This is described by the Prandtl-Blasius boundary layer solution [1.22, 1.23] for a flat surface positioned parallel to the flow. The flow velocity v stagnates on the surface, which results in a shear-stress boundary layer extending into the fluid. The boundary layer builds up in the direction of the flow (x -direction), starting from the edge of the flat surface. The boundary layer thickness δ , defined at the specific height above the flat plate where $v = 0.99 v_\infty$ is given by (1.1), with v_∞ the free-stream velocity and $Re_x = v_\infty x / \nu$ the Reynolds number depending on x . A thermal boundary layer (1.2) resides inside the fluidic boundary layer [1.22, 1.24], with the surface having an elevated temperature and with fluids α having a Prandtl number $Pr = \nu / \rho c$ larger than one, with κ the thermal conductivity, ρ the density and c the heat capacity of the fluid.

$$\delta = \frac{5x}{\sqrt{Re_x}} \quad (1.1)$$

$$\delta_{th} = \delta Pr^{-1/3} \quad (1.2)$$

The heat flux normal to the surface can be related to the local temperature gradient also normal to the surface. This is expressed in the Nusselt number $Nu_x = h_x x / \kappa$, with h_x the heat transfer coefficient. The relation (1.3) is proportional to the thermal boundary layer thickness as this determines the local temperature gradient, where λ is a constant specific to the sensor geometry [1.24, 1.25]. The forced convection by fluid

flow from an on-chip heater dissipating a fixed amount of power P results in a temperature change ΔT of the heater. This anemometric sensor response is described by King's law (1.4), where α determines the contribution by conduction and β determines the sensitivity to forced convection by the fluid flow.

$$Nu_x = \lambda Pr^{1/3} Re_x^{1/2} \quad (1.3)$$

$$\Delta T = \frac{P}{\alpha + \beta Re^{1/2}} \quad (1.4)$$

Thermal anemometers can e.g. be used to measure wind speed. Oudheusden et al. [1.26] measured temperature difference between thermopiles integrated at the edge of a square chip (Fig. 1.2a). The whole chip is elevated in temperature by dissipating heat in on-chip resistors. In this way, the wind direction and wind speed can be obtained from the thermal boundary layer [1.26-1.28]. A boundary layer is not present at low flow speed. Instead, a temperature disturbance linear with flow speed can be measured [1.22], which is the calorimetric regime of the flow sensor. The temperature difference can be measured with resistors [1.29, 1.30] or thermopiles [1.31-1.34].

Putten et al. [1.35] demonstrated that long-term drift in transducer properties can be eliminated by the alternating direction method, whereby the chip is rotated in the flow. This allows the distinction between sensor-drift and flow-velocity signals.

Brushi et al. [1.36, 1.37] used an operational amplifier to control the power between two heaters cancelling the voltage generated by two integrated thermopiles. The control power provides a measure for the flow speed, according to the temperature-

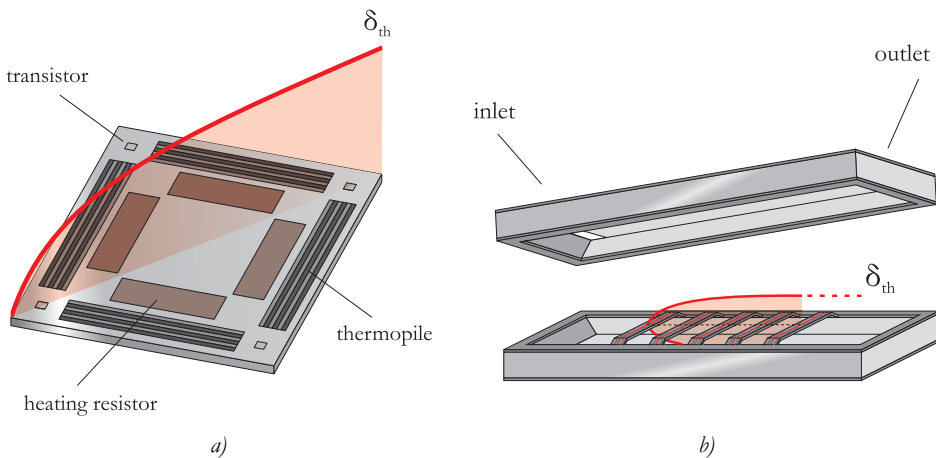


Fig. 1.2 Thermal flow sensors where thermal boundary layers define flow sensor characteristics, with a) wind direction flow sensor by Oudheusden et al. [1.26] and b) microchannel distributed thermal flow sensor by van Baar et al. [1.41].

balancing anemometer principle [1.38].

Microfluidic system integration requires sensor chips with integrated microchannels, such that the flow rate through the microchannel can be determined accurately [1.39, 1.40]. Van Baar et al. [1.41, 1.42] realised distributed sensing arrays on microbeams suspended in a microchannel created at the interface between two silicon chips (Fig. 1.2a). This means however that the fluid is in contact with the beams. The structures can be used for the determination of flow rate and other fluid parameters. The fluid boundary layer thickness δ is constant after the fluid becomes fully developed over the suspended beams. This means that the Nusselt number (1.3) is no longer dependent on position [1.43].

Meng et al. [1.44, 1.45] realised a similar sensing array, where heaters were integrated on top of a Parylene membrane suspended over a microchannel etched in a silicon substrate. Liquid flow rate down to $0.5 \mu\text{l}\cdot\text{min}^{-1}$ could be measured. In this type of microchannel flow sensors [1.39-1.45], the fluid never becomes fully heated, as it approaches the chip temperature close to the wall of the microchannel.

1.3.2 Internal Forced-Convection Sensors

Highest sensitivity can be obtained when the complete fluid is heated. This requires the integration of freely suspended microchannels, thermally isolated from the chip. With on-chip microchannel capillaries the fluid flow develops to a fully developed flow profile within the hydrodynamic entrance length X_b (1.5), where $Re_{D_b} = Q D_b / A v$ is the Reynolds number based on the average flow velocity Q/A and D_b the hydraulic diameter, which is equal to the capillary diameter D in case of a circular perimeter [1.46].

$$X_b = 0.05 Re_{D_b} D_b \quad (1.5)$$

A thermal entrance length $X_t = Pr X_b$ is defined for a capillary with an already established fully developed flow profile, where the fluid is at a different temperature than the wall or in case a wall heat flux is defined. The developing thermal profile along the capillary is expressed by the non-dimensional Graetz number Gz , where a thermal fully developed profile beyond X_t corresponds with $Gz < 16$.

For a cylindrical capillary with a $1 \mu\text{l}\cdot\text{min}^{-1}$ water flow rate Q the Reynolds number is $Re_D = 20 \mu\text{m}/D$ and the hydrodynamic entrance length X_b is $1 \mu\text{m}$. The thermal entrance length X_t for a uniformly heated capillary is $7 \mu\text{m}$ independent of the tube diameter D , but directly proportional to the flow rate Q . The thermal entrance length for micromachined flow sensors in the $\text{nl}\cdot\text{min}^{-1}$ range is therefore sufficiently short to be considered thermally fully developed over the complete length of the microchannel used for flow metering.

The forced convection in a thermally fully developed microchannel is determined by the total flow rate Q in the microchannel. The Nusselt number is constant, with an order of magnitude close to one for a cylindrical channel [1.46]. The radial temperature gradient is therefore independent of the Reynolds number, but directly proportional to the wall heat-flux. The temperature profile along the microchannel can be understood by determining the heat conduction through the whole sensor structure and surrounding air, taking into account the radial heat flux from the microchannel due to forced convection by a given flow rate Q [1.47]. The radial heat flux due to convection is again dependent on the gradient in the temperature profile along the microchannel.

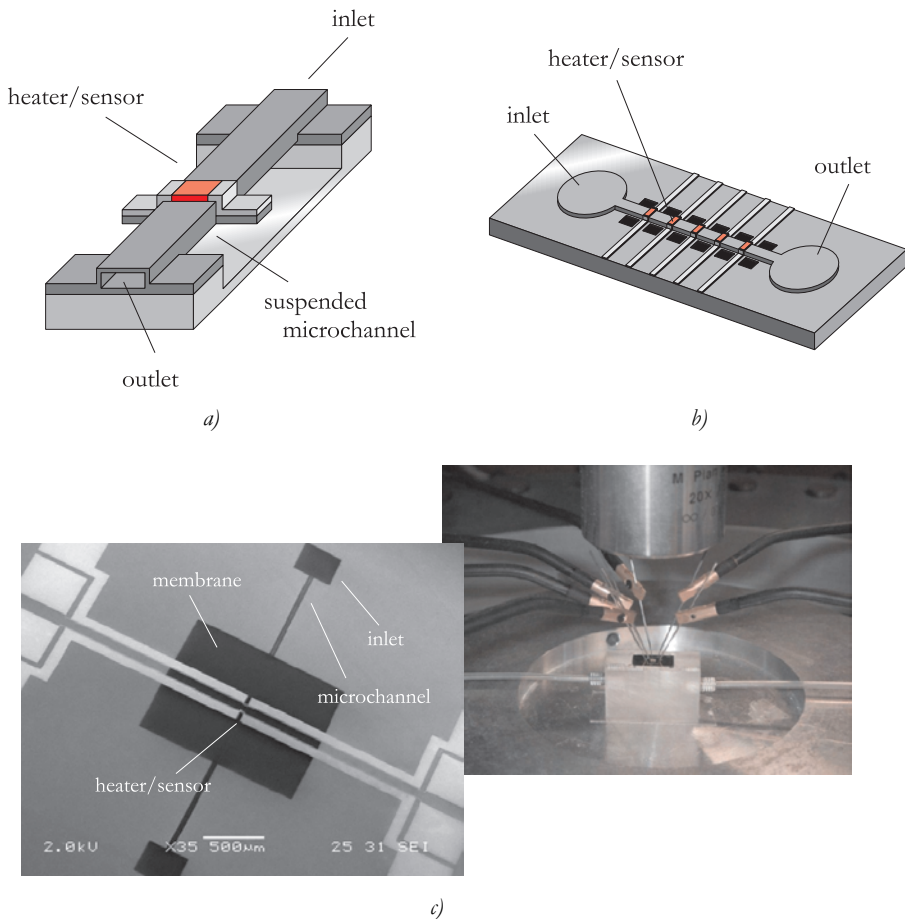


Fig. 1.3 Thermal flow sensors where thermally-fully-developed suspended microchannels are used to measure liquid flow, with a) nano-fluidic flow sensor by Wu et al. [1.48], b) nano flow sensor with carbon sensing elements by Mizuno et al. [1.49] and c) realised ω - 2ω nano flow sensor [1.50].

Wu et al. [1.48] were the first to use a suspended $2\ \mu\text{m}$ high microchannel over an etched cavity (Fig. 1.3a) for thermal flow sensing. Flow rate resolution down to $4\ \text{nl}\cdot\text{min}^{-1}$ for water was obtained with anemometry, using an integrated poly-Si heater/sensor. Mizuno et al. [1.49] used high-temperature sensitive carbon doped Parylene resistors (Fig. 1.3b) resulting in a higher flow rate resolution. Figure 1.3c shows a realised micro flow sensor [1.50], where a microchannel is suspended across a membrane. Heat waves with a ω - 2ω lock-in sensing method were used to measure water flow rate down to a few $\text{nl}\cdot\text{min}^{-1}$.

Freely-suspended-microchannel flow sensors [1.48-1.53] have been fabricated using sacrificial layer etching, where the height of the microchannel is limited by technology. The hydraulic resistance $R = \Delta p / Q$ of a rectangular microchannel R_{rect} [1.16] is significantly being determined by the microchannel height h , where w is the width and l the length of the microchannel and μ the dynamic viscosity.

$$R_{\text{rect}} = \frac{12\mu l}{h^3 w} \quad (1.6)$$

In the ω - 2ω sensor design, the hydraulic resistance R_{rect} is $29\ \text{bar}/\mu\text{l}\cdot\text{min}^{-1}$ for water flow through the microchannel (Fig. 1.3c), where the microchannel height is $1\ \mu\text{m}$, the width is $100\ \mu\text{m}$ and the length is $1.3\ \text{mm}$. This large hydraulic resistance is impractical for micro flow sensing applications.

Commercially available micro thermal flow sensors [1.54-1.57] (Fig. 1.4) in the $\text{nl}\cdot\text{min}^{-1}$ flow range make use of cylindrical capillaries made either of stainless steel, PEEK™ or fused silica. The hydraulic resistance R_{circ} (1.7) of the cylindrical capillary

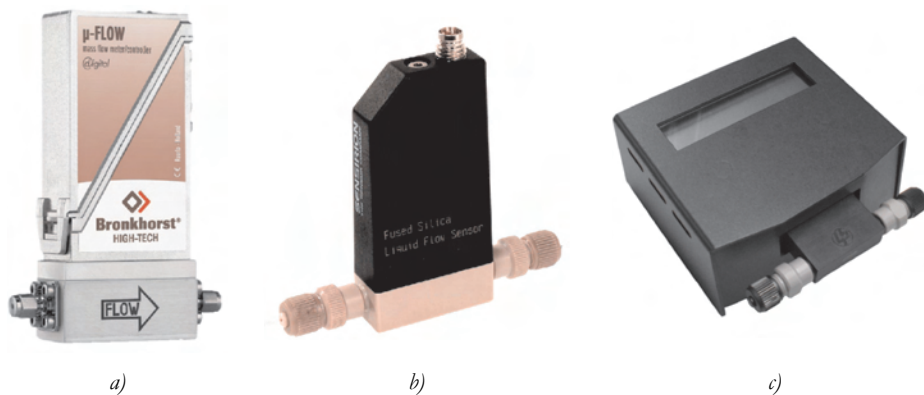


Fig. 1.4 Commercially available thermal flow sensors for measuring liquid flow in the $\text{nl}\cdot\text{min}^{-1}$ range, with a) μ -flow digital mass flow meter for liquids by Bronkhorst Nederland B.V. [1.54], b) SLG 1430 liquid mass flow meter by Sensirion AG [1.56] and c) nano flow sensor by Upchurch Scientific [1.57].

is dependent on the radius r of the capillary. Capillary radii range from 20 μm to 200 μm , therefore the pressure drop over these micro sensors remains relatively small.

$$R_{\text{inv}} = \frac{8\mu l}{\pi r^4} \quad (1.7)$$

The micro flow sensor from Sensirion AG (Fig. 1.3b) and Upchurch Scientific (Fig. 1.3c) are based on MEMS technology. The Sensirion AG sensor applies a MEMS based sensor chip mounted on the capillary tube [1.56]. A similar approach is followed by Weiping et al. [1.58].

1.4 Aim of the Research

The problem that is investigated in this thesis is related to performance limitations of thermal micro flow sensors due to miniaturisation. Miniaturisation means that flow channel dimensions and flow rates become smaller. This requires thermally-isolated flow channels where the complete fluid can be heated in order to obtain maximum sensitivity. Furthermore, the pressure drop across the flow channels increases significantly with miniaturisation, where the best possible solution is to use circular flow channel cross-sections.

With miniaturisation also sensor elements become smaller. In current micro flow sensors these elements are made by metal thin films on top of thermally-isolated flow channels. The problem is that thin films reproduce poorly and that practically all materials properties are subject to drift. This drift and poor reproducibility translates directly into the accuracy of thermal micro flow sensors.

The work presented in this thesis aims at solving material drift problems by a combination of two innovations. One innovation is to use power control on heater elements. By continuously measuring the resistance value the dissipated power can be precisely controlled, while resistance value can drift over time. The other innovation relates to the use of a temperature-balancing control system in combination with a thermopile measuring a differential temperature. Of importance is that the thermopile has zero-offset, not giving an output voltage if the thermopile is at a uniform temperature. Alternatively a sensor resistor and heat waves can be used to provide for a low offset-drift error signal (Chap. 8).

1.5 Outline of the Thesis

The content of this thesis is largely based on published articles or conference articles to be published in literature, therefore each chapter can be read by itself. In the next chapter (Chap. 2) design aspects of low-drift micro flow sensors are discussed, focussing on drift influences and flow sensing concepts to compensate for

drift. In the last chapter general conclusions are drawn, with recommendations for future research. The following sub-sections give a general outline of the research results set forth in this thesis.

Chapter 3 – Surface Channel Technology

In this chapter a microchannel fabrication concept is described, allowing for easy fluidic interfacing and integration of transducer material in close proximity to the fluid. This is achieved by the reliable fabrication of completely sealed microchannels directly below the substrate surface. The microchannels are subsequently released for thermal isolation. The viability of the concept was demonstrated by fabrication of several micro-fluidic device templates.

Chapter 4 – Miniaturised Calorimetric Flow Sensor

One of the device templates in Chap. 3 was applied in the fabrication of a calorimetric miniaturised flow sensor, with a linear sensor response measured for water flow up to flow rates in the order of $300 \text{ nl}\cdot\text{min}^{-1}$. The realised flow sensor consists of a microchannel with low hydraulic resistance and 4.5 nl total fluid volume. The sensor demonstrates the applicability of the microchannel technology for micro thermal flow sensor for measuring minute amounts of liquid flow.

Chapter 5 – Low-Drift Flow Sensor with Thermopile-Based Power Feedback

In this chapter a more advanced micro flow sensor is presented using two heaters and a thermopile in order to eliminate material drift. The low offset drift of the thermopile is exploited in a feedback loop controlling the dissipated powers in the heater resistors, minimising inevitable influences of resistance drift, mismatch of thin-film metal resistors and thermopile material drift. The control system cancels the flow-induced temperature difference across the thermopile by controlling a power difference between both heater resistors, thereby giving a measure of the flow rate. The flow sensor was characterised for power difference versus water flow rates up to $1.5 \mu\text{l}\cdot\text{min}^{-1}$. It is demonstrated that material drift is largely compensated, however the sensor still suffers from externally applied temperature gradients over the chip.

Chapter 6 – Low-Drift U-shaped Thermopile Flow Sensor

To compensate for external temperature gradient a different sensor layout is investigated, comprising of a freely-suspended U-shaped microchannel with integrated thermopile. The structure is symmetrically heated by a heater at the top of the U-shape. The thermal imbalance caused by liquid flow is sensed by the thermopile. The U-shape microchannel facilitates the integration of a large number of

thermocouple junctions, resulting in a highly-sensitive calorimetric flow sensor ($40 \text{ mV}/\mu\text{l}\cdot\text{min}^{-1}$ at 2 mW heating power). Accurate measurements up to $400 \text{ nl}\cdot\text{min}^{-1}$ water flow have been obtained applying a temperature-balancing control system.

Chapter 7 – Ambient Temperature-Gradient Compensated Low-Drift Flow Sensor

The U-shaped sensor presented in Chap. 6 shows a significant improvement in sensitivity, however a small dependence on external temperature gradients remains, caused by conduction through the surrounding air. In this chapter a special meandering layout of the microchannels is used, resulting in a fully symmetrical sensor. The thermopile junctions are placed, such that a fluid flow results in summation of thermopile voltages, while the influence of external temperature gradients is completely eliminated in the measured thermopile voltage.

Chapter 8 – AC-Driven Temperature-Balancing Flow Sensor with Power Feedback

In this chapter another measurement technique is explored to realise a low-drift sensor using AC-driven heat waves. A simple sensor structure similar to the sensor presented in Chap. 4 is used, with three resistors on a suspended segment of the thermal-isolated flow channel. The outer resistors are heated by an alternating current, while the addition of the heat waves arriving at the centre resistor is measured. In this way, a drift performance similar to the thermopile-based sensors is obtained at the expense of a much slower response time. This is caused by the low operating frequency of the lock-in technique in order to detect the small signal amplitudes.

Chapter 9 – Nano-Nozzle Electrospray Emitters Fabricated by a Micro- to Nano-Fluidic Via Technology

In this chapter the surface channel technology is extended by the possibility to integrate nanochannels using a micro- to nano-fluidic via technology. The main advantage of the technology is the ability to position freely-suspended nanochannels anywhere on a micro-fluidic chip. Nano-nozzle electrospray emitters were fabricated using this process on freely-suspended microchannels. Leak-tight delivery of fluid from a fluidic reservoir was established through long microchannels. Stable electrospray IV-curves could be obtained from fabricated nano-electrospray emitters.

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2

Low-Drift Flow Sensing

This chapter discusses design aspect of low-drift micro flow sensors. The accuracy of thermal micro flow sensors is influenced by the degradation of thin-film resistors, measured as flicker and electromigration noise, resulting in long-term drift of the sensor output. This makes thin-film resistors unsuitable as accurate absolute temperature sensors for micro flow sensors in the $\text{nl}\cdot\text{min}^{-1}$ range. Power control and four-point contacts can be used to accurately dissipate heat in the thin-film resistors. Thermopiles can be applied to generate a voltage proportional to a temperature difference, where a thermopile at uniform temperature has zero-offset. Additionally, sensitivity drift of thermal flow sensors can be eliminated by using the temperature balancing concept, where a power difference provides a measure of the flow rate.

2.1 Introduction

Thin-film metal and thin-film semiconductor resistors have been used as heating elements and as temperature sensing elements in micromachined thermal flow sensors for the $\text{nl}\cdot\text{min}^{-1}$ range [2.1-2.4]. Fabrication is simple, requiring deposition and patterning of a single thin-film layer. However, the resistance value of thin-film resistors are subject to degradation mechanisms and reproduce poorly. This chapter discusses the degradation mechanisms of thin-film resistors, where it can be concluded that accurate micro thermal flow sensors cannot be obtained using thin-film resistors as absolute temperature sensors. Instead thermopiles can be used to measure a differential temperature [2.5-2.8], where no output voltage is generated if the thermopile is at a uniform temperature. The temperature-balancing flow sensing concept is explained where power control on heater resistors and a thermopile are used in combination with a temperature-balancing controller to obtain a micro thermal flow sensor with high-accuracy and low offset drift.

2.2 Thin-Film Transducers

2.2.1 Thin-Film Resistors

Thin-film resistors used as absolute temperature sensing elements in micro-machined thermal flow sensors determine the temperature ΔT by measuring the resistance R change with temperature (2.1), where α is the temperature dependency of resistivity and R_0 the resistance defined at the reference temperature $\Delta T = 0$.

$$R = R_0 (1 + \alpha \Delta T) \tag{2.1}$$

A current flowing through the resistor is used to measure the voltage across the resistor and hence the resistance value is obtained. Additionally, the measured voltage contains Johnson noise, with power spectral density $S_V = 4kTR$, caused by thermal excitation of the charge carriers in the resistor, with k the Boltzmann constant and T the absolute temperature. Johnson or thermal noise is independent of frequency f . It determines the maximum precision in temperature ΔT obtained by averaging the resistance value of a resistor R , with given temperature dependency of resistivity α .

Materials in microengineering have many defects and irregularities at the microscopic level. Thin-film metal resistors are non-uniform and contain many grain-boundaries. The quality of the thin-film directly relates to the impact of degradation mechanisms (e.g. oxidation) on thin-film properties [2.9]. The degradation is measured in the voltage across the resistive material as $1/f$ or flicker noise, with a measurement current flowing through the resistor. The power spectral density S_V follows Hooge's formula (2.2), with V the voltage across the resistor, N the number of charge carriers and γ_H a constant related to the quality of the thin-film resistor. The constant γ_H can depend weakly on temperature, with α between 0.7 and 1.3 [2.9].

$$S_V = \frac{\gamma_H V^2}{N f^\alpha} \quad (2.2)$$

Noise spectra with higher α relate to diffusion mechanisms along the resistor [2.10]. This form of degradation is exponentially dependent on temperature as it depends on a diffusion coefficient D . Electromigration takes place if additionally the power spectral density finds relation with the current density j , with α close to 2 [2.11-2.13]. Electromigration occurs by a flux of atoms \bar{J}_{em} (2.3) due to a driving force \vec{F} , which is caused by the electron wind and the charge on the atom resulting in an effective charge number Z^* , with q the elementary charge and \vec{E} the electrostatic field [2.14]. The flux by electromigration \bar{J}_{em} is opposed by a diffusion flux \bar{J}_{diff} depending on the atom concentration N and diffusion coefficient D , with activation energy E_a (2.3).

$$\vec{J} = \bar{J}_{diff} + \bar{J}_{em} = -D \nabla N + \frac{ND}{kT} \vec{F} \quad D = D_0 e^{-E_a/kT} \quad \vec{F} = Z^* q \vec{E} \quad (2.3)$$

Figure 2.1 shows the power spectral density of an aluminium resistor where a $1/f^2$ slope due to electromigration is apparent. Electromigration noise (2.4) relates to the flux \bar{J}_{em} by electromigration and is both dependent on the current density j and on tem-

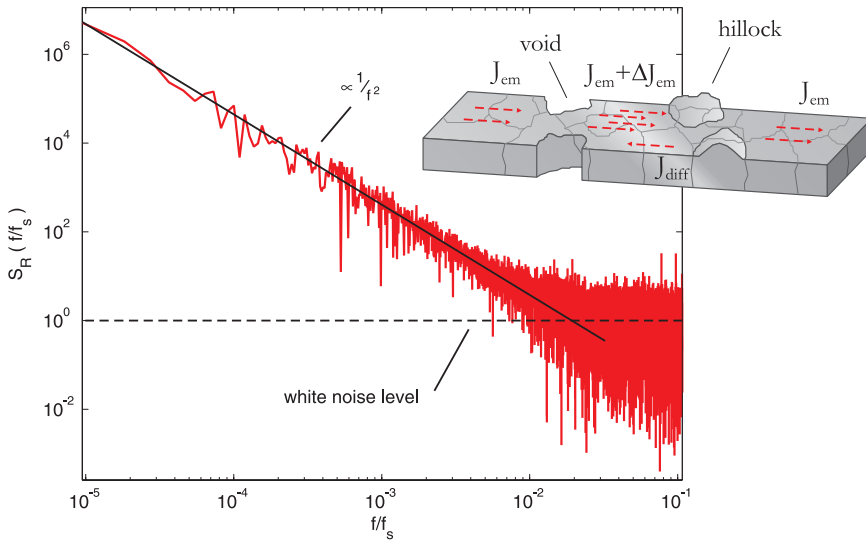


Fig. 2.1 Measured power spectral density of electromigration induced degradation (inset) in an aluminium resistor applying $6 \text{ mA} \cdot \mu\text{m}^{-2}$ current density for 58 days at 0.04 Hz sampling frequency f_s .

perature T , where ρ is the resistivity of the resistor and γ a constant depending on the quality of the film [2.9].

$$S_R = \frac{\gamma \rho j}{f^\alpha kT} e^{-E_a/kT} \quad (2.4)$$

Local differences in the electromigration flux can cause the creation of voids and hillocks at high current densities (Fig. 2.1), which can alter the thermal profile on the resistor when used as heater. The increase in current density near voids can eventually lead to failure of the thin-film resistor. Electromigration mainly occurs along grain boundaries having lowest activation energy E_a , which explains the direct correlation of the grain-size distribution with the electromigration noise and reliability performance of the thin-film [2.15, 2.16]. Annealing can be applied to increase grain size after recrystallisation, which can improve electromigration performance [2.17, 2.18].

Thin-film resistors used for temperature sensing are measured using small currents. The noise will therefore largely be consistent with flicker noise, in the absence of other diffusion mechanisms. The flicker noise affects the accuracy of flow sensors using thin-film resistors as temperature sensors. Thin-film resistors are noisier than bulk material, due to the thin-film microstructure and are therefore not accurate enough to be used in flow sensors in the $\text{nl} \cdot \text{min}^{-1}$ range.

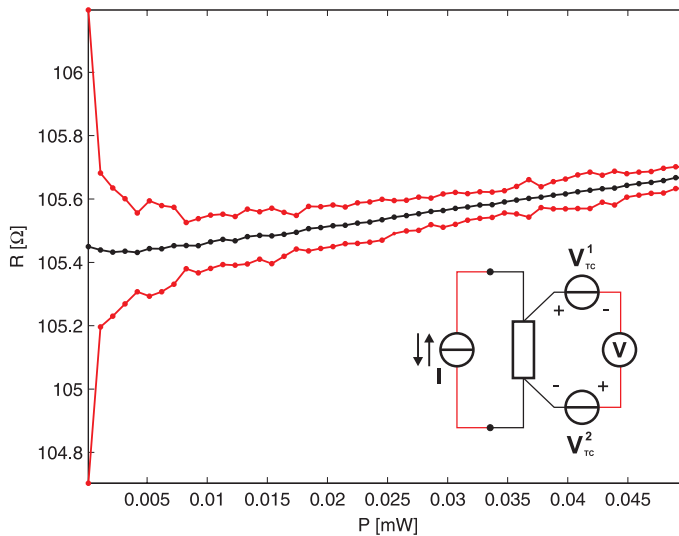


Fig. 2.2 Thermocouple offset compensation by alternating the direction of the current used during four-point measurement of the resistance value R at different dissipated powers P .

Thin-film resistors used for heating have high current densities at elevated temperatures. The noise will therefore depend on degradation by electromigration. Thin-film resistors can however still be used as heaters in thermal flow sensors for the $\text{nl}\cdot\text{min}^{-1}$ range. The dissipated heat can be controlled accurately by adjusting the current through the resistor according to $I = (P/R)^{1/2}$, where the resistance has to be determined by four-point contact.

Controlling a small power in a heater resistor leads to a small voltage drop across the resistor. Thermocouple voltages V_{TC} , generated in contact leads (Fig. 2.2) between the voltmeter and the resistor, have therefore influence on the measured resistance value R . The effect of the thermocouple offset voltages can be eliminated by alternating the current I through the resistor, where the resistance value R is determined by averaging over the resistance values obtained with alternating directions of the heating current (Fig. 2.2).

2.2.2 Thin-Film Thermopiles

In leads of electrically conductive material a temperature gradient along the conducting lead gives rise to a thermo-diffusion current, which is additional to the current given by Ohm's law. Both currents must balance if the lead has open ends, meaning that no net current can flow through the lead. The thermo-diffusion current is proportional to the temperature gradient, which through the drift-current by Ohm's law results in an electrical potential being generated (Seebeck effect). The generated

electrical potential can actually be measured with a thermopile, where leads of different material are connected in series, parallel to the direction of the temperature gradient ΔT_{TC} to be measured (Fig. 2.3).

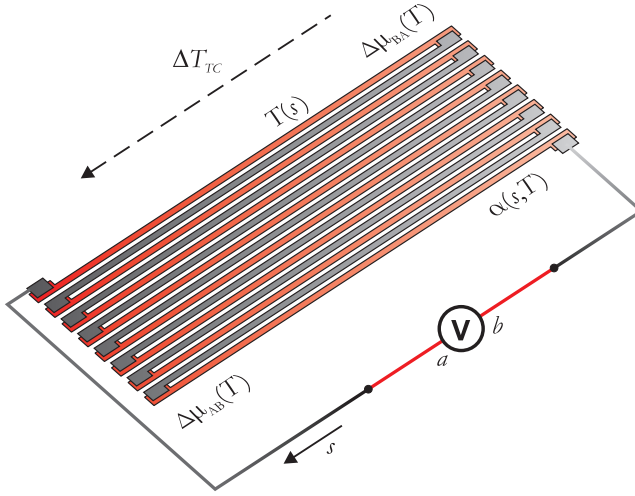


Fig. 2.3 Measurement of thermopile voltage V_{ab} generated by a temperature gradient ΔT_{TC} along thermopile leads of different conductive materials.

A complete description of thermal-electrical conduction in conductive materials is gained considering the interrelation of the heat flux \vec{h} with the electrical current density \vec{j} . Equations (2.5) describe the transport of charge carriers, which depends on the temperature gradient ∇T in the material and the electrochemical field $\vec{E}_{e-ch} = \vec{E} - \nabla\mu/q$, which is the combination of the electrostatic field \vec{E} and the gradient in the chemical potential μ [2.19]. Equations (2.5) describe the Seebeck, Peltier and Thomson effects by a matrix L , where L^{11} is the electrical conductivity σ of the material.

$$\begin{aligned}\vec{j} &= L^{11}\vec{E}_{e-ch} - L^{12}\nabla T \\ \vec{h} &= L^{21}\vec{E}_{e-ch} - L^{22}\nabla T\end{aligned}\quad (2.5)$$

A voltage V_{ab} can be measured due to the aforementioned Seebeck effect, which is generated by the temperature gradient $\nabla T(s)$ along leads of different material. The voltage is generated without a current density \vec{j} flowing through the material. This means that the electrochemical field \vec{E}_{e-ch} can be derived from the equation for \vec{j} in (2.5), which can then be substituted in the equation for \vec{h} in (2.5). This results in the equation for the electrochemical field \vec{E}_{e-ch} in (2.6), generated by a temperature

gradient, where α is the Seebeck coefficient of the material. The other equation in (2.6) gives the heat flux \vec{b} through the material due to the temperature gradient, with κ the thermal conductivity, which for metals is approximately equal to L^2 , defined as $L^2 = LT\sigma$ according to the Wiedemann-Franz law, where L is the Lorentz number and T the absolute temperature [2.19].

$$\begin{aligned} \vec{E}_{e-ch} &= \frac{L^{12}}{L^{11}} \nabla T = \alpha \nabla T \\ \vec{b} &= -\left(L^{22} - L^{21} (L^{11})^{-1} L^{12}\right) \nabla T = -\kappa \nabla T \end{aligned} \tag{2.6}$$

The voltage measured V_{ab} on a thermopile equals the line integral over the equation for the electrochemical field \vec{E}_{e-ch} in (2.6) taken along the electrical path connecting a voltmeter end terminal at a to an end terminal at b (Fig. 2.3). The line integral (2.7) is written in terms of the electrostatic field \vec{E} , the chemical potential μ , the effective Seebeck coefficient α_{eff} evaluated over the cross-section of the lead [2.20] and the temperature T along the lead.

$$\begin{aligned} \int_a^b \vec{E} \cdot d\vec{s} - \frac{1}{q} \int_a^b \partial_s \mu(s, T) ds &= \int_a^b \alpha_{eff}(s, T) \partial_s T(s) ds \Rightarrow \\ V_{ab} &= - \int_a^b \alpha_{eff}(s, T) \partial_s T(s) ds \end{aligned} \tag{2.7}$$

The term for the chemical potential μ in (2.7) does not contribute to the measured voltage, assuming that both end terminals are of the same material at the same temperature (the same μ), and that the voltmeter has high impedance [2.19]. It also means that the measured voltage V is independent of contact potentials between materials with different chemical potential, e.g. the thermocouple junctions $\Delta\mu_{AB}$ in Fig. 2.3. The term with the temperature gradient in (2.7) gives a measurable voltage only when the Seebeck coefficient varies along the line integral, which is the case for a thermopile made of leads of different material. A thermopile has a drift-free zero offset, in that no voltage is generated if the thermopile is at a uniform temperature.

A figure of merit (2.8) for thermocouple materials determines the signal-to-noise ratio [2.21, 2.22], where the thermal conductivity κ should be low allowing for the largest possible temperature difference between the thermopile junctions. The Seebeck coefficient α should be high to have high output voltage and the electrical resistivity ρ should be low to have low Johnson noise.

$$\xi = \frac{\alpha^2}{\rho\kappa} \tag{2.8}$$

The figure of merit is high for materials with large Seebeck coefficient, which for semiconductors is larger than for metals. High output voltage is obtained by combining materials with a large difference in Seebeck coefficient, which can be obtained with metal-semiconductor thermocouples. For instance, in this thesis aluminium with a Seebeck coefficient of $\alpha_{Al} = -1.66 \mu\text{V}\cdot\text{K}^{-1}$ [2.23] has been used in combination with boron doped poly-Si with approximately a Seebeck coefficient of $\alpha_{poly-Si} = 0.6 \text{mV}\cdot\text{K}^{-1}$ [2.22] at a measured resistivity of $\rho_{poly-Si} = 0.7 \text{m}\Omega\cdot\text{m}^{-1}$. Many different semiconductor materials, with high Seebeck coefficient are commonly used, like bismuth-telluride and indium-arsenide [2.23], while Al/poly-Si⁺⁺ thermopiles can easily be fabricated with standard CMOS technology.

2.3 Flow Sensing Concepts

2.3.1 Constant-Power Calorimetric Sensing with Resistors

A thermally fully heated calorimetric flow sensor requires the integration of resistors for heating and temperature sensing on a freely suspended flow channel (Fig. 2.4). Most calorimetric flow sensors use constant heating power on a heater resistor centred between two sensing resistors. Constant power usually means that a

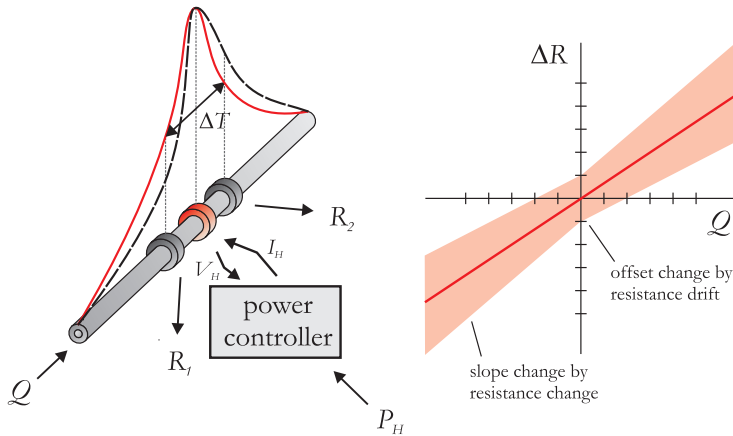


Fig. 2.4 Influence of material degradation on a calorimetric flow sensor using resistors for temperature sensing.

fixed current is applied. However, accurate micro thermal flow sensors require four-point measurement of the heater resistor and power control of the heating power P_H in order to at least eliminate the influence of resistance drift on the sensor output. Four-point resistance measurement can be applied, while alternating the measurement current (Fig. 2.2) in order to accurately obtain the resistance value of the heater. A

power controller uses the measured resistance value to control the current required to dissipate the exact heating power required. The temperature imbalance due to the flow rate Q is measured by two resistors R_1, R_2 , which can still cause long-term drift in the sensor output, where a change in offset in the sensor output is caused by a difference in resistance drift between both sensor resistors (Fig. 2.4). A change in slope and offset is caused by drift in the temperature coefficient of resistivity of the resistors.

Figure 2.5 shows thermal model results evaluating drift influences on a calorimetric flow sensor using Pt resistors (Chap. 4), where the temperature imbalance due to the flow rate Q is measured by a voltage difference ΔV output of a Wheatstrone bridge containing the sensing resistors R_1, R_2 . It is assumed that drift due to heater resistor degradation is fully eliminated. Already a 1% increase in the resistance value $R_{0,1}$ causes a significant offset drift compared to the fully symmetrical flow sensor without offset. The offset is relatively large compared to the small change in resistance due to the flow, because of a small temperature coefficient of resistivity. An exaggerated 20% increase in the temperature coefficient of resistivity also causes an offset, while simultaneously the sensitivity for the flow rate Q changes. The modelled drift in material properties indicate that, because of long-term drift in resistance values, thin-film resistors cannot be used as absolute temperature sensors for accurate micro thermal flow sensing.

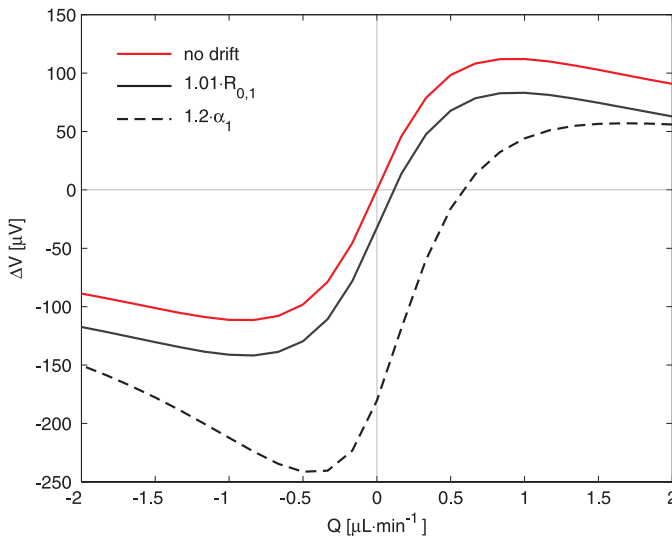


Fig. 2.5 Modelled influence of material degradation on a calorimetric flow sensor using resistors (Chap. 4), with 1.9 mW constant-power applied.

2.3.2 Constant-Power Calorimetric Sensing with Thermopiles

The use of resistors as absolute temperature sensors can be avoided by using a thermopile (Fig. 2.6), with the thermocouple leads positioned parallel to the flow channel. Four-point resistance measurement with power control can be applied on resistors used for heating, cancelling the influence of resistance drift on the sensor output signal. A measure of the flow rate is given by the thermopile output voltage V_{TC} , generated by a temperature difference ΔT_{TC} with the two heaters dissipating equal heating power P_H . The voltage output V_{TC} is obtained by integrating over the gradient in the thermal profile along the leads of the thermopile (2.7). Uniform drift in the Seebeck coefficient in one of the materials of the thermopile causes a change in the slope of the sensor output, because the thermopile changes its overall sensitivity. Non-uniform drift in the profile of the Seebeck coefficient along the thermopile leads [2.24, 2.25] can result in voltage offset drift even with a symmetrical temperature profile on the leads, with zero temperature difference ΔT_{TC} between the upstream and downstream thermopile junctions.

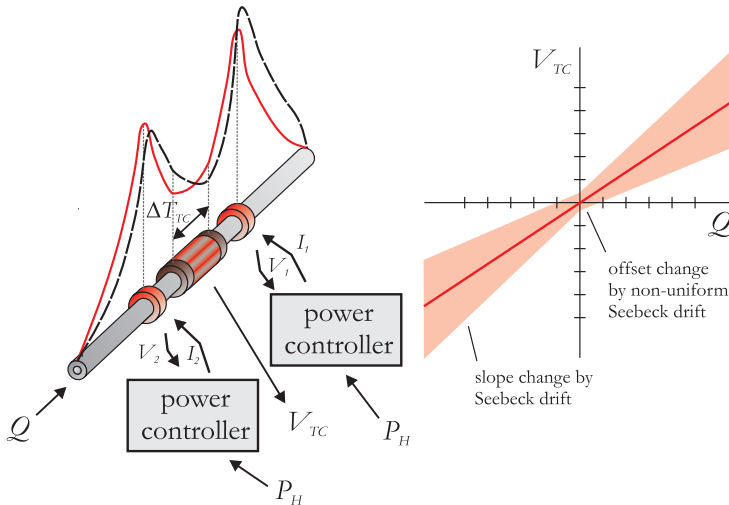


Fig. 2.6 Influence of material degradation on a thermopile calorimetric flow sensor.

Figure 2.7 shows thermal model results evaluating drift influences on a calorimetric flow sensor applying an Al/poly-Si⁺⁺ thermopile (Chap. 5). The thermopile output voltage ΔV_{TC} gives a measure of the flow rate Q , with two heaters dissipating an equal amount of power P_H . A 50% overall change in the Seebeck coefficient of the doped poly-Si leads in the thermopile causes an increase in the output sensitivity of the flow

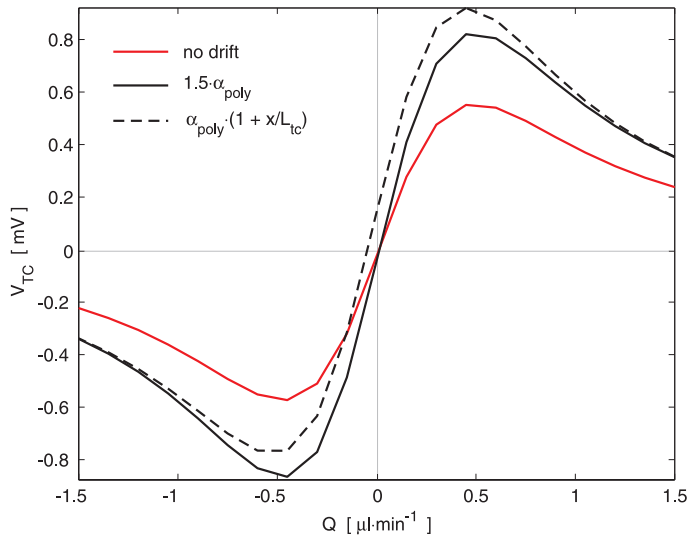


Fig. 2.7 Modelled influence of material degradation on a calorimetric flow sensor using a thermopile (Chap. 5), with $50 \mu\text{W}$ constant-power applied in each heater resistor.

sensor, without introducing offset drift. An exaggerated linear increasing Seebeck coefficient along the doped poly-Si leads of the thermopile results in an increase in sensor output sensitivity and the sensor output also obtains a small offset voltage. Long-term drift of the Seebeck coefficient of the thermocouple material, due to degradation, affects mostly the sensitivity of the calorimetric thermopile flow sensor. Non-uniform introduction of impurities and defects along the thermopile leads can result in a small drift in offset voltage.

2.3.3 Temperature-Balancing Calorimetric Sensing with Thermopiles

The calorimetric thermopile flow sensor can be made independent of thermopile sensitivity drift and thermopile non-linear characteristics applying a control-system feedback loop cancelling the generated output voltage V_{TC} . This provides for an almost drift-free error signal (Fig. 2.8). The output voltage V_{TC} and temperature difference across the thermopile is cancelled by controlling a power difference ΔP between the the heater resistors using a balancing controller, with a fixed amount of total power P_T being dissipated, according to the temperature-balancing anemometry principle [2.26, 2.27]. Additionally, power controllers are used to control the powers P_1, P_2 dissipated in the heater resistors up- and downstream from the thermopile, where the heater resistance values are determined by four-point resistance measurements, cancelling influence of heater resistance drift. Applying a fluid flow

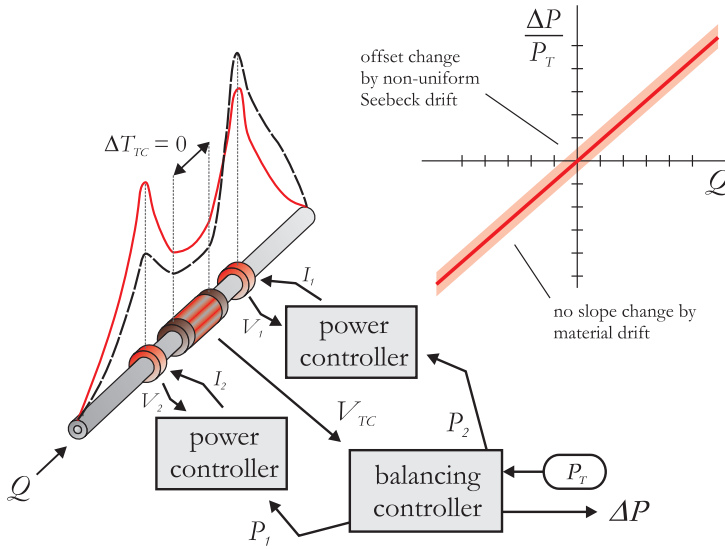


Fig. 2.8 Influence of material drift on thermopile-based temperature-balancing flow sensor.

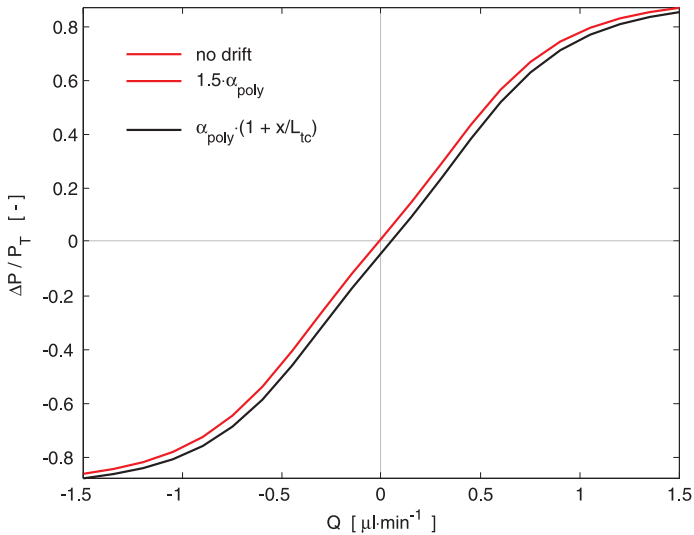


Fig. 2.9 Modelled influence of material drift on a temperature-balancing flow sensor with thermopile (Chap. 5), with 0.1 mW total power dissipated.

changes the temperature distribution around the heaters by advection of heat. This results in the relation $\Delta P/P_T$ being dependent on the flow rate Q , which is linear for small flow rates Q .

Figure 2.9 shows thermal model results evaluating drift influences on a calorimetric micro flow sensor using a thermopile (Chap. 5), while applying the temperature-balancing anemometry concept. The sensitivity drift observed in open-loop (Fig. 2.7) is fully compensated. The sensor output also increases monotonically over a larger flow range. A linear increasing Seebeck coefficient along the doped poly-Si leads of the thermopile can still result in a minor offset voltage. Recalibration of the flow sensor can readily be performed by measuring the offset voltage without flow.

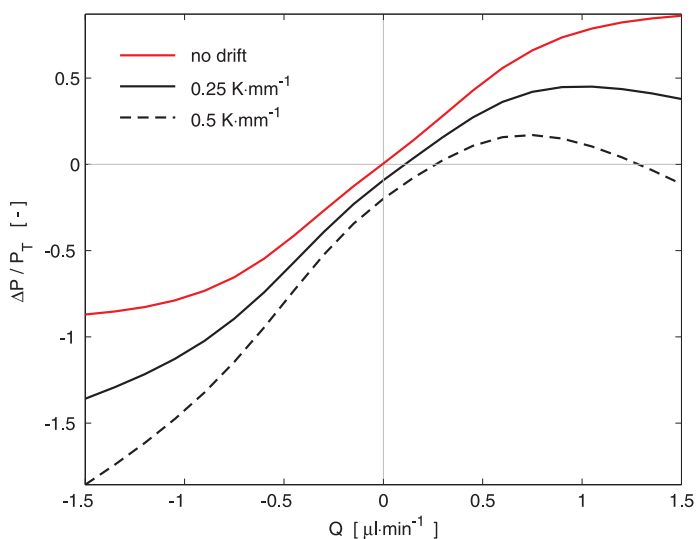


Fig. 2.10 Modelled influence of an external temperature gradient along the flow channel of a temperature-balancing flow sensor with thermopile (Chap. 5), with 0.1 mW total power dissipated.

Calorimetric flow sensors measure a flow induced temperature imbalance, which inadvertently means that the sensor also measures external temperature gradient in the direction of the flow. Figure 2.10 shows thermal model results, where the sensor output shows drift due to an external temperature gradient imposed on the freely-suspended flow channel. Multiple channels with flow in opposite direction can be used to compensate for these external temperature gradients (Chap. 8).

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3

Surface Channel Technology

MEMS fluidic devices often require the integration of transducers with freely suspended microchannels. This chapter presents a microchannel fabrication concept, allowing for easy fluidic interfacing and integration of thin-film transducer material in close proximity to the fluid. This is achieved by the reliable fabrication of completely sealed microchannels directly below the substrate surface. The resulting planar surface allows for the deposition of transducer material and pattern transfer by lithography. The microchannels are subsequently released and fluidic entrance holes are created, while transducer material can be protected by photoresist. The viability of the concept was demonstrated by fabrication of several monolithic micro-fluidic device templates. Fabricated surface microchannel devices can be packaged or optionally be vacuum sealed using a glass capping wafer, which is under investigation at the time of writing.^{3.1}

3.1 Introduction

The downscaling of microfluidic applications and devices requires the fabrication of small microchannels with the ability of integrating various types of transducers. Often the transducer needs to be integrated in close proximity to the fluid. For instance, in thermal applications the fluid should be heated or the fluid temperature has to be measured. Moreover, thermal isolation from the heat-conducting substrate requires the ability for the microchannels to be suspended freely [3.2-3.4].

Freely-suspended microchannels are also being used for fluid-mechanical resonance applications [3.5-3.13], requiring mechanically-strong microchannels, preventing buckling of the channels due to bending, but sufficiently compliant for actuation.

Microchannels have to be mechanically strong for high pressure applications, with proper fluidic interfacing to macro-fluidic connections, possibly with zero dead volume, while the interfacing to the microchannel and the microchannel itself should be chemically inert for various chemical applications [3.14-3.16].

Microchannels fabricated by bulk-bond micromachining and surface micromachining are limited in fulfilling mentioned requirements. For instance, integration of transducers in close proximity to the fluid is usually difficult for freely suspended microchannels fabricated using bulk micromachining and high-temperature bonding

^{3.1} This chapter is based on an article published in J. Micromech. Microeng. [3.1]. Contents has been added on experimental fluidic resonance measurements (Sec. 3.4.1) and packaging (Sec. 3.2.5, Sec. 3.4.3).

techniques. Silicon fusion bonding has successfully been used by Enoksson et al. [3.5, 3.6] in defining freely-suspended microchannels (Fig. 3.1a) for a Coriolis mass-flow sensor. However, integration of capacitive read-out electrodes and packaging requires bonding of additional process wafers, leading to large wafer-stack implementations [3.7-3.9]. Smaller cross-section freely-suspended microchannels were made by Sparks, Tadigadapa et al. [3.10, 3.11] for fluid density and Coriolis mass flow sensing. The devices are fabricated using doping as etch stop and lost wafer processing of two silicon wafers with a glass wafer as support (Fig. 3.1b). Similarly, doping and etching of a Pyrex wafer is used by Agah, Lu et al. [3.14, 3.15] for micro gas-chromatography column fabrication (Fig. 3.1c). The microchannels are thermally isolated from the silicon by backside etching of the support wafer.

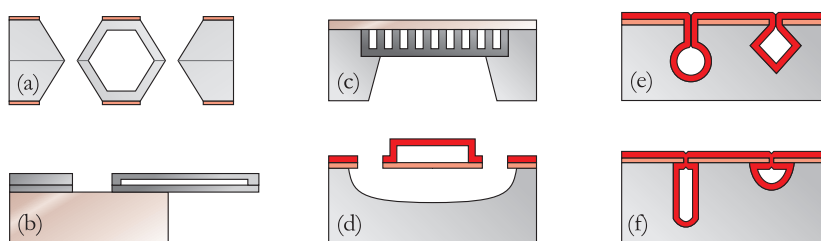


Fig. 3.1 Overview of possible freely suspended microchannel fabrication schemes, with a) wafer bonding by Enoksson et al. [3.5], b) lost-wafer process by Sparks et al. [3.10], c) doping etch stop and Pyrex wafer etching by Agah et al. [3.14], d) freely-suspended sacrificial microchannels, e) buried channel technology (BCT) by de Boer et al. [3.21] and f) embedded Parylene-microchannel technology by Chen et al. [3.24].

Surface micromachined channels (Fig. 3.1d) generally allow for easy integration of transducers on a single wafer [3.3, 3.4], however step-coverage problems can occur for thin-film transducer material deposited across the microchannel. Furthermore, release of the sacrificial layer is time consuming, while wet-etching requires inert transducer materials for the etchant used. A major disadvantage of surface micromachined channels is the large hydraulic resistance, due to the limitation in channel height, determined by the sacrificial layer.

A good alternative is to make microchannels directly below the substrate surface [3.16-3.27]. This chapter describes the development of a versatile technological concept entitled surface channel technology (SCT), which is based on buried channel technology (BCT) (Fig. 3.1e) developed by de Boer et al. [3.21]. The BCT process has been applied in the fabrication of chromatography columns [3.22] (Fig. 3.2a) and micropipettes (Fig. 3.2b) for studying the mechanical properties of single DNA molecules [3.23]. Surface channel technology (SCT) combines freely-suspended low-stress microchannels directly below the substrate surface, with excellent properties for chemical-inert fluidic interfacing and integration of transducers in close

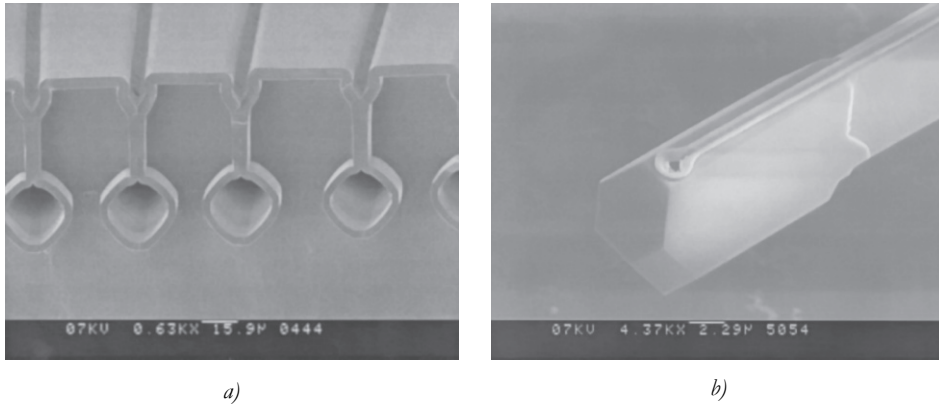


Fig. 3.2 Devices fabricated with the buried channel technology, with a) a chromatography column and b) a micro-pipette, by de Boer et al. [3.21].

proximity to the fluid. A similar embedded parylene-microchannel technology (Fig. 3.1f) has recently been developed by Chen et al. [3.24]. The technology is compatible with CMOS back-end and further low-temperature MEMS processes, which enables the fabrication of embedded structures integrated with on-chip transducers for miniaturised lab-on-a-chip systems. A number of microfluidic devices have been fabricated with the embedded parylene technology [3.25-3.27].

3.2 Surface Microchannel Concept

The principle behind surface channel technology (SCT) is to make low-stress chemical-inert microchannels directly below the substrate surface, which are completely sealed after processing, leaving a planar substrate with negligible surface topology. This allows for the deposition of transducer materials and the pattern transfer by lithography, without material being deposited inside the microchannels. Subsequently the microchannels are released and fluidic entrance holes are created. Additionally, the microfluidic devices can be packaged and possibly vacuum sealed using a glass capping wafer.

3.2.1 Surface Microchannel Fabrication

Surface microchannels are fabricated (App. A.1) using a one-mask process scheme (Fig. 3.3). The mask layout defines etch holes, 2 μm in width, combined into segmented lines, where each line defines a microchannel. The mask layout is transferred into a 500 nm tensile low-stress (<300 MPa) SiRN layer [3.28] deposited on a silicon wafer by LPCVD (Fig. 3.3a). The channels are isotropically etched

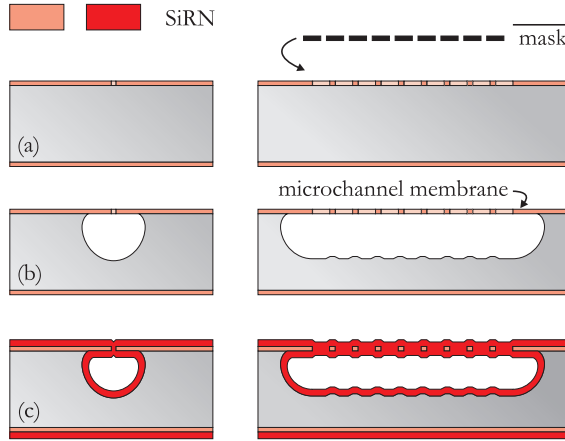


Fig. 3.3 Surface microchannel process scheme, applying a single mask defining etch holes in a 500 nm SiRN layer, showing perpendicular (left column) and longitudinal (right column) cross sections through the microchannel.

through the etch holes, leading to coalescence of neighbouring etch surfaces, resulting in long channels with semicircular profiles (Fig. 3.3b).

Applying etch holes in a slightly tensile SiRN layer ensures mechanical stability of the SiRN microchannel membranes formed during etching. This means that the etch holes can be properly sealed by a second LPCVD SiRN deposition (Fig. 3.3c). The inside surfaces of the microchannels are uniformly coated by SiRN during the deposition, where the thickness is limited by the sealing of the etch holes.

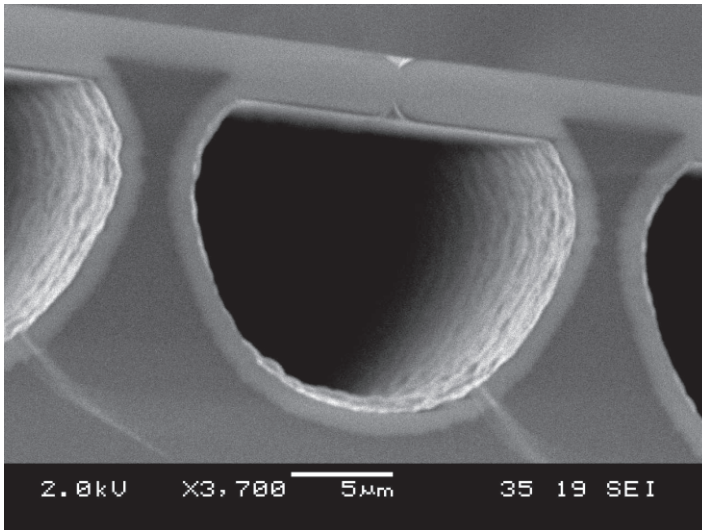


Fig. 3.4 Cross-section of sealed surface microchannels.

A nearly perfect transfer of the mask layout into the initial SiRN layer is necessary, and high etching selectivity of Si compared to SiRN is required during microchannel etching, in order to limit the SiRN layer thickness required to seal the etch holes to about $2\ \mu\text{m}$, limiting internal stress in the layer. This is achieved using a $50\ \text{nm}$ Cr mask during microchannel etching. UV-ozone treatment is required, to improve wetting of the photoresist during wet-etching of the Cr mask layer.

Figure 3.4 shows a cross-section of sealed microchannels. The etch holes in the $500\ \text{nm}$ SiRN layer were $2.3\ \mu\text{m}$ in width after SiRN etching and were perfectly sealed by the LPCVD deposition of a $1.3\ \mu\text{m}$ thick SiRN layer, leaving a planar wafer surface.

Figure 3.5 shows the formation of surface microchannels due to the coalescence of etch surfaces, leaving notches on the microchannel wall. Surface microchannels of varying diameter can be created, but the notch pattern and lithographic resolution puts a limit to the smallest channel diameter feasible.

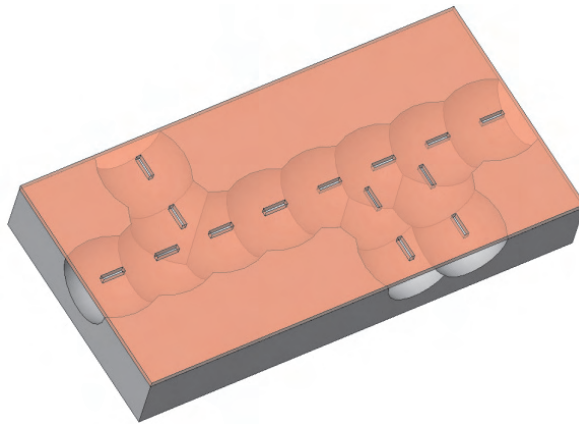


Fig. 3.5 Surface microchannels formed by the coalescence of etch surfaces.

The arrangement of etch holes determines the final size and shape of the microchannel network, independent of crystallographic orientation, giving large freedom in designing merged parallel channels, cavities and microchannel junctions. Etch-hole junctions and corners are to be avoided, because more SiRN would need to be deposited for proper sealing of the etch holes.

Etch holes were designed with a fixed width of $2\ \mu\text{m}$, which is still reproducible using contact lithography. The etch-hole length and separation length between etch holes were varied from $2\ \mu\text{m}$ up to $15\ \mu\text{m}$. The most reproducible microchannels with sufficient mechanical stability of the microchannel membrane were obtained at $5\ \mu\text{m}$ etch-hole length and $3\ \mu\text{m}$ separation between etch holes.

3.2.2 Transducer and Surface Micromachining Integration

The sealed SiRN microchannel membranes are mechanically strong allowing for further processing, whereby thin-film transducers and surface micromachined structures can be integrated on top of the microchannel membranes. Figure 3.6 shows integration of various structures on top of the sealed microchannels, implemented in microfluidic devices described in Chap. 4-7, 9. The SEM micrograph in Fig. 3.6 shows the integration of 200 nm Pt structures, without adhesion layer, on top of several merged surface microchannels. The Pt structures were fabricated by sputtering, with proper step-coverage across sealed etch holes (Chap. 4). Several electrical resistance measurements of 4 μm wires crossing etch holes 6 μm in width did not show any electrical failures, confirming that standard fabrication processes can readily be applied for the integration of thin-film layers. The microchannels have sufficient mechanical stability, such that Al/poly-Si⁺⁺ thermopiles (App A.2, Fig. 3.6), requiring solid-source diffusion (SSD) at 1050 °C (Chap. 5-7), and even poly-Si sacrificial nanochannels (Chap. 9) can be created on top of the microchannel membranes.

3.2.3 Surface Microchannel Release

Surface microchannels are ready to be released after transducers have been fabri-

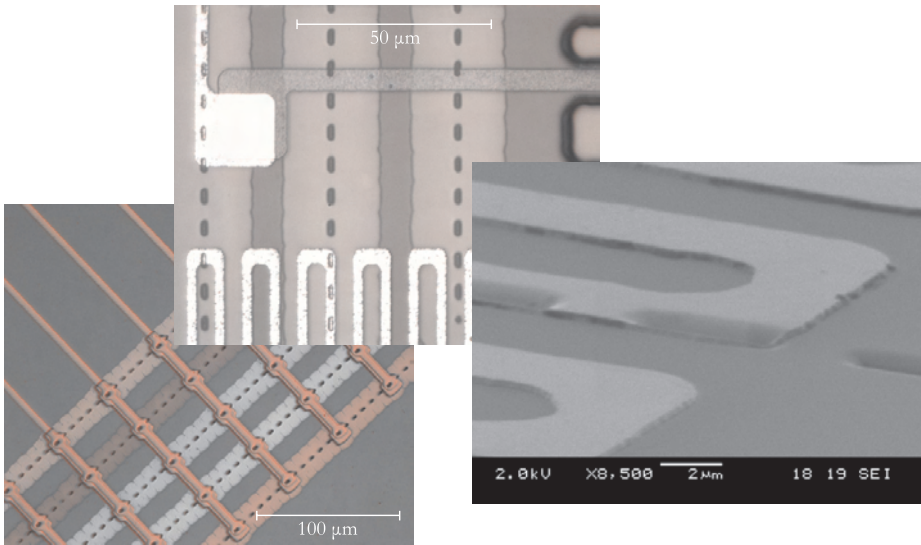


Fig. 3.6 Integration of various thin-film transducers and surface micromachined nanochannels on top of surface microchannels.

cated Figure 3.7 depicts two possible process schemes. KOH or TMAH wet-etching or non-directional plasma etching can be used for the release of the surface microchannels. Both release methods require the definition of access windows to the bulk silicon (Fig. 3.7a), while fluidic entrance holes are defined in the same mask as the access windows to the silicon substrate and are etched through the SiRN layer simultaneously. The diameter of the etched microchannel should not exceed the edge of the access window or otherwise the microchannel is opened on the side during SiRN etching of the access windows. The spacing between access window and microchannel can be minimised by accurate determination of the microchannel diameter during etching.

Isolation cavities are etched, with KOH or TMAH, underneath the microchannels, provided that the microchannels are designed not to be aligned to the $\langle 110 \rangle$ -direction (Fig. 3.7b). Release by wet etching is straightforward, however the etch solution can be aggressive against desired transducer materials and the microchannels need to be cleaned, in order to remove remaining etch solution inside the microchannels after processing.

An alternative is the release of surface microchannels by non-directional plasma etching (Fig. 3.7c), which allows for the protection of transducer material by a thick photoresist layer, where the photoresist layer is removed after etching (Fig. 3.7d). This makes it possible to integrate any required type of transducer material.

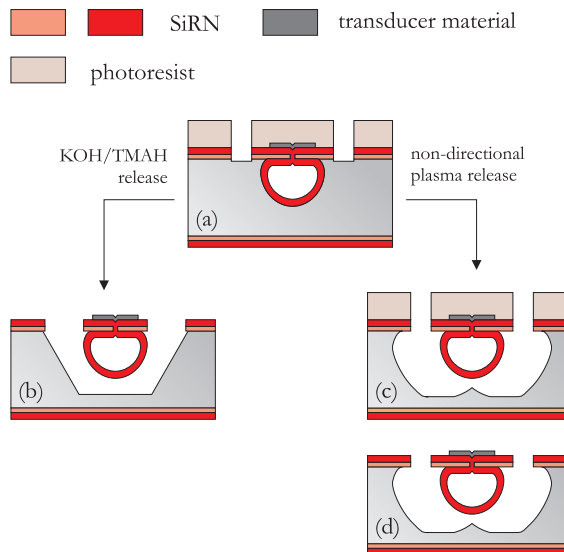


Fig. 3.7 Process schemes for the release of surface microchannels.

3.2.4 Fluidic Entrance Holes to the Surface Microchannels

Fluidic entrance holes to the microchannel can be made on the frontside of the wafer, with minimum processing effort required (Fig. 3.8). The fluidic entrance holes have to be defined in the same photoresist mask containing the access windows, because it is the last lithography step possible on a planar substrate and because etch-

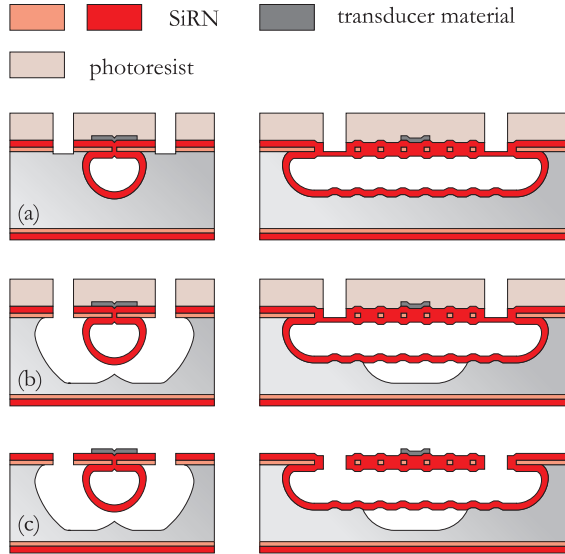


Fig. 3.8 Process scheme for the fabrication of fluidic entrance holes to the surface microchannels, showing perpendicular (left column) and longitudinal (right column) cross sections through the microchannel.

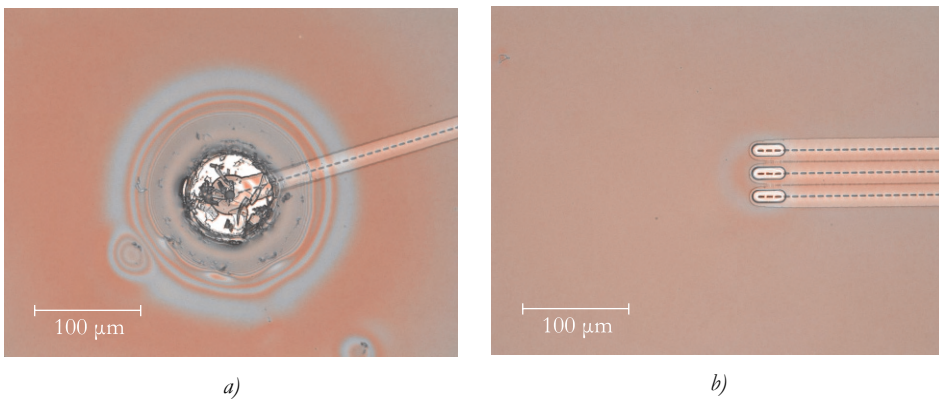


Fig. 3.9 Mask transfer in photoresist with a) fluidic entrance holes defined in microchannel mask and b) fluidic entrance holes defined in access window mask.

ing of fluidic entrance holes prior to the release of the microchannels causes the problem of improper mask transfer due to photoresist flowing into the microchannels (Fig. 3.9a).

However, the access windows to the silicon can be opened, while keeping the microchannels closed. This is possible, because the microchannel membranes are thicker than the SiRN layer on the substrate (Figs. 3.8a, 3.9b). This can be used for the protection of the inside surfaces of the microchannels during plasma release of the channels (Fig. 3.8b). The method can only be applied when the photoresist mask is not being removed, so that the fluidic entrance holes can be opened completely after the release of the microchannels (Fig. 3.8b), after which the photoresist can be completely removed (Fig. 3.8c).

Figure 3.10 shows separate fluidic entrance holes to several merged microchannels. The entrance holes are separated to maintain mechanical stability of the microchannel membrane. Chemically inert interfacing to the microchannel is possible, because fluid passing through the surface microchannels remains in contact with SiRN. Direct connection to the fluidic entrance holes can be made, applying fluidic connectors (e.g. Upchurch Scientific®-NanoPort™ assemblies) or O-rings on top of the substrate. Haneveld et al. [3.29] showed that with additional backside processing chemical inert fluidic connections can be made from the backside of the wafer.

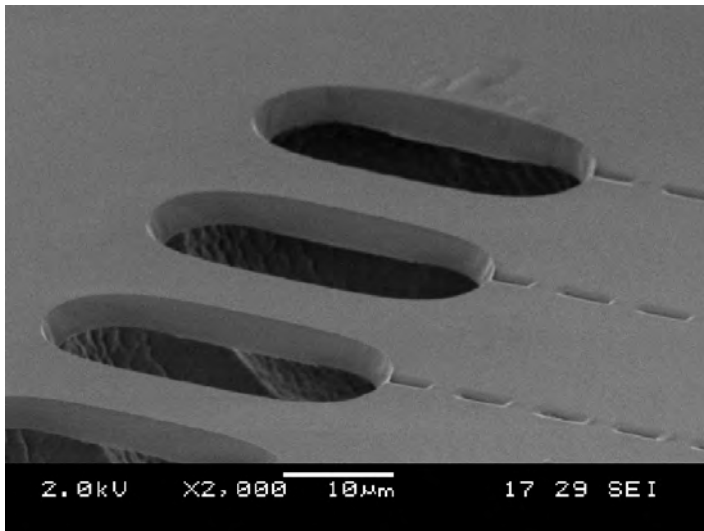


Fig. 3.10 Surface microchannel fluidic entrance holes, etched through the microchannel membrane.

3.2.5 Packaging of Surface Microchannel Devices

Fabricated surface microchannel devices are susceptible to dust contamination after devices leave the cleanroom. Component-level packaging [3.30] can be applied to keep devices dust free. However, a wafer-level vacuum packaging solution can additionally reduce thermal loss from the microchannel to the surrounding silicon, increasing thermal flow sensor sensitivity and can increase Q-factors for fluidic resonance devices [3.31]. Wafer-level vacuum packaging solutions include the sealing of thin-shell vacuum capsules sealed by LPCVD deposition (e.g. [3.32]). Bartek et al. [3.33] showed that high vacuum, with pressures down to 10^{-3} Pa, can be obtained by sealing capsules with an evaporated metal seal, where the sealing material is not being deposited inside the capsule. Burg et al. [3.31] achieved wafer-level vacuum packaging of microchannels fabricated below the wafer surface using glass-frit bonding of a Pyrex capping wafer, where electrical contact leads were fed through the glass-frit layer.

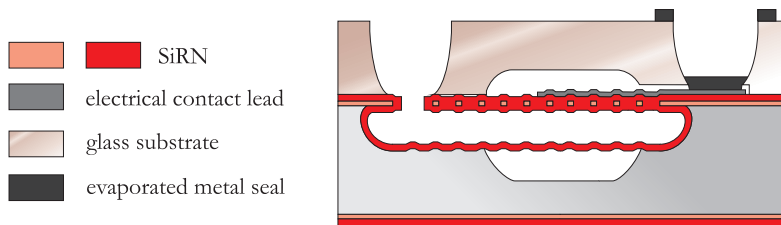


Fig. 3.11 Process scheme for wafer-level packaging of surface microchannel devices.

Microfluidic devices fabricated with surface channel technology are created using a single wafer. However, the surface of the wafer containing the microchannels is flat, also between etch holes, which means that the microchannel membranes can be properly bonded to a glass wafer, without having leakage through micro-cavities along the microchannels. This allows for the wafer-level packaging and possible vacuum sealing of surface channel devices by anodic bonding of a glass capping wafer containing vacuum cavities (Fig. 3.11). Electrical contact leads are contained in nanocavities etched in the capping wafer and are contacted by wafer-through contact holes etched through the wafer [3.34]. This allows for the anodic bonding of the glass-wafer before vacuum sealing, such that oxygen formed during bonding [3.35] can be pumped out of the vacuum cavity, before sealing the cavity with an evaporated metal seal. In this way high-vacuum can be obtained [3.33]. The evaporated metal seal also functions as an electrical feed-through.

3.3 Channel Profile Etching

An isotropic etch process is required for the fabrication of the microchannels, aiming for good channel uniformity across the wafer and good selectivity against SiRN. Either HF/HNO₃ wet-chemical etching or non-directional plasma etching can be used.

Wet-chemical etching using HF/HNO₃ has a significant loading effect on microchannel uniformity, requiring compensation structures [3.22]. Selectivity against

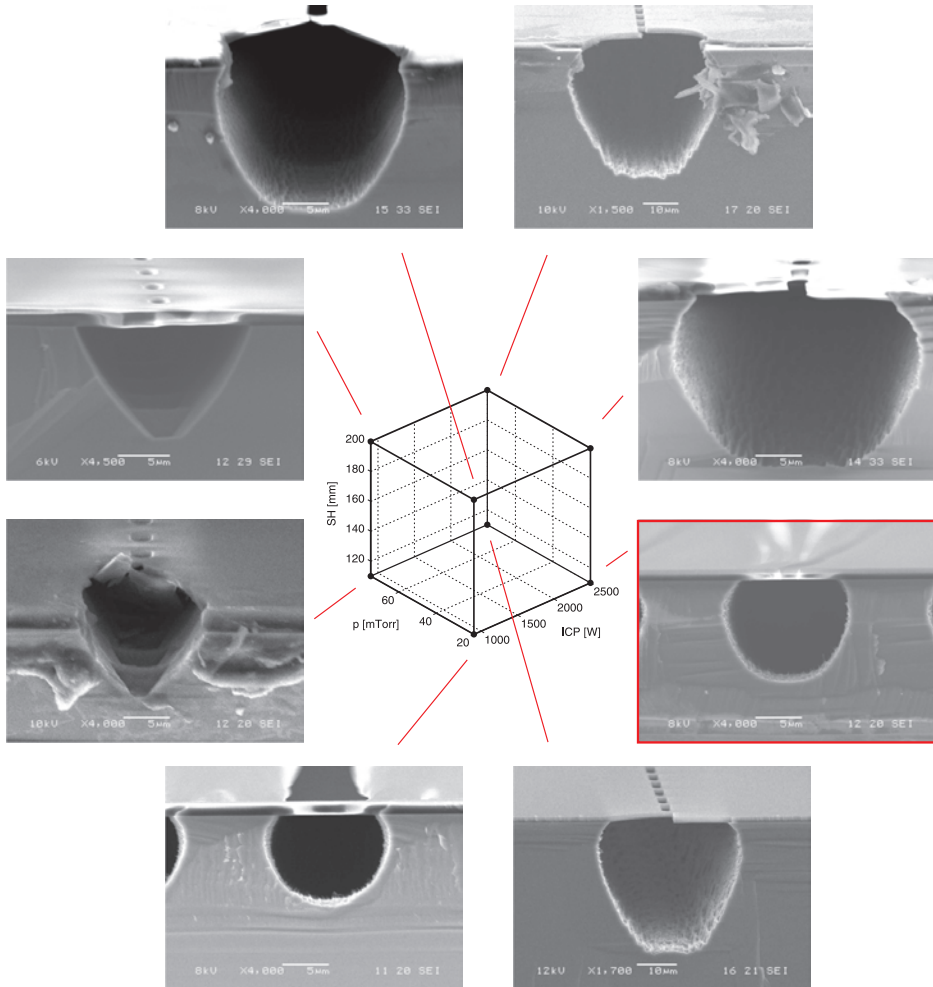


Fig. 3.12 Channel-profile parameter study using Adixen AMS 100 SE, applying fixed plate power, CCP = 0 W, SF₆ = 400 sccm and T_{electrode} = 20 °C for 10 min etching at about 10% loading. The substrate height (SH), chamber pressure (p) and ICP power were varied.

SiRN is low and pressure build-up of reaction gas inside the channels might lead to cracking of the SiRN membranes during etching.

Non-directional SF₆ plasma etching or highly selective XeF₂ vapour etching [3.36] offers good alternatives to wet-chemical etching. A parameter study on high-density SF₆ plasma etching with zero self-bias was conducted using the Adixen AMS 100 SE with photoresist as mask (Fig. 3.12). An isotropic profile was expected due to the small 2 μm etch holes, however a significant anisotropic etch effect was observed.

The etch profiles and surface roughness were observed to be in agreement with results obtained using similar etch parameters on larger mask openings [3.37]. In general, it was observed that etching at a lower pressure results in a more isotropic channel profile.

The best channel profile, with good uniformity across the wafer at high etch speed, was obtained applying an ICP of 2500 W at 20 mTorr with a substrate height of 110 mm. A vertical etch speed of 1.5 μm·min⁻¹, with a relative lateral etch speed of 0.75, was measured for microchannels etched with these settings for 10 minutes, with an etch rate of 25 nm·min⁻¹ measured for SiRN.

3.4 Surface Microchannel Process Results

Several wafers were processed with surface microchannels etched using the Adixen AMS 100 SE with the described process for 10 minutes, yielding microchannels with approximately 15 μm diameter. Several aspects of the surface channel technology were thereafter investigated and a functional thermal flow sensor was fabricated.

3.4.1 Release of Device Templates by KOH Etching

Various device templates were designed, exploring the use of surface microchannels for a range of microfluidic applications. The device templates were released by KOH etching and fluidic entrance holes were created on the frontside of the wafer. An additional mask was used for the release of the surface microchannels from the backside of the wafer, allowing for a closer inspection of the surface microchannels.

Figure 3.13 shows crossing microchannels observed from the backside of the wafer. The microchannels are emerging from {111}-planes created by KOH etching and are merged with the SiRN membrane on the frontside of the wafer. The template demonstrates the possibility in creating microfluidic networks for lab-on-chip applications. A range of designs can be realised, including e.g. reactor chambers, static micromixers and micro heat exchangers. Integration of heaters on top of freely-suspended microchannels allow for the efficient heating of the fluid, while rapid cool-

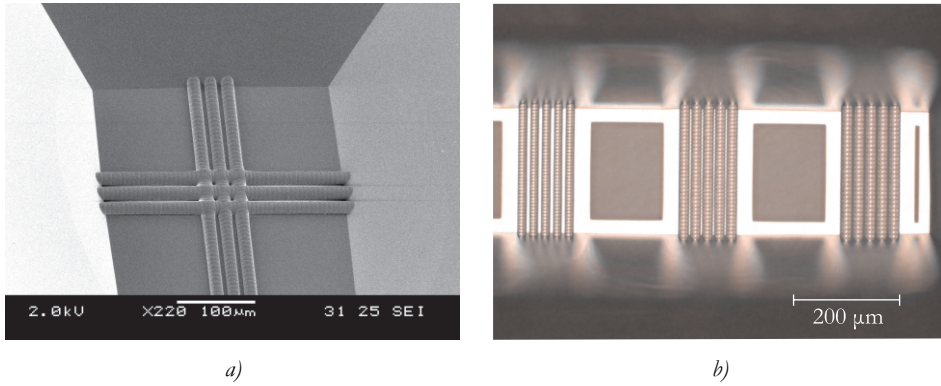


Fig. 3.13 Device template for lab-on-chip applications, with a) crossing microchannels on a Si₃N₄ membrane and b) suspended microchannels seen from the backside.

ing to ambient temperature can be achieved by the embedding of microchannels in the bulk silicon of the substrate.

Figure 3.14 shows a freely suspended surface microchannel on the frontside of the wafer. The channel is thermally isolated from the substrate by a cavity created by KOH etching. The device template has been used in the implementation of a calorimetric thermal flow sensor (Chap. 4).

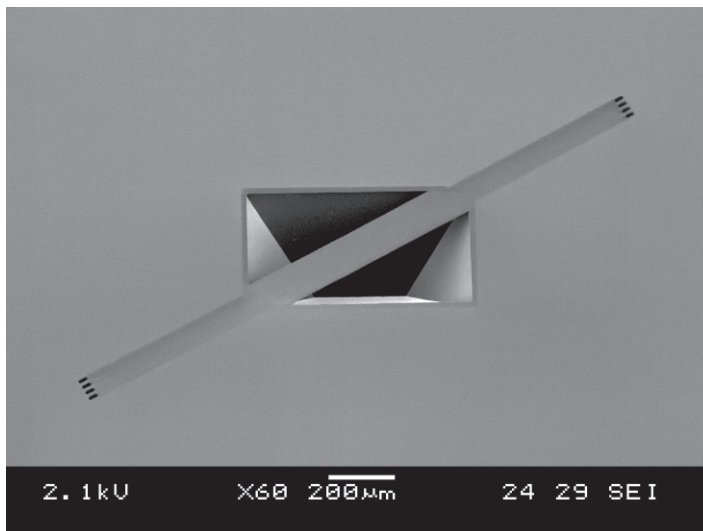


Fig. 3.14 Device template for surface microchannel thermal flow sensor.

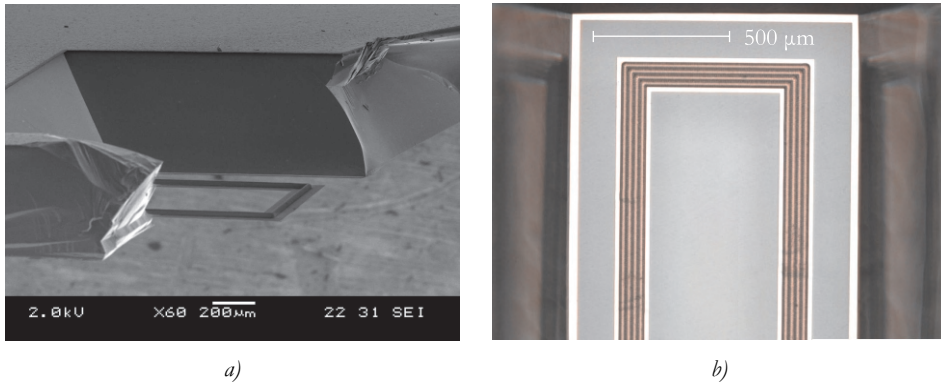


Fig. 3.15 Device template for fluid-mechanical resonance sensor applications, with a) wafer partially broken away and b) U-shaped resonance structure containing several merged microchannels seen from the backside.

Figure 3.15 shows 1 mm long surface microchannel resonance structures broken out of the wafer. The freely-suspended surface microchannel is slightly deflected out-of-plane, due to the inherent structural strength of the surface microchannel.

The device template can be used in fluid-mechanical resonance applications. Figure 3.16 shows the first resonance mode on a freely suspended microchannel resonance structure measured with a laser-vibrometer using a piezo stack for actuation.

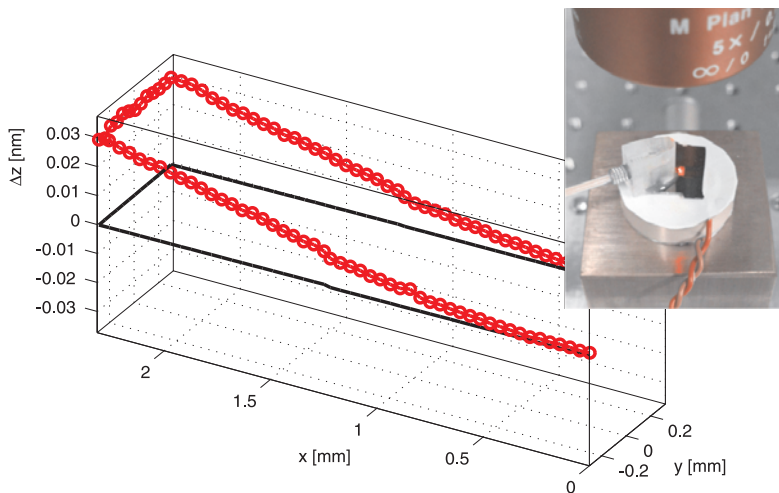


Fig. 3.16 First resonance mode at 6.4 kHz of a fluid-resonance device template filled with air. The inset shows the fluidic connection to the microchannels, the laser-vibrometer objective and a piezo stack for actuation.

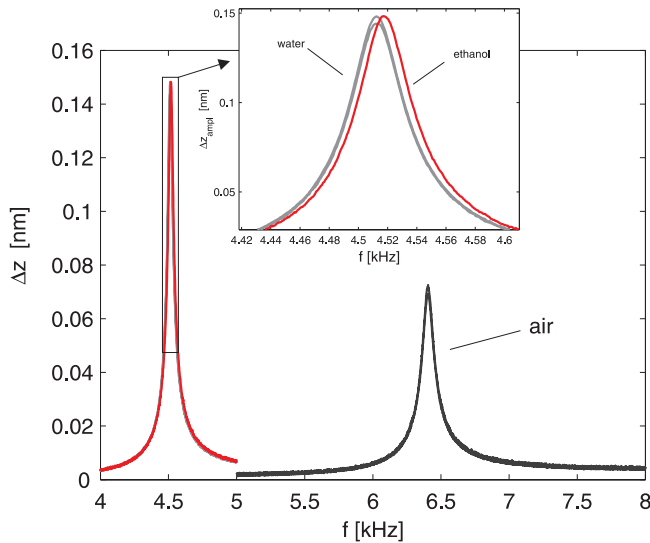


Fig. 3.17 Resonant shift of first resonance mode, due to filling of the microchannels, with fluids having different densities.

Figure 3.17 shows the possibility to measure fluid density by measuring the frequency shift of the first resonance mode. Haneveld et al. [3.29] demonstrated that a highly-sensitive Coriolis flow sensor can be obtained by adding comb-like electrodes and applying Lorentz-force actuation.

3.4.2 Release of Device Templates by SF₆ Plasma

Release of surface microchannels by non-directional SF₆ plasma release (App. A.3) was investigated on device templates intended for use in thermal flow-sensor applications, while photoresist should remain on the microchannels for the protection of transducer material. Surface microchannel structures were released by SF₆ plasma, applying the same recipe used for the etching of the microchannels and applying a 4 μm thick photoresist layer. Figure 3.18a shows a device templates released after 15 min etching, with photoresist still remaining on the freely-suspended microchannel. Large freely-suspended microchannels require SF₆ plasma etching at 1.8 kW ICP power, 220 mm SH, 350 sccm SF₆ flow and 15% valve at 0 °C preventing the heating of the freely-suspended microchannels and burning of the photoresist.

Figure 3.18a also shows that ridges remain below the released microchannels and electrical contact-lead support structures. The height of these ridges depends on the size of the access windows and width of the freely-suspended structures.

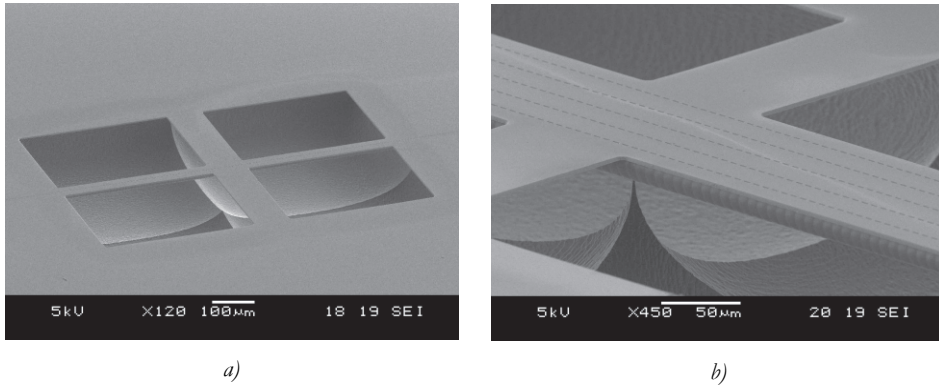


Fig. 3.18 a) Device template released by SF_6 plasma, with the possibility for the protection of transducer material with photoresist. b) Nearly released device, a small spike of Si makes thermal contact with the channels.

Figure 3.18b shows a device template also etched for 15 min, which is nearly released. The structure has several merged microchannels, requiring further etching to be released fully. To successfully release surface microchannel devices with SF_6 plasma requires sufficiently large access windows to the silicon and sufficiently thick photoresist for the protection of transducer material.

3.4.3 Packaging of Surface Microchannels Devices

The possibility for the wafer-level packaging of surface microchannel devices by anodic bonding was investigated. A wafer containing surface microchannels was anodically bonded to a Pyrex wafer, maintaining 450 °C for 1.5 hrs applying 800 V. Anodic bonding to the SiRN layer is made possible by wet-oxidation of the SiRN layer at 1050 °C for 1.5 hrs [3.38]. The oxidation has to be done before the integration of thin-film transducers. Wet-oxidation resulted in large stress in the SiRN layer, which can lead to breaking of the microchannel membranes. This was later solved by 1 min deposition of TEOS, which resulted in low stress in the SiRN layer, while anodic bonding to a Pyrex wafer was successful.

The Si wafer was removed by KOH etching after wet-oxidation and anodic bonding, resulted in surface microchannels on a Pyrex wafer, allowing for the inspection of the bond interface (Fig. 3.19a). Figure 3.19b shows that the SiRN layer surrounding the microchannels is everywhere perfectly bonded to the Pyrex wafer. Inspection from the opposite side revealed that the microchannel membranes were also properly bonded to the Pyrex wafer, confirming that wafer-level packaging and possibly vacuum sealing of surface microchannel devices can be achieved.

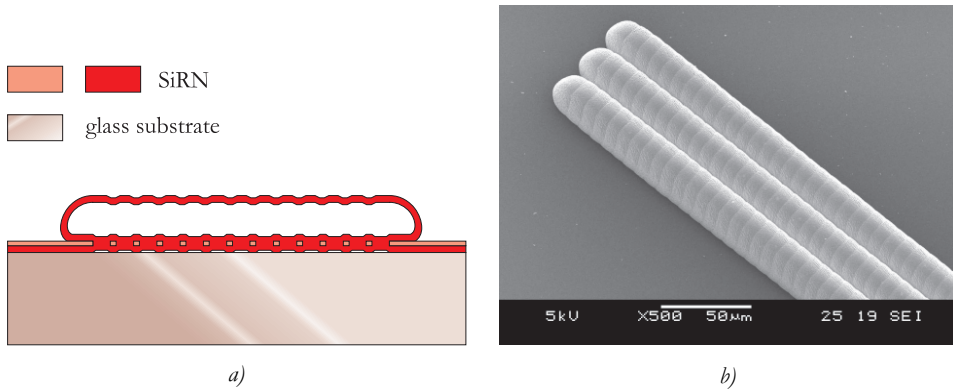


Fig. 3.19 Surface microchannels on a glass substrate, after anodic bonding and removal of the Si wafer by KOH wet-etching.

A capping wafer using a 200 μm thin Borofloat substrate has been designed, together with Micronit Microfluidics B.V., for the wafer-level packaging of surface microchannel devices. The capping wafer contains 50 μm deep wet-etched vacuum chambers (Fig. 3.20a), which continue as far as possible towards the wafer through contact holes (Fig. 3.20b). This is required for fast pump-down time, reaching high-vacuum in the Knudsen flow regime. Electrical contact leads on the process are fed through 300 nm deep nanocavities wet-etched in the capping wafer. Wafer through contact holes are wet-etched from the opposite side of the capping wafer, intentionally giving a sharp edge near the electrical contacts on the device wafer. This is to reduce shadowing effect during evaporation of the metal seal (Fig. 3.11). It should allow for proper sealing of the vacuum chamber under high-vacuum.

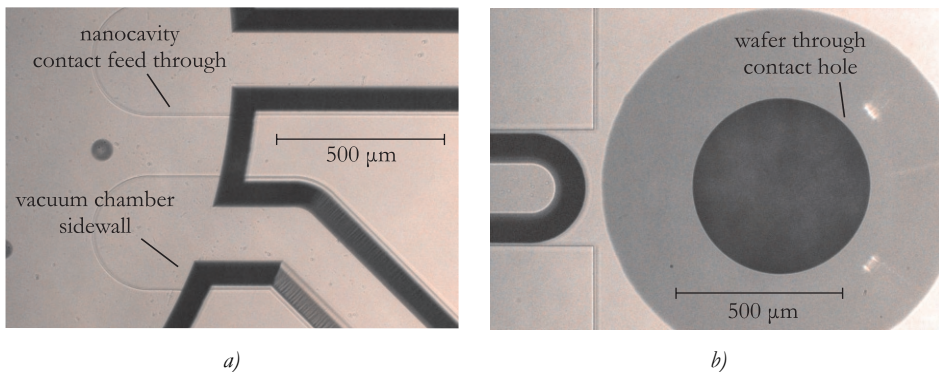


Fig. 3.20 Glass capping wafer for the wafer-level packaging of surface microchannel devices.

3.5 Conclusions

A versatile concept for the creation of freely suspended surface microchannels using a straightforward and reliable process was demonstrated. Surface channel technology (SCT) allows for easy fluidic interconnection and integration of transducer material in close proximity to the fluid. Complex microfluidic networks can be created, using a single mask defining etch holes. The microchannels were fabricated directly below the substrate surface, leaving a planar substrate, allowing for standard fabrication processes to be readily applied for the integration of transducers. The microchannels were etched using high-density SF₆ plasma with zero self-bias, resulting in good uniformity of the microchannels across the wafer. The microchannels were released and fluidic entrance holes were subsequently created. Several device templates have demonstrated the versatility of the concept. Release by non-directional SF₆ plasma allows for the protection of transducer structures by photoresist, required for the integration of any required type of transducer material. Additionally, surface microchannel devices can be packaged by anodic bonding using a glass capping wafer.

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4

Miniaturised Calorimetric Flow Sensor

A calorimetric miniaturised flow sensor was realised with a linear sensor response measured for water flow up to flow rates in the order of 300 nl·min⁻¹. The surface channel technology has been applied in the realisation of a sensor with a thermally isolated freely-suspended silicon-rich silicon-nitride microchannel directly below the substrate surface. The microchannel fabrication concept allowed for the planar integration of sensor structures in close proximity to the fluid, while chemical-resistant fluidic connections were made directly on top of the microchannel, without introducing large dead volumes. The realised flow sensor consists of a microchannel with low hydraulic resistance and 4.5 nl total fluid volume. A pressure driven flow setup was used to force water through the microchannel, measuring output sensitivity in the order of 0.2 μV/nl·min⁻¹ for flows up to about 300 nl·min⁻¹. The measured sensor output is in close agreement with results obtained from both a detailed and an approximate numerical model of the sensor.^{4.1}

4.1 Introduction

Fluidic applications within the field of micro- and nanofluidics often require the accurate measurement of small fluid flows, in the order of nl·min⁻¹. For such small flow rates, standard bulk micromachined flow channels are not suitable, because a large part of the flow is in thermal contact with the substrate and does not contribute to the sensor signal. Instead, thermally isolated channels are required. This chapter describes a miniaturised flow sensor based on freely suspended silicon-rich silicon-nitride semicircular microchannels, which are thermally isolated from the substrate. The hydraulic resistance of the microchannels can be freely chosen by adjusting the microchannel diameter. This gives advantage over freely-suspended surface micromachined channels [4.2-4.4], which have large hydraulic resistance of the microchannel, mostly determined by the channel height being limited to only a few μm. Additionally, surface micromachined channel can give step-coverage problems for sensor structures required to be deposited across the microchannel, whereas the flow sensor described in this chapter is readily made below the substrate surface, allowing for the planar integration of sensor structures.

^{4.1} This chapter is based on an article published in Sensor. Actuat. A-Phys. [4.1].

4.2 Sensor Design

Figure 4.1 illustrates the design of the miniaturised calorimetric flow sensor. The sensor measures liquid flow through a freestanding microchannel crossing a thermal-isolation cavity over a length of $800\ \mu\text{m}$. Fluidic interfacing to the microchannel is made possible by directly applying O-rings on top of the microchannel inlet and outlet embedded in the substrate surface. The $5\ \text{mm}$ long microchannel is approximately $15\ \mu\text{m}$ deep and $80\ \mu\text{m}$ wide, having $4.5\ \text{nl}$ total volume and a hydraulic resistance as low as $6.17\ \text{kPa}/\mu\text{l}\cdot\text{min}^{-1}$ for water.

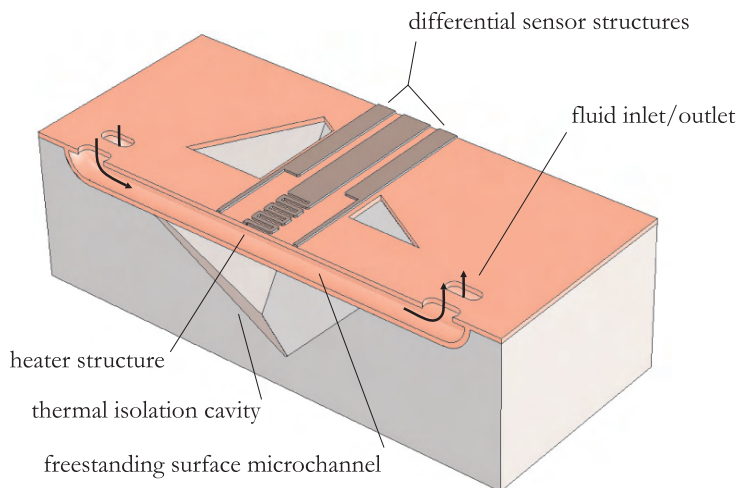


Fig. 4.1 Cross-section of calorimetric flow sensor design, with heater structure and differential sensor structures integrated on a freestanding surface microchannel

Fluid flow is measured by heating the fluid using a heater centered on the suspended microchannel. Heat conduction through the surrounding air, the microchannel and the electrical connection leads results in a symmetric temperature profile around the heater. The temperature profile is disturbed by forced convection on applying fluid flow through the microchannel. The disturbance does not affect the temperature of the heater for small flows, because there is no temperature gradient at the heater for the undisturbed symmetric temperature profile, required for the advection of heat.

The temperature disturbance is measured differentially by measuring the resistance change of two resistors located $40\ \mu\text{m}$ up- and downstream from the heater. The sensor output is linearly proportional to the flow rate for small flows, resulting in a highly-sensitive calorimetric flow sensor. The differential sensor output is insensitive to common-mode ambient temperature fluctuations and variations in fluid tempera-

ture [4.5]. The disturbed temperature profile upon increased fluid flow causes a gradient in temperature across the heater, leading to heat-loss at the heater, eventually decreasing the heater temperature, with the sensor output becoming non-linear.

4.3 Sensor Fabrication

The fabrication of the miniaturised calorimetric flow sensor is based on surface channel technology [4.6] as explained in Chap. 3. Surface channel technology is a good alternative, compared to surface micromachined channels, in realising mechanically strong freely suspended microchannels, without the requirement for a sacrificial layer. This allows for the integration of sensor structures in close proximity to the fluid, with improved fluidic interfacing to the microchannels.

Surface microchannels were created by isotropic dry etching, using high-density SF_6 plasma with zero self-bias, through small etch holes in a low-stress 500 nm SiRN (silicon-rich silicon nitride) layer. The coalescence of the etched surfaces resulted in semicircular channel profiles (Fig. 4.2a). Four parallel microchannels were merged into a single microchannel. This reduces the hydraulic resistance of the microchannel to $6.17 \text{ kPa}/\mu\text{l}\cdot\text{min}^{-1}$ for water and the microchannel membrane gives a sufficiently large platform for the integration of sensor structures.

The small etch holes and the inner surfaces of the microchannels are uniformly coated by the LPCVD deposition of a second low stress $1.3 \mu\text{m}$ SiRN layer (Fig 4.2b), thus creating completely sealed microchannels, while leaving a planar substrate surface

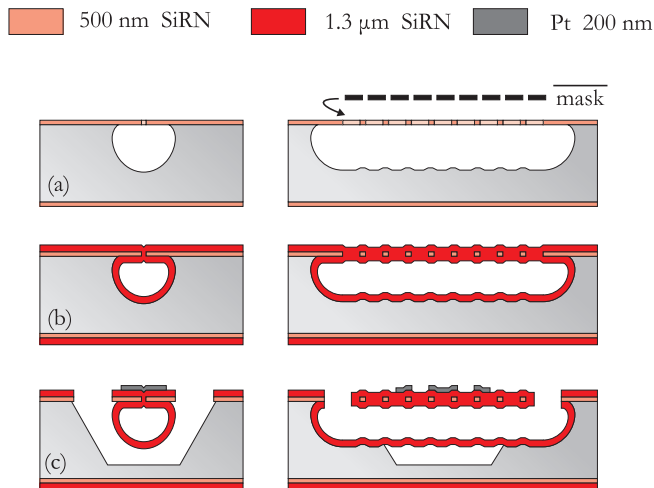


Fig. 4.2 Process scheme for the fabrication of the miniaturised calorimetric flow sensor using Pt sensor structures.

for further processing (Fig. 4.3). The semicircular channels are mechanical strong and are insensitive to deformation, possibly caused by the integration of sensor materials with high stress across the microchannel. Because of the resulting planar substrate surface, step-coverage problems are avoided and 200 nm thin-film Pt sensor structures could be deposited directly on top of the microchannels (Fig. 4.2c). The SiRN microchannel membrane was roughened with an SF₆-O₂ plasma for proper adhesion of the Pt without the need for an adhesion layer. The Pt was patterned by oxidation using O₂ plasma, with the thin oxidation layer acting as a mask during Aqua Regia etching [4.7], creating perfectly defined sensor structures (Fig. 4.4b).

Fluid entrance holes were etched through the SiRN layer (Fig. 4.2c), allowing for the direct interfacing to the microchannels on the frontside of the wafer. Thermal-isolation cavities are defined using the same mask (Fig. 4.2c) and are etched by KOH etchant. The microchannel and electrical contact leads are offset at a 30° angle to the <100>-direction, such that the Si {111}-planes can be etched underneath the freely-suspended structure, suspending the surface microchannels above the heat conducting substrate.

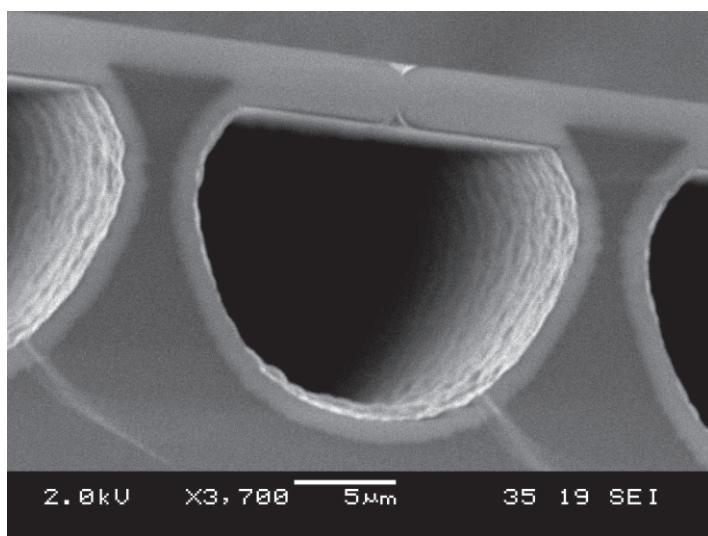
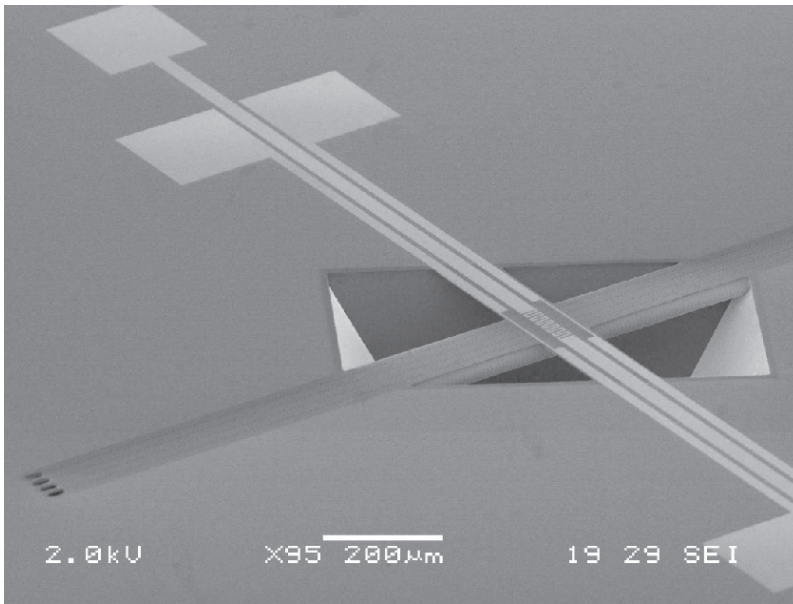
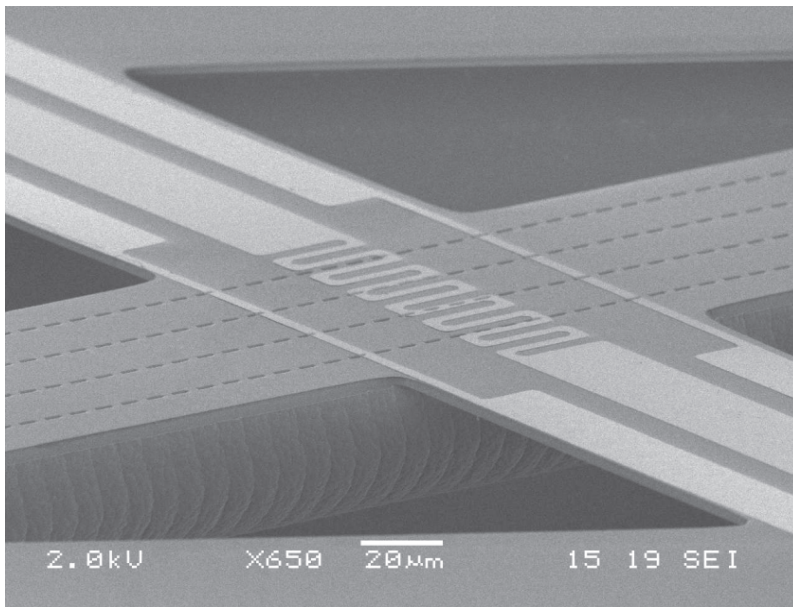


Fig. 4.3 Cross-section of sealed surface channels, leaving a planar substrate surface for further processing.



a)



b)

Fig. 4.4 Micrographs of a) calorimetric flow sensor suspended above thermal-isolation cavity and b) Pt resistors crossing surface microchannels.

4.4 Sensor Modelling

4.4.1 Fluidic-Thermal CFDRC Model

The temperature field surrounding the heater was solved with CFDRC [4.8] using a high-fidelity mesh of the sensor. The thermal field was solved numerically for the freely-suspended microchannel containing water, the electrical contact leads, the substrate and the surrounding air in the isolation cavity as well as part of the air above the sensor. Material properties were defined according to values in Tbl. 4.1. Boundary conditions were set to reside at ambient temperature. Uniform heating power-densities, determined by sheet-resistance and mask-layout data, were imposed on the heater and on the freely-suspended heater connection leads, with the heater dissipating 1.9 mW. A laminar flow profile was solved inside the microchannel for various water flow rates, visualising the disturbance on the temperature field by forced convection due to the fluid flowing through the surface microchannels.

Tbl. 4.1 Material properties used in model calculations, with κ the thermal conductivity, ρ the density, c the specific heat and μ the dynamic viscosity.

κ_{SiRN}	$3 \text{ W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$	κ_{water}	$0.6 \text{ W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$
κ_{Pr}	$72 \text{ W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$	ρ_{water}	$998 \text{ kg}\cdot\text{m}^{-3}$
κ_{Si}	$130 \text{ W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$	c_{water}	$4179 \text{ J}\cdot\text{kg}^{-1}\cdot\text{K}^{-1}$
κ_{air}	$0.024 \text{ W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$	μ_{water}	$1.12 \text{ N}\cdot\text{s}\cdot\text{m}^{-2}$

Figure 4.5 visualises the calculated asymmetric temperature field on the freely-suspended microchannel due to various water flow rates. It clearly shows that the temperature field extends into the surrounding air due to heat conduction. The temperature isosurface at $0 \mu\text{l}\cdot\text{min}^{-1}$ flow (Fig. 4.5a) is almost spherical. This means that the difference in heat flux through the microchannel and through the surrounding air is proportional to the differences in thermal conductivity. The heat loss to the surrounding air is however significant, because of the increase in surface area of air at larger distances from the heater, whereas the surface area of the microchannel remains constant. This is clear from the isosurface being more ellipsoidal near the heater and rapidly becoming spherical at larger distances from the heater.

The temperature field becomes asymmetric by forced convection on applying fluid flow through the microchannel. At $0.5 \mu\text{l}\cdot\text{min}^{-1}$ water flow rate the fluid inside the microchannel is fully heated to a radial-uniform temperature profile at small axial distances from the heater (Fig. 4.6a). The forced convection is thus dependent on the

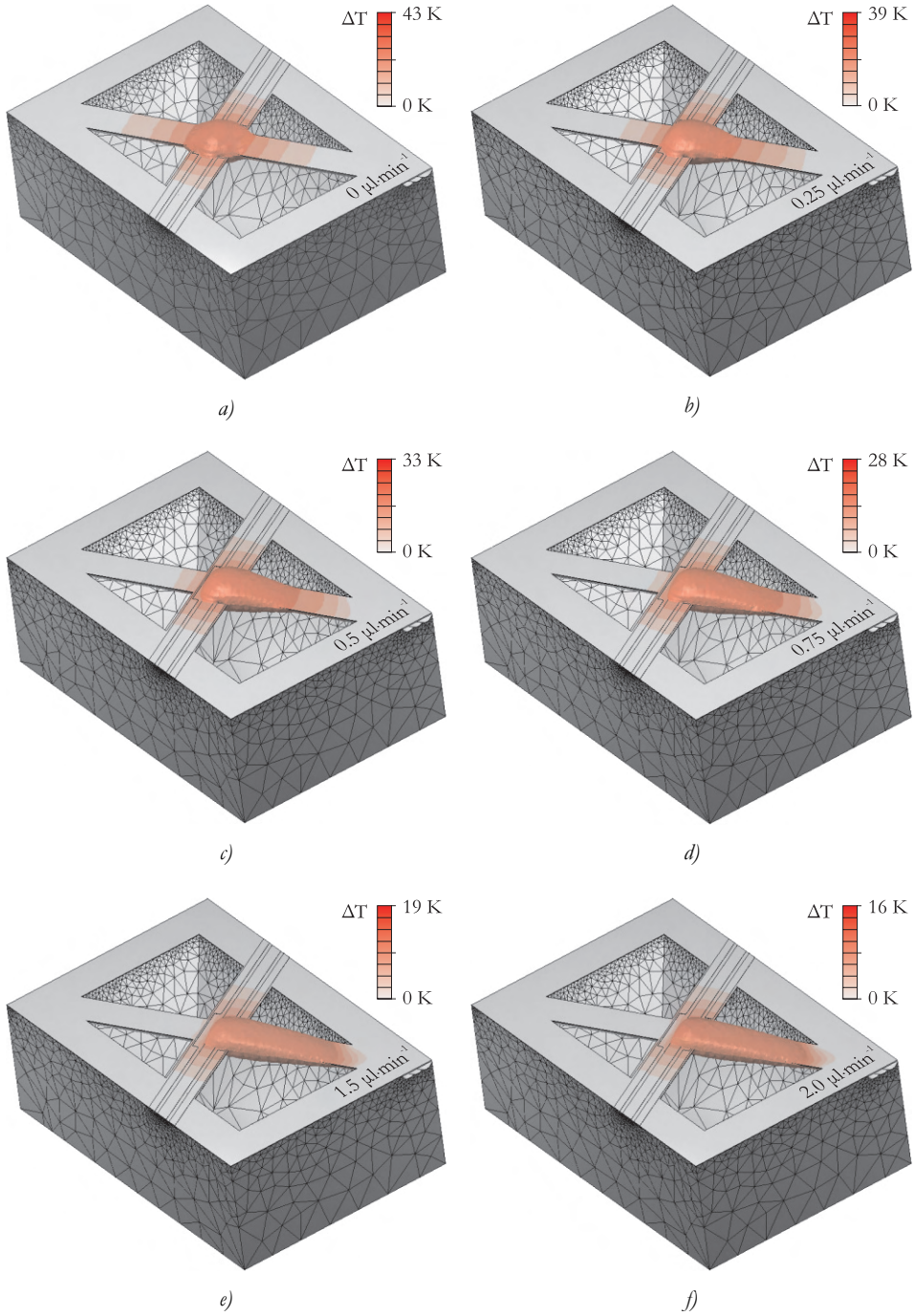
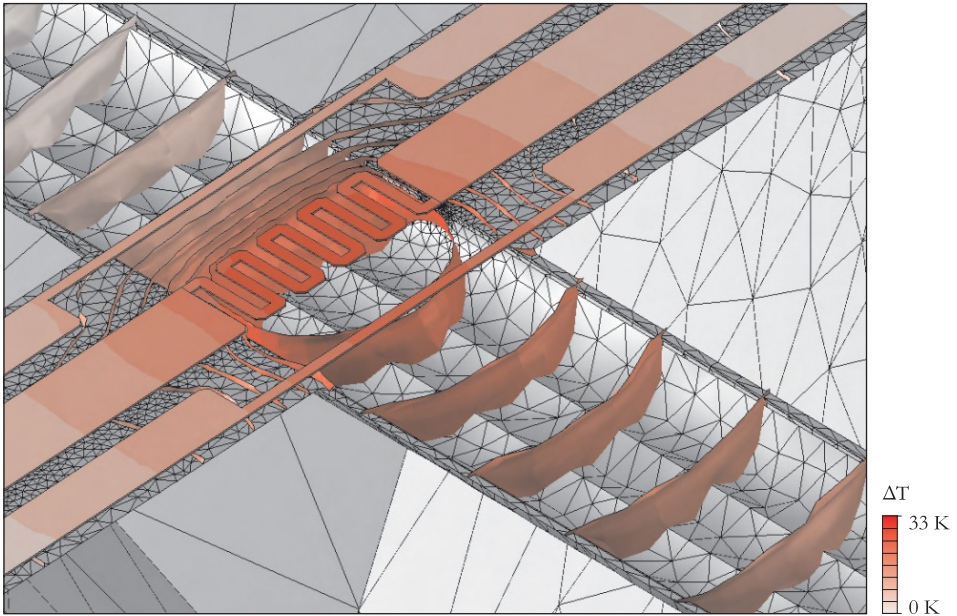
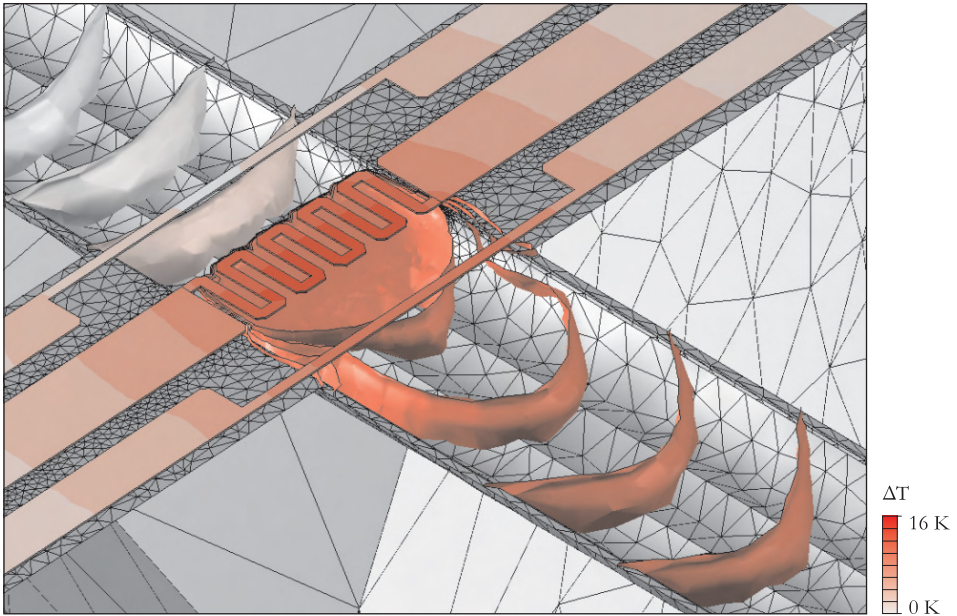


Fig. 4.5 CFDRC model results, showing asymmetric thermal profiles on the thermal flow sensor at various water flow rates applying 1.9 mW heating power. Isosurfaces at half-maximum temperature are shown.



a)



b)

Fig. 4.6 The temperature field inside the freely-suspended microchannel visualised by isosurfaces at 1.9 mW heating power at a) $0.5 \mu\text{l}\cdot\text{min}^{-1}$ water flow and b) $2 \mu\text{l}\cdot\text{min}^{-1}$ water flow, showing a visible thermal boundary layer.

total volume flow through the microchannel (Sec. 1.3.2). At $2 \mu\text{l}\cdot\text{min}^{-1}$ water flow a thermal boundary layer extends from the heater, but reaches the microchannel depth before reaching the downstream temperature sensor (Fig. 4.6b). The downstream sensor at $2 \mu\text{l}\cdot\text{min}^{-1}$ water flow measures a temperature, which is not determined by

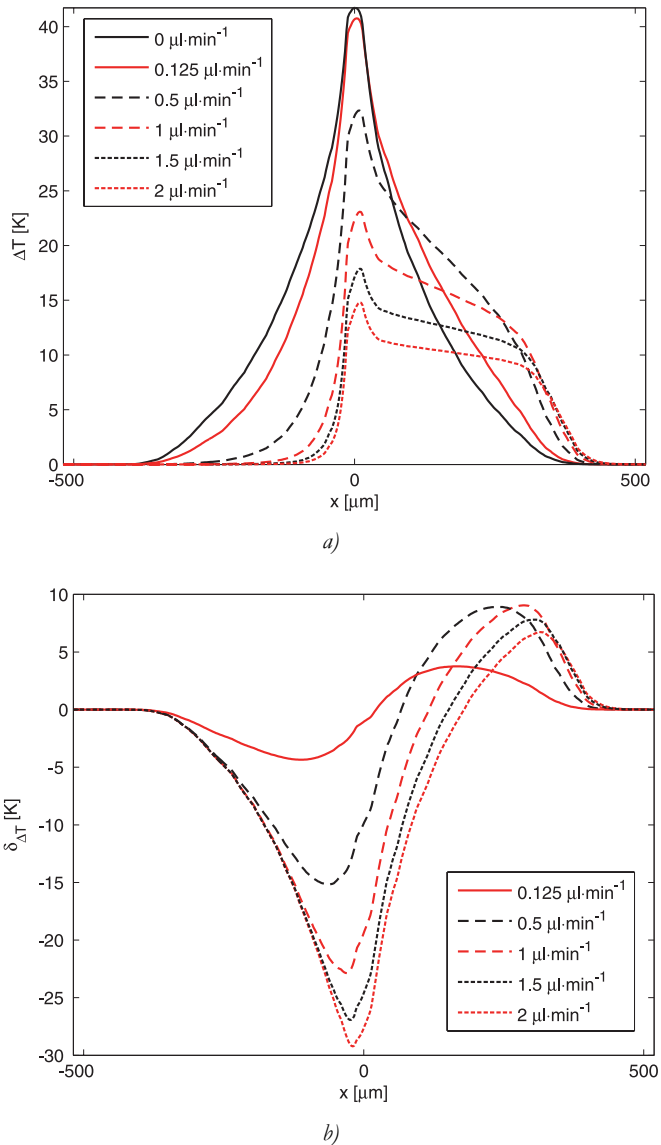


Fig. 4.7 a) Temperature profiles along the central axis of the microchannel, at various flow rates and b) temperature changes relative to the $0 \mu\text{l}\cdot\text{min}^{-1}$ temperature profile due to various flow rates. The microchannel is fixed to the substrate at $\pm 400 \mu\text{m}$.

the thermal boundary layer. Instead, the downstream sensor measures the average of the thermal profile along the width of the microchannel, visible from the large density of isosurfaces near the edge of the microchannel at the downstream sensor and heater. This indicates a relatively large heat loss to the heater and sensor contact leads at $2 \mu\text{l}\cdot\text{min}^{-1}$ water flow rate. A better solution is to have the contact leads positioned along the microchannel (Chap. 5).

The numerical model was solved for various water flow rates from $0 \mu\text{l}\cdot\text{min}^{-1}$ up to $2 \mu\text{l}\cdot\text{min}^{-1}$. Figure 4.7 shows the resulting temperature profiles and the temperature changes by heat convection on the freely-suspended surface microchannel, with the heater centred at $0 \mu\text{m}$. As expected, the heater temperature for small flows is hardly affected and the disturbance by the flow increases the temperature downstream from the heater as much as it decreases the temperature upstream. The numerical results in Fig 4.7b indicate that the optimal position of the temperature sensors is halve way between the heater and the end of the suspended part of the microchannel. The temperature profiles decrease with decreasing heater temperature at higher flow rates, with a rapid decrease in temperature between $300 \mu\text{m}$ and $400 \mu\text{m}$, because there the microchannel is in close proximity to the silicon substrate.

4.4.2 Thermal FVM Model

The CFDRC model shows that the temperature field along the microchannel is nearly symmetric around the central axis, where the magnitude of the thermal gradient from the microchannel to the surrounding air is largely being determined by the shortest radial distance to the silicon substrate. The spherical shape of the temperature

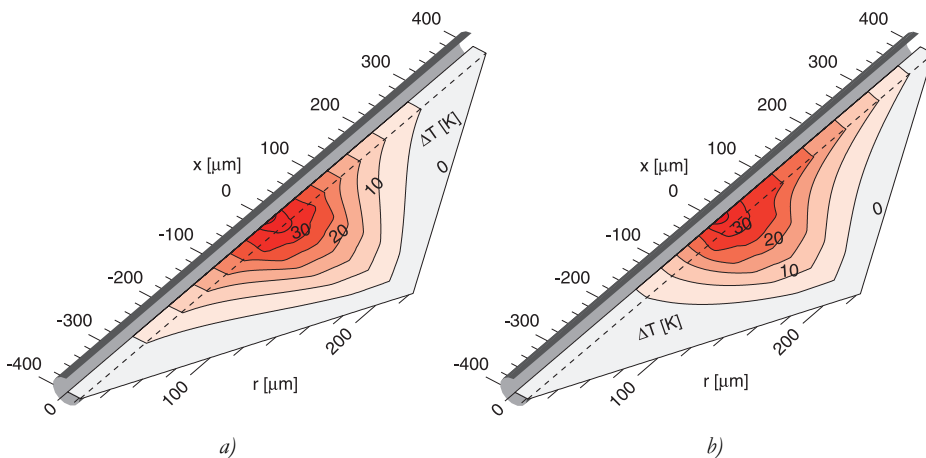


Fig. 4.8 Temperature field obtained from FVM model calculations a) at no flow applied and b) with $0.25 \mu\text{l}\cdot\text{min}^{-1}$ water flow applied.

isosurface in Fig. 4.5a indicates that heat fluxes in both x -direction and r -direction have to be considered. These conditions allow for a fast two-dimensional cylindrical-symmetric model of the sensor giving accurate quantitative results. Such a thermal model was constructed in MATLAB [4.9], applying the finite volume method [4.10]. Appropriate boundary conditions and material parameters (Tbl. 4.1) were defined.

Figure 4.8 shows the temperature field resulting from FVM model calculations, with no flow and $0.25 \mu\text{l}\cdot\text{min}^{-1}$ water flow applied at 1.9 mW heating power. The mesh boundary in r -direction is approximated by taking the shortest radial distance to the thermal isolation cavity at each x -position along the microchannel. The contact leads are modelled by an approximated thermal conductivity function κ_{lead} dependent on r .

Figure 4.9 shows the temperature profiles from FVM model calculations. The FVM model gives equivalent results to CFDRC for flow rates up to $500 \text{ nl}\cdot\text{min}^{-1}$. The temperature profiles at higher flow rates differ slightly from CFDRC results shown in Fig. 4.7a, especially near the microchannel suspension due to the close proximity to the silicon substrate. The profiles near the heater are also slightly different from CFDRC results, showing the approximation made by the thermal conductivity function κ_{lead} for the contact leads. The FVM model overall gives fast approximate solutions to the temperature field, giving sufficiently accurate quantitative results.

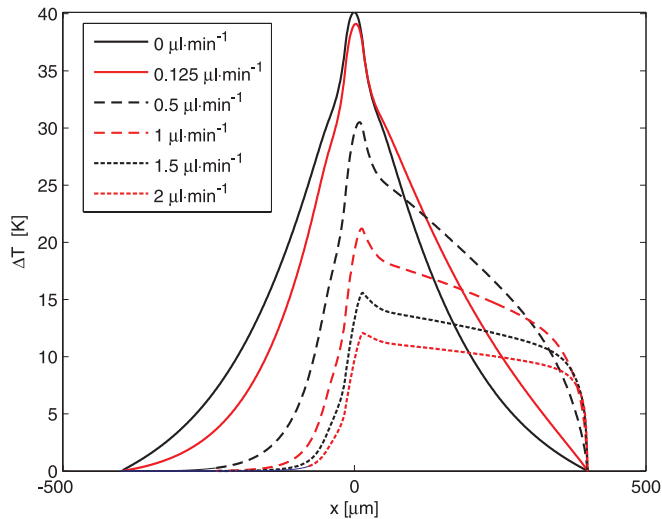


Fig. 4.9 Temperature profiles along the central axis of the microchannel, at various flow rates.

4.5 Experimental Setup

Measurements were performed using a pressure driven flow setup as indicated in Fig. 4.10. Water was forced through the microchannel by applying pressure from a N_2 gas cylinder. The flow rate was determined for each measurement point by measuring the advancing meniscus in the fluidic connection tube, which contains approximately $0.5 \mu\text{l}\cdot\text{mm}^{-1}$, after applying a constant pressure for several minutes. The flow was kept constant by regulating a membrane valve to a fixed pressure in the range from 0 to 1 bar.

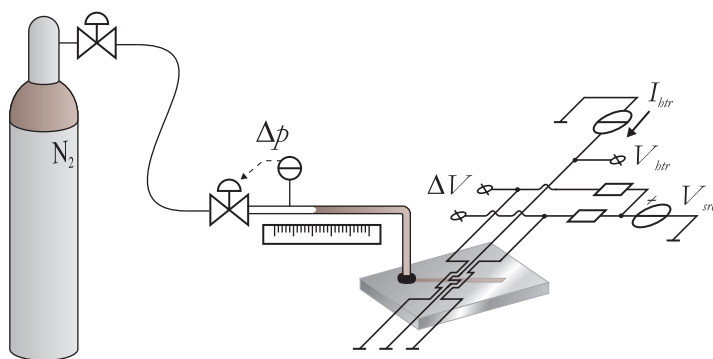


Fig. 4.10 Schematic of experimental setup. Water flow is forced through the sensor chip by a regulated pressure Δp from a N_2 gas cylinder. The heater resistor is heated by a constant current I_{htr} and the sensor output ΔV is obtained using a Wheatstone bridge.

Water flow was measured by applying a current I_{htr} of 5 mA to the heater resistor dissipating 1.9 mW. The resistance change of the heater upon fluid flow is measured and can be related to the temperature change of the heater according to $\Delta R = \alpha \cdot \Delta T \cdot R_0$, with α the temperature coefficient of resistivity and R_0 the resistance value at ambient temperature. Van der Pauw structures were calibrated against a PT100 temperature sensor applying a temperature sweep from 20 °C up to 120 °C, measuring temperature coefficient of resistivity $\alpha = 0.0024 \text{ K}^{-1}$ and sheet resistance $R_{0,S} = 0.83 \Omega \cdot \square^{-1}$ for the 200 nm Pt sputtered layer. The sensor resistors are placed in a Wheatstone bridge, applying a voltage V_{src} of 0.1 V, where a differential output voltage ΔV is obtained, which is nearly linearly proportional to the resistance change of both sensor resistors.

4.6 Experimental and Model Results

Figure 4.11a shows the measured resistance change for water flow rates from $0 \mu\text{l}\cdot\text{min}^{-1}$ up to $2 \mu\text{l}\cdot\text{min}^{-1}$. The average change in temperature of the heater, obtained from the numerical models, is plotted for comparison. The average temperature

measured by the sensor structures is for a small part determined by the contact leads, because the heater and sensor resistors are not measured using four-point contacts. Figure 4.11b shows the simultaneously measured differential voltage output of the Wheatstone bridge compared to numerical results. As expected, the heater tempera-

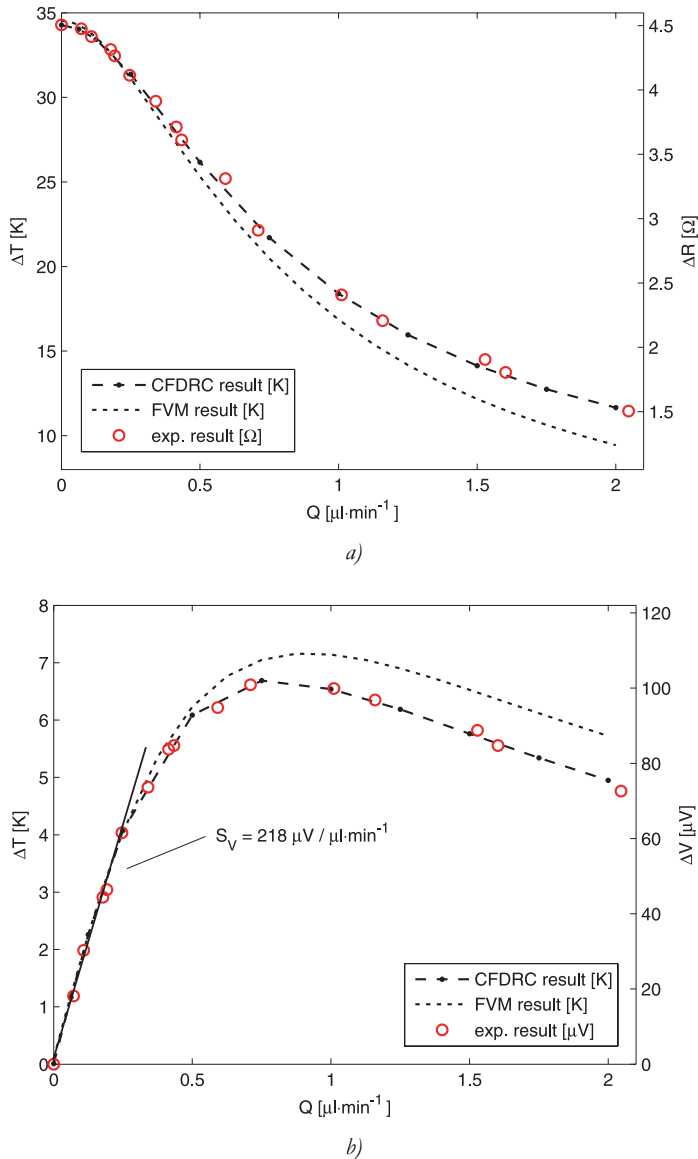


Fig. 4.11 a) Measured resistance change of the heater resistor compared to the average temperature of the heater obtained from the numerical models and b) measured voltage output of the Wheatstone bridge compared to the average differential temperature between both sensor resistors obtained from the numerical models.

ture is nearly constant at small flow rates with a linear sensor response ΔV having a flow sensitivity of $S_V = 218 \mu\text{V}/\mu\text{l}\cdot\text{min}^{-1}$. The sensor output remains linear up to flow rates in the order of $300 \text{ nl}\cdot\text{min}^{-1}$.

The experimental and numerical results are in close agreement. However, R_0 in the numerical models is calculated from sheet-resistance and mask-layout data, while the actual R_0 of the heater and sensor resistors is somewhat different and is slightly influenced by temperature. Therefore, the experimental results were scaled to fit the numerical data. Obtained values of R_0 after scaling are close to expected values from mask-layout and sheet-resistance data as well as four-point resistance measurement of the heater resistor between bondpads. The experimental data fits accurately to CFDRC results and differs only at higher flow rates for the FVM model.

Figure 4.12 shows additional measurements for water flow below $300 \text{ nl}\cdot\text{min}^{-1}$. A fluid flow restriction was used in the measurement setup for flow rates below $100 \text{ nl}\cdot\text{min}^{-1}$, allowing to measure at higher pressures, with higher calibration precision in determining the flow rate. Changes to the fluidic connection tubes introduced a systematic error in flow rate accuracy giving a different sensor sensitivity $S_V = 203 \mu\text{V}/\mu\text{l}\cdot\text{min}^{-1}$ compared to measurement results in Fig. 4.11. The flow rate became unstable for pressures approaching zero, which was found to be the limiting factor in measuring liquid flow rates below $40 \text{ nl}\cdot\text{min}^{-1}$, with the pressure driven flow setup used.

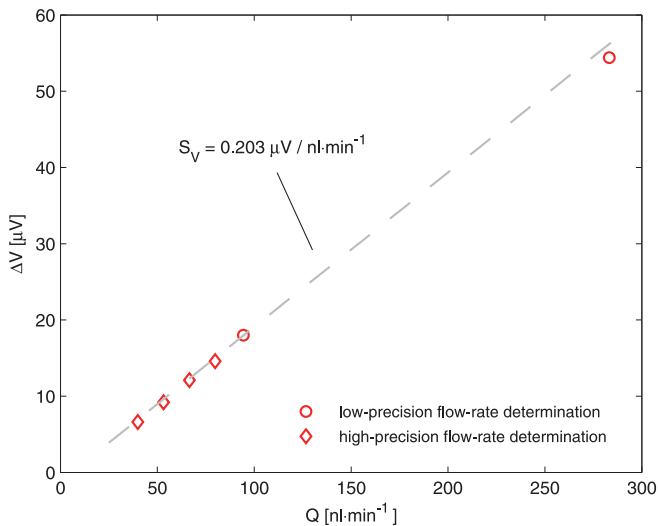


Fig. 4.12 Linear sensor output for water flow below $\text{nl}\cdot\text{min}^{-1}$, with $\tau_{\text{sample}} = 1 \text{ s}$. A flow restriction was used to determine the flow rate, with higher calibration precision below $100 \text{ nl}\cdot\text{min}^{-1}$.

4.7 Conclusions

A miniaturised calorimetric flow sensor was realised using a versatile surface channel concept, demonstrating the capability of the technology for use in thermal flow sensor applications, where the microchannel is thermally isolated from the substrate. The concept allows for the fabrication of microchannels with low hydraulic resistance and total fluid volume, directly below the substrate surface, while sensor structures are deposited on a planar substrate surface avoiding step-coverage problems. The surface microchannel technology can be used in the integration of other sensor materials, like materials with high temperature coefficient of resistivity [4.3, 4.4] or the integration of thermopiles.

A linear differential calorimetric sensor response was measured for water flow up to flow rates in the order of 300 nl·min⁻¹. The measured sensor output is in close agreement with results obtained from the CFDRC model and the approximate FVM model of the sensor, with the FVM model giving direct accurate results in the linear regime of the calorimetric flow sensor. The highly-accurate CFDRC model shows that the conduction through the surrounding air is significant for the calculation of the temperature profiles along the microchannel. The output of the sensor and the heater temperature give the typical response of a calorimetric flow sensor, however the CFDRC model shows that this is not due to a thermal boundary layer in the fluid. It is through the asymmetric temperature field in the surrounding air by advection of heat in the microchannel, partially measured by the contact leads, that the sensor obtains a King's law behaviour.

The FVM model is sufficiently accurate to be used with other sensor designs. The Pt contact leads resulted in a radial heat loss, which is significant close to the microchannels and can be avoided by running the leads over the microchannel.

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5

Low-Drift Flow Sensor with Thermopile-Based Power Feedback

A thermal flow sensor has been realised consisting of freely suspended silicon-rich silicon-nitride microchannels with an integrated Al/poly-Si⁺⁺ thermopile in combination with up- and downstream Al heater resistors. The zero offset of a thermopile at uniform temperature is exploited in a feedback loop controlling the dissipated powers in the heater resistors, minimising inevitable influences of resistance drift, mismatch of thin-film metal resistors and thermopile material drift. The control system cancels the flow-induced temperature difference across the thermopile by controlling a power difference between both heater resistors, thereby giving a measure of the flow rate. The flow sensor was characterised for power difference versus water flow rates up to 1.5 $\mu\text{l}\cdot\text{min}^{-1}$. Results are in good agreement with a thermal model of the sensor, and the low-drift operation of the temperature-balancing control system was verified.^{5.1}

5.1 Introduction

The miniaturisation of microfluidic components in chemical analysis, synthesis, biotech and nanotechnology asks for the accurate and reliable measurement of tiny fluid flow rates. Current micromechanical thermal flow sensors are capable of measuring liquid flows down to a few $\text{nl}\cdot\text{min}^{-1}$ [5.1, 5.2]. However, the long-term measurement accuracy of these sensors is ultimately limited by drift in the properties of temperature sensing and heating elements, for which thin-film layers are commonly used [5.3, 5.4]. Alternatively, thermopiles can be used as differential temperature sensors [5.5-5.8].

This chapter discusses a thermal flow sensor and control system that exploits the low offset-drift differential-temperature dependent output V_{TC} of a thermopile [5.9] in order to obtain a flow sensor that is independent of resistance drift and mismatch of thin-film metal resistors used (Fig. 5.1). The dissipated powers P_1 , P_2 in the heater resistors up- and downstream from the thermopile are determined and controlled independently of resistance values by measuring both the current through and the voltage across the resistors, thereby eliminating influences of resistance drift. Additionally, the control-system feedback loop minimises the dependency on non-

^{5.1} This chapter is based on an article accepted for publication in J. Microelectromech. S.

linear thermopile characteristics $m_{TC}(\Delta T_{TC})$ and drift in thermocouple material properties by cancelling the generated output voltage V_{TC} .

The output voltage V_{TC} and temperature difference across the thermopile is cancelled by controlling a power difference ΔP between the two heater resistors, with a fixed amount of total power P_T being dissipated, according to the temperature-balancing anemometry principle [5.10, 5.11]. Applying a fluid flow changes the temperature distribution around the heaters by forced convection. This results in the relation $\Delta P/P_T$ being dependent on the flow rate Q . A measure of the flow rate $m_{TB}(\Delta P/P_T)$ can be obtained, with $\Delta P/P_T$ being linear for small flow rates.

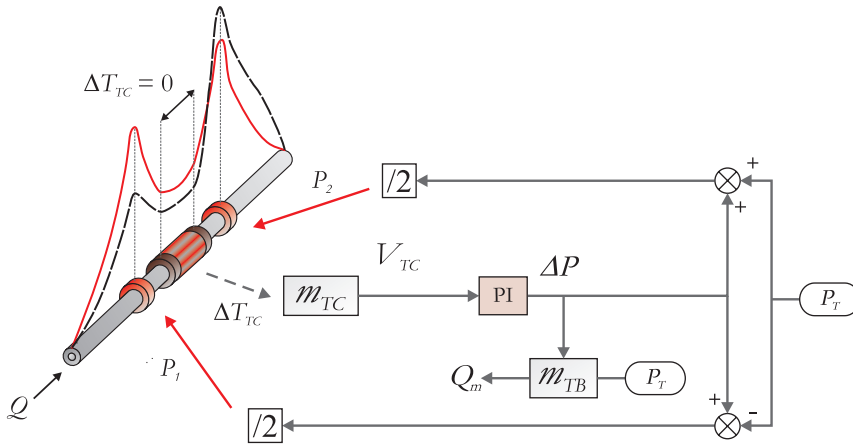


Fig. 5.1 Thermopile-based temperature-balancing flow sensor.

Ultimately, the proposed measurement concept can be realised using standard electronics, AD/DA-converters and a microcontroller. This can lead to fully integrated flow sensors with improved accuracy for flow rates in the $\text{nl}\cdot\text{min}^{-1}$ range. In this chapter results of liquid flow measurements on a fabricated thermopile flow sensor using a temperature-balancing control system are discussed and compared with thermal model calculations. Long-term drift performance is analysed, where the effect of long-term drift in material properties is observed to be cancelled in the sensor output signal. Other influences on flow sensor output drift are also discussed.

5.2 Sensor Fabrication

The fabricated thermopile flow sensor incorporates mechanically-strong freely suspended surface microchannels for on-chip transport of tiny amounts of fluid flow. Surface microchannels are fabricated without the requirement for a sacrificial layer

and allow for the integration of sensor elements in close proximity to the fluid [5.12] as explained in Chap. 3.

Figure 5.2 gives a schematic overview of the process scheme for the fabrication of the low-drift thermopile flow sensor. Surface microchannels are created by isotropic dry etching, using high-density SF_6 plasma with zero self-bias. The microchannels are etched through etch holes $2\ \mu\text{m}$ in width, in a low-stress $500\ \text{nm}$ silicon-rich silicon-nitride (SiRN) layer. The etch holes and inner surfaces of the microchannels are conformally coated by a second low-stress LPCVD deposited $1.3\ \mu\text{m}$ SiRN layer, resulting in completely sealed microchannels, while leaving a planar substrate surface for the integration of an Al/poly-Si $^{++}$ thermopile and Al heater resistors. The thermopile is created by LPCVD deposition and boron doping by solid source diffusion of a $200\ \text{nm}$ poly-Si layer and sputtering of a $200\ \text{nm}$ Al layer (Fig. 5.2a).

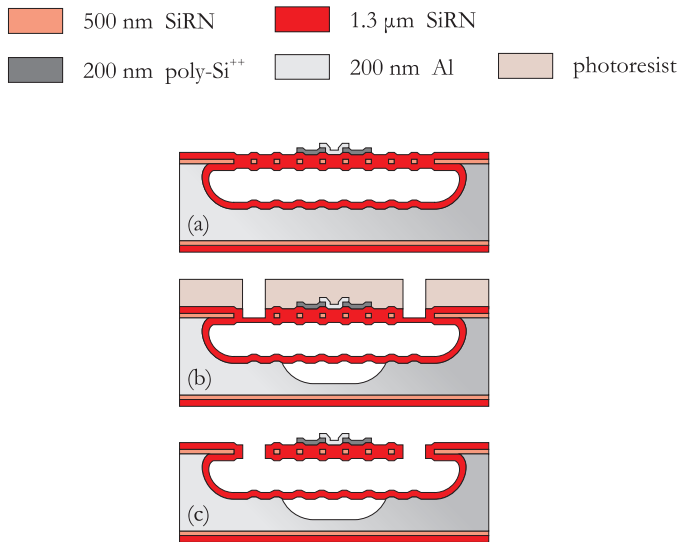
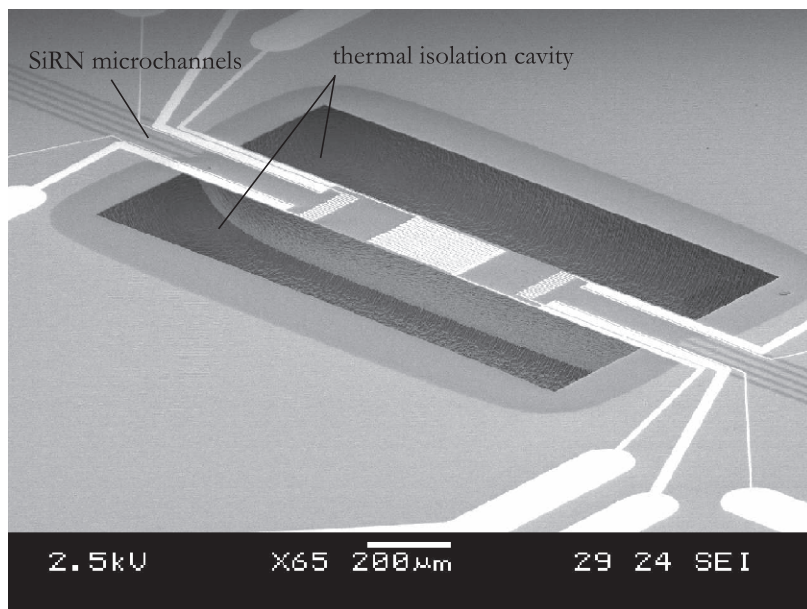


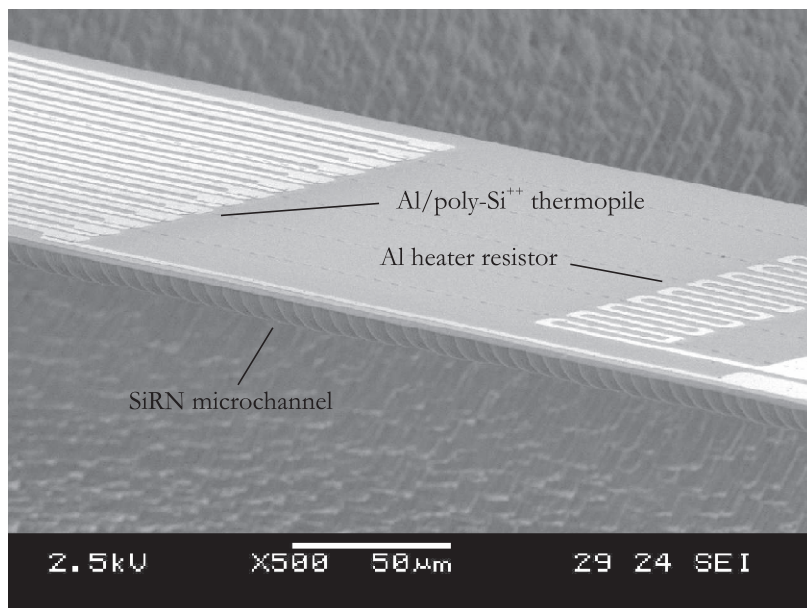
Fig. 5.2 Process scheme for the fabrication of the microchannel thermopile flow sensor.

The surface microchannels are released by SF_6 plasma etching, for thermal isolation from the heat-conducting substrate, with the photoresist mask protecting the sensor elements during the release (Fig. 5.2b). The photoresist is removed after fluidic entrance holes are etched through the SiRN layer, allowing for direct interfacing to the microchannels on the substrate surface.

Figure 5.3 shows SEM micrographs of the fabricated thermopile flow sensor. The sensor contains five parallel $20\ \mu\text{m}$ diameter microchannels, spanning a $1.6\ \text{mm}$ long thermal isolation cavity. The parallel microchannels have $200\ \mu\text{m}$ total width, which allows for the integration of an Al/poly-Si $^{++}$ thermopile (Fig. 5.4) with 26 junctions



a)



b)

Fig. 5.3 Microchannel flow sensor with integrated Al heater resistors and Al/poly-Si⁺⁺ thermopile.

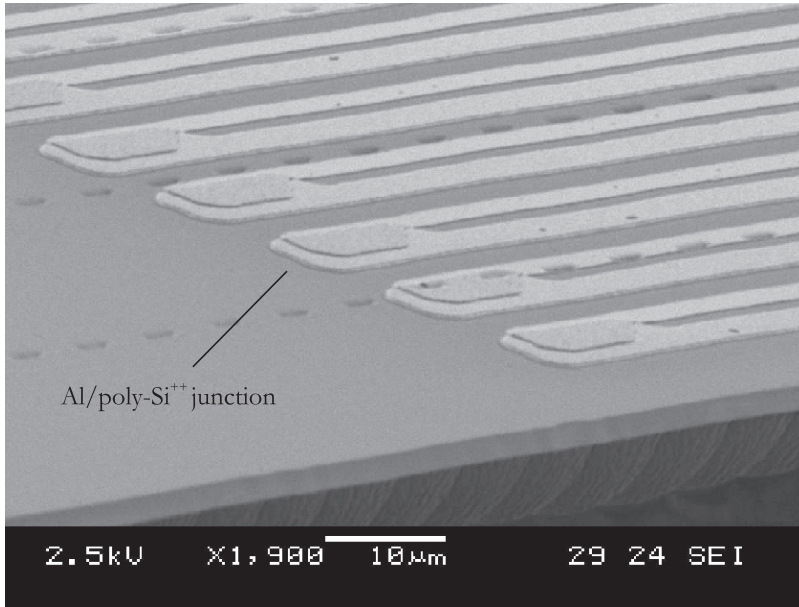


Fig. 5.4 Integration of Al/poly-Si⁺⁺ thermopiles on microchannel membrane.

and up- and downstream Al heater resistors. The Al heater resistors have four-point measurement contacts for accurate resistance measurements.

5.3 Sensor Modelling

A two-dimensional cylindrical finite volume method (FVM) thermal model [5.13] of the flow sensor was constructed in MATLAB. The model gives approximate solutions to the heat advection-diffusion equation (5.1) solving the temperature field T surrounding the freely-suspended microchannel, taking into account the thermal conductivities κ of the freely-suspended microchannel containing fluid, the sensor elements, electrical connection leads and surrounding air. Ambient temperature is imposed as a boundary condition at a radius of $450\ \mu\text{m}$ from the microchannel, modelling the under-etched thermal isolation cavity. A fully-developed fluid flow profile v_x is modelled inside the microchannel along the x -direction, with fluid density ρ and fluid specific heat capacity c . The heater elements are modelled by a heat-flow source term per unit volume P''' .

$$\left(c \rho v_x \frac{\partial}{\partial x} - \nabla \cdot (\kappa \nabla) \right) T = P''' \quad (5.1)$$

Equation (5.1) can be solved on a cylindrical mesh of the flow sensor applying Gauss theorem. This results in a sparse-matrix equation (5.2), with convection matrix \mathbf{GQ} , volume flow rate Q and conduction matrix \mathbf{Gk} , where \vec{T}_e and \vec{P}_e are column vectors of respectively the temperature and heat-flow source term of each element in

$$\begin{aligned} (Q \cdot \mathbf{GQ} - \mathbf{Gk}) \vec{T}_e &= \vec{P}_e \\ \mathbf{G} \vec{T}_e &= \vec{P}_e \end{aligned} \quad (5.2)$$

the cylindrical mesh. Equation (5.2) can directly be solved in MATLAB, using the Gauss-Seidel method.

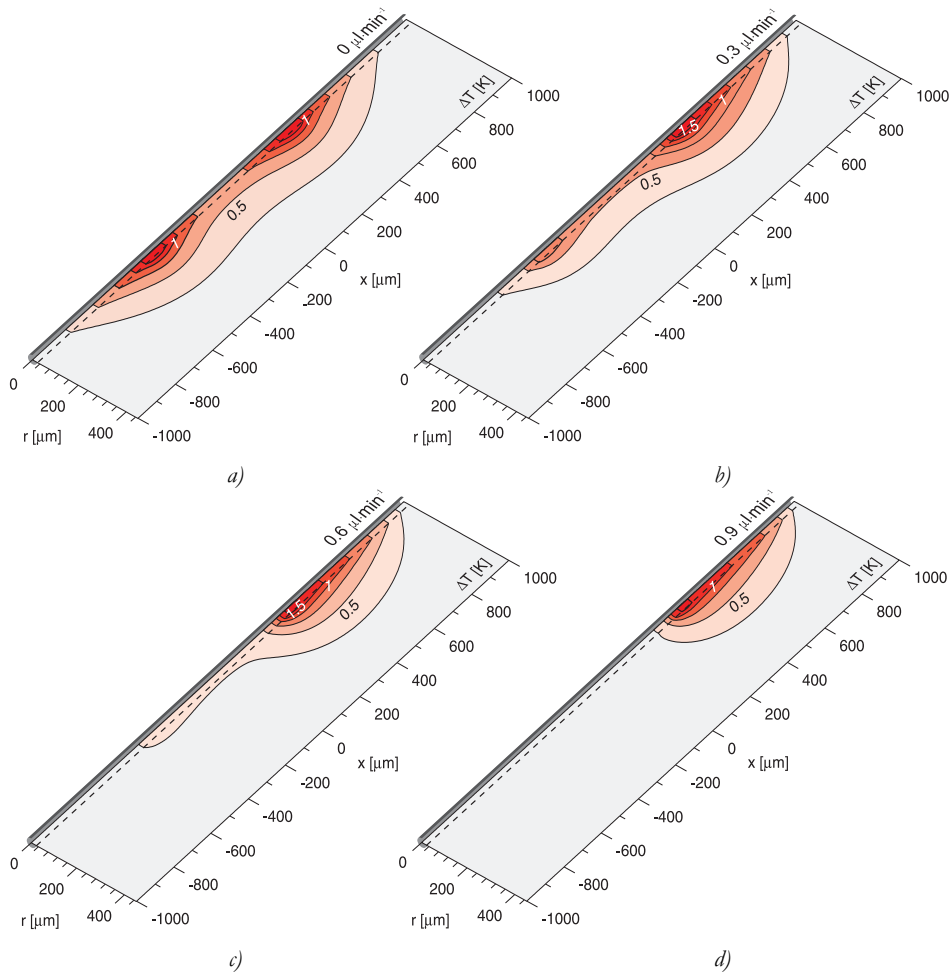


Fig. 5.5 FVM thermal model result, showing the temperature field at various water flow rates in positive x -direction, with 0.1 mW total power P_T .

The model allows for fast successive iteration at a given flow rate in order to find the power difference ΔP required for both sides of the thermopile to be at the same temperature, in accordance with the temperature-balancing measurement concept [5.10].

Figure 5.5 shows the temperature field at $0.3 \mu\text{l}\cdot\text{min}^{-1}$ water flow, with 0.1 mW total power dissipation. Several temperature profiles, along the freely-suspended microchannel, were obtained from model calculations at different flow rates (Fig. 5.6).

The temperature profile at $0 \mu\text{l}\cdot\text{min}^{-1}$ is symmetric, with both heaters dissipating equal amount of power. The temperature profile around the heaters is disturbed by advection of heat on applying water flow through the microchannel. Maintaining zero temperature difference between both sides of the thermopile ΔT_{TC} requires the upstream heater to dissipate less power than the downstream heater.

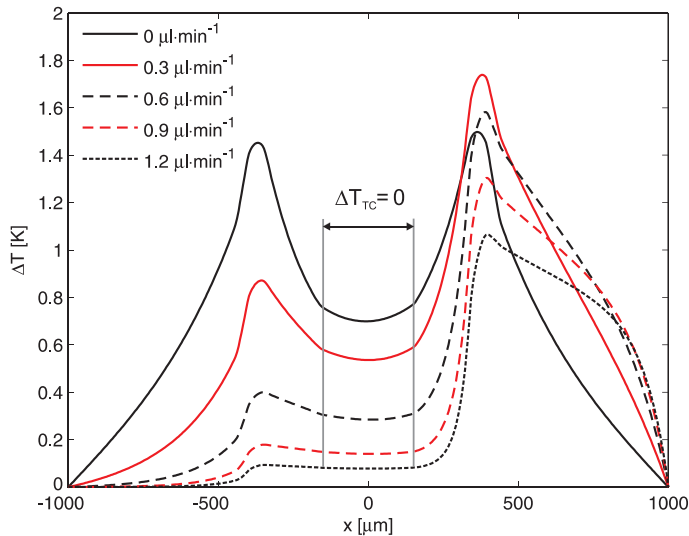


Fig. 5.6 FVM model calculation results giving temperature profiles along the freely-suspended microchannel, with water flow in the positive x -direction.

The upstream heater temperature decreases with increasing flow rate and decreasing heating power. However, the downstream heater does not increase in temperature for water flow rates higher than $0.6 \mu\text{l}\cdot\text{min}^{-1}$, due to the total power P_T being limited, while a large amount of heat is advected from the heater, limiting the sensitivity of the flow sensor for flow rates higher than $0.6 \mu\text{l}\cdot\text{min}^{-1}$.

5.4 Experimental Results

Thermopile flow sensor chips were fabricated 12.5×12.5 mm in size, with fixed bondpad and microchannel entrance hole positions for self alignment with a chip holder, containing O-rings for fluidic interfacing and twelve pogo-pins for electrical connections to the sensor chip (Fig. 5.7).

An elevation head up to a few meters Δb was used to force water through the thermopile flow sensor (Fig. 5.8). The flow is restricted to a few $\mu\text{l}\cdot\text{min}^{-1}$ by the hydraulic resistance of the surface microchannels. The elevation head gives a stable flow rate, which is determined by weighing. The volume flow rate is calculated by taking the time-derivative of the amount of liquid flowing onto a scale, divided by the density of water.

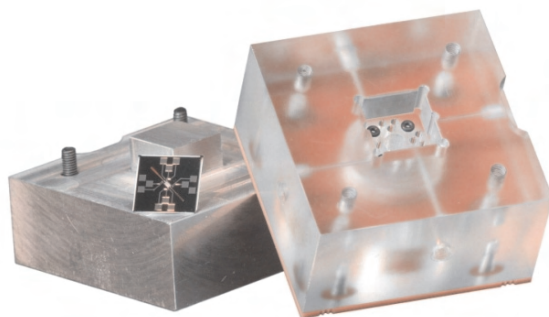


Fig. 5.7 Chip holder for fluidic and electrical interfacing with the sensor chip. The sensor chip measures 12.5×12.5 mm.

The power feedback loop and temperature-balancing controller were implemented in MATLAB. A Hewlett Packard 34401A multimeter was used to measure the thermopile output voltage V_{TC} . The heating powers P_1, P_2 dissipated in the resistors are controlled by a Keithley 2602 two-channel source measurement unit (SMU). The SMUs measure the voltage across the heater resistors and the dissipated power is controlled by adjusting the current through the resistors. In this way the exact resistance value is not important and may drift.

The total heating power P_T was kept constant at 0.1 mW during these experiments, resulting in a temperature rise of a few Kelvin of the heater resistors with respect to the substrate. The difference in heating power ΔP needed to cancel the thermopile output voltage V_{TC} was measured for applied water volume flow rates Q up to $1.5 \mu\text{l}\cdot\text{min}^{-1}$. Figure 5.9 shows the measured $\Delta P/P_T$ ratio, together with results obtained from FVM model calculations. FVM model and measurement results show

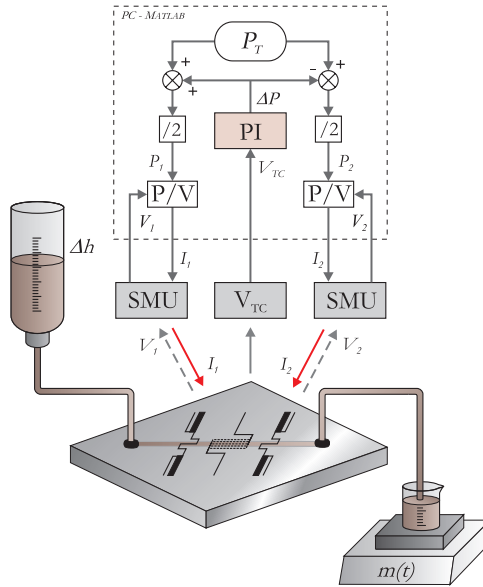


Fig. 5.8 Schematic overview of experimental setup, applying temperature-balancing power feedback control with $\Delta P/P_T$ being the flow dependent sensor output.

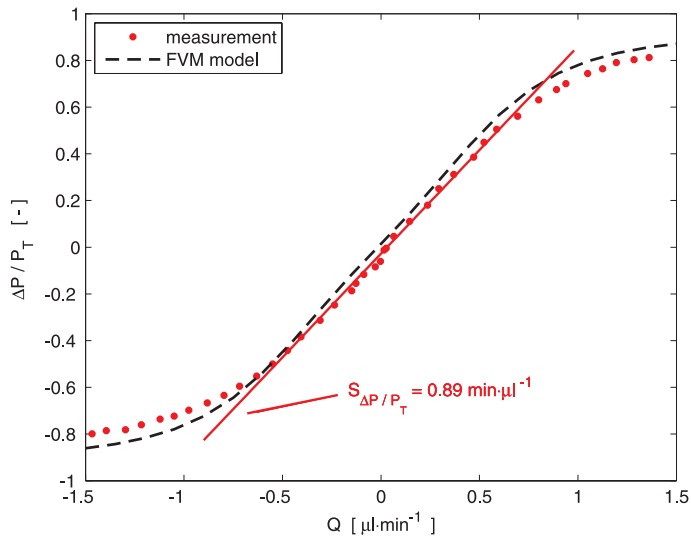


Fig. 5.9 Measured sensor output $\Delta P/P_T$ as a function of water volume flow rate Q . The total beating power P_T is kept constant at 0.1 mW.

the sensor having a linear response, with $0.89 \mu\text{l}\cdot\text{min}^{-1}$ sensitivity $S_{\Delta P/P_T}$, up to about $0.5 \mu\text{l}\cdot\text{min}^{-1}$.

Figure 5.10 shows the control system response on applying a stepwise changing volume flow rate. It shows the measured heating powers P_1, P_2 being adjusted in response to the flow. Correct operation of controlling the dissipated powers in the heater resistors is demonstrated by the fact that a change in the resistance values R_1 and R_2 , mostly due to a change in ΔP , is not visible in the output signal. The output signal $\Delta P/P_T$ is a measure of the flow rate, with the thermopile output voltage V_{TC} being controlled back to zero after each step in the flow rate. The discrete-sampling time of the controller is mostly determined by the integration time required to sample the thermopile voltage down to the required precision. The settling time of the PI controller required to adjust to a new flow rate is about 30 s, which is too slow for thermal flow sensor applications, although PI controller settings were not fully optimised. A state-free controller can be used, based on a linear least square method,

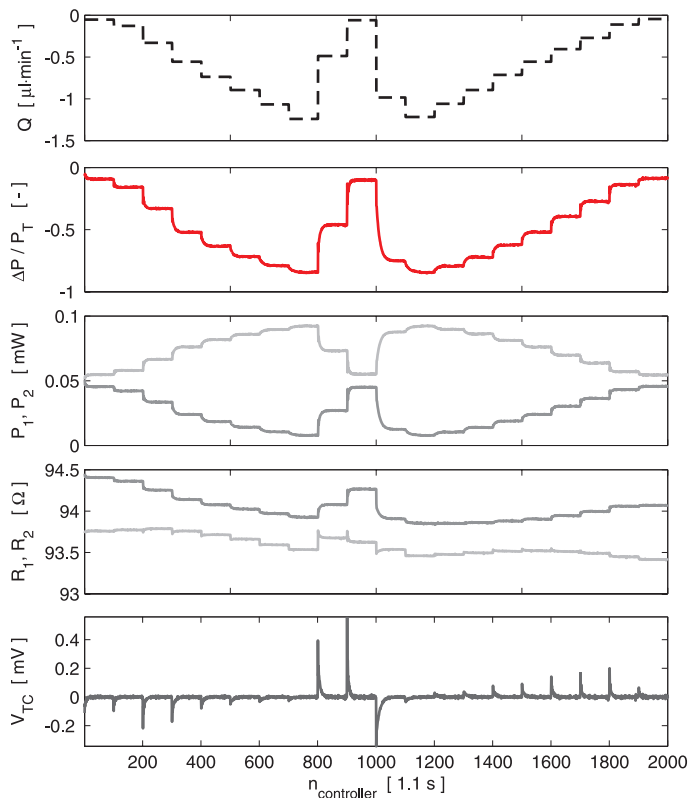


Fig. 5.10 Measured sensor control system response with changing flow rate. The total heating power P_T is kept constant at 0.1 mW.

which would require at least two thermopile voltages V_{TC} to be measured at different power differences ΔP .

The low-drift operation of the temperature-balancing control system was analysed in an ambient surrounding by controlling ΔP for 13 hrs with no flow applied, while dissipating 1 mW total power P_T . A Hewlett Packard 34420A nanovolt meter was used to measure the thermopile output voltage V_{TC} . Figure 5.11 shows obtained normalised power spectral densities. The power spectral densities of the average of both heater resistors R_μ and the difference between both heater resistors R_Δ show a $1/f^\alpha$ slope. The drift in R_μ can mostly be attributed to fluctuations in ambient temperature, while drift in R_Δ is caused by degradation (Sec. 2.2.1). The integral action of the PI controller cancels low frequency components in the thermopile output voltage V_{TC} , resulting in a smaller power spectral density at lower frequencies. The power spectral density for ΔP is nearly flat, where the power control on the heater resistors is most effective in reducing output drift of the flow sensor.

The power difference ΔP shows an offset, which can be explained by heat-conduction asymmetries in the sensor, internal offsets in the measurement equipment and thermocouple offset voltages on contact lead and bondwire leads required for electrical interfacing to the sensor chip and due to thermopile material irregularities. As well as a non-uniform Seebeck profile along the leads of the thermopile. The thermocouple voltages on the heater resistors are compensated by the control system, by averaging the voltage measured across the resistor, while alternating the direction

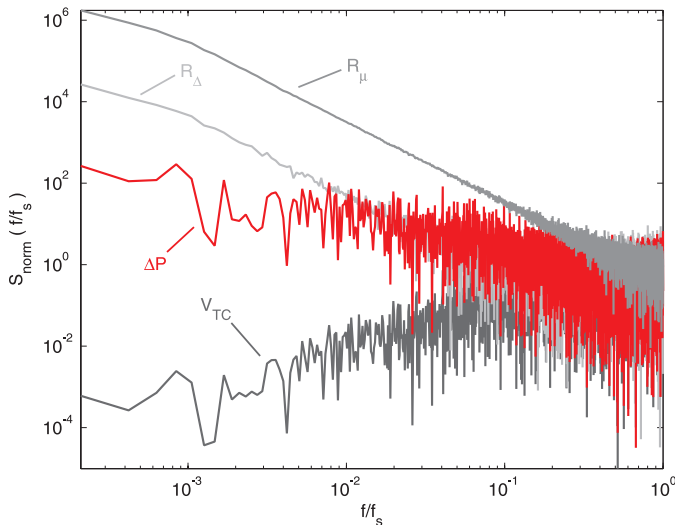


Fig. 5.11 Drift measurement for 13 hrs with 0.24 Hz sampling frequency f_s , showing normalised power spectral density S_{norm} at 1 mW total power P_T .

of the current. The Hewlett Packard 34420A nanovolt meter has low offset drift, but alternatively a chopper opamp or zero-drift opamp (AD8628) can be integrated close to the sensor chip minimising amplifier offset drift. Influence of offset voltages by thermocouple interface leads to the sensor chip can be reduced by increasing the amount of junctions in the thermopile.

The offset in power difference ΔP was analysed further by repeating the long term drift experiment in an oven at 90 °C average temperature for 147 hrs. Also the drift in the thermopile sensitivity was measured independently from the feedback loop. Figure 5.12 shows measurement results, where the average heater resistance value ΔR_{μ} is slowly changing by a 2 °C decrease in oven temperature. The difference between the resistance values ΔR_{Δ} shows the influence of induced degradation at an elevated temperature. The power difference ΔP for low frequencies does not correlate to changes in heater resistance values as well as drift in thermopile sensitivity, demonstrating the correct low-drift operation of the temperature-balancing control system. The power difference ΔP for relatively high frequencies shows to have high correla-

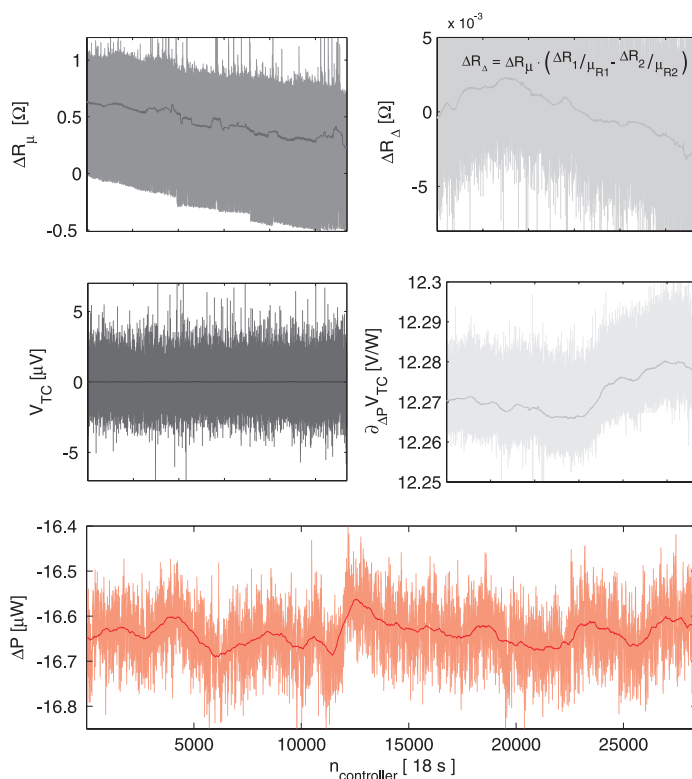


Fig. 5.12 Drift measurement for 147 hrs at 90 °C average oven temperature, while dissipating 0.1 mW total power P_T with 1000 pnt. moving average.

tion to the difference between the resistance values $\Delta R/\mu$ and, in lesser extend, to the drift in thermopile sensitivity. Indicating that, at 90 °C oven temperature, residual sensor output fluctuations are mainly caused by asymmetric temperature profile fluctuations on heater resistors and contact leads on the freely-suspended microchannel.

Measurements show that the flow sensor remains sensitive to temperature gradients across the sensor chip. This temperature gradient is modulated by oven temperature, due to spatial differences in heat conduction to ambient, caused by the connection leads going outside the oven. Temperature cycles in oven temperature therefore influence the power difference ΔP . The fluctuations in power difference ΔP in Fig. 5.12 are around 0.3% of the total power P_T dissipated and are mostly caused by the modulation of temperature gradients across the chip.

Measurements with increased total power P_T , inducing electromigration, show stable operation of the temperature-balancing control system. A change in power difference ΔP does however occur in the extreme case that voids are formed by electromigration. This can be explained by a change in the temperature profile on the heater resistor, due to a local increase in current density.

5.5 Conclusions

A thermally-isolated microchannel flow sensor has been fabricated, with integrated Al/poly-Si⁺⁺ thermopile and up- and downstream Al heater resistors. Flow measurements and FVM model calculations have shown that a temperature-balancing control system, which cancels the flow-induced temperature difference across the thermopile by controlling a power difference between both heater resistors, can give a linear measure of water flow rates up to about 0.5 $\mu\text{l}\cdot\text{min}^{-1}$. The temperature-balancing control system requires about 30 s settling time to adjust to a change in flow rate. The settling time can be avoided using a state-free controller. The discrete-sampling time of the controller being 1.1 s is limited by the integration time required to obtain the thermocouple voltage with sufficient precision. The differential-temperature dependent output voltage generated by the thermopile and power control on heater resistors has been exploited to minimise influences of drift in material properties. The sensor output drift which remains, is caused by temperature gradients across the sensor chip and by measurement errors. Temperature gradients have to be compensated for by sensor design. Measurement errors can be caused by interface thermocouple voltages, thermopile material irregularities or measurement offsets. A chopper or zero-drift opamp and a large number of thermopile junctions can improve the reduction in offset drift of the thermopile voltage due to interface lead thermo-

thermocouple voltages. An offset in power difference ΔP can still remain, due to thermopile material irregularities and flow-sensor heat-conduction asymmetries to the substrate.

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6

Low-Drift U-Shaped Thermopile Flow Sensor

A thermal flow sensor has been realised consisting of a freely-suspended U-shaped microchannel. The structure is symmetrically heated by a heater at the top of the U-shape. The thermal imbalance caused by liquid flow is sensed by an integrated Al/poly-Si⁺⁺ thermopile. The U-shape microchannel facilitates the integration of a large number of thermocouple leads, resulting in a highly-sensitive calorimetric flow sensor (40 mV/ $\mu\text{l}\cdot\text{min}^{-1}$ at the largest heating power being 2 mW). The heating power is controlled accurately by forcing a current, while measuring the voltage over the heater resistor. Influences of thermal gradients across the chip are minimised by the freely-suspended microchannel ends being fixed to the substrate over a small distance. The zero-offset of the thermopile at a uniform temperature can furthermore be exploited in a control system minimising temperature imbalance by liquid flow using additional heaters. This makes the flow sensor insensitive to heater resistor values and thermopile output characteristics. Accurate measurements up to about 300 nl·min⁻¹ water flow have been obtained applying a temperature-balancing control system.^{6.1}

6.1 Introduction

Flow sensors for accurate and reliable measurement of tiny fluid flow rates down to a few nl·min⁻¹ have been developed [6.2, 6.3]. So far, straight microchannels have been used (Chap. 5), applying thermopile read-out and power feedback [6.4], cancelling drift in the thermopile output voltage and heater resistors. With advanced control principles [6.4-6.7] significant improvement in drift performance can be obtained, but the sensor structure still suffers from ambient thermal gradients over the chip. An external temperature gradient will generate a thermopile output voltage and cannot be distinguished from an actual fluid flow through the device. In this chapter an attempt is made to realise a sensor structure that is also insensitive to external temperature gradients.

Figure 6.1 shows an existing commercially available mass flow sensor that is insensitive to thermal gradients due to a U-shaped tube structure. The sensor incorporates a single heater at the end of the U-shape creating a well defined thermal gradient to the base. A liquid flow results in a temperature imbalance over a thermopile suspended between the two legs of the U-shape structure. The sensor

^{6.1} This chapter is based on an article presented at IEEE Sensors conference [6.1].

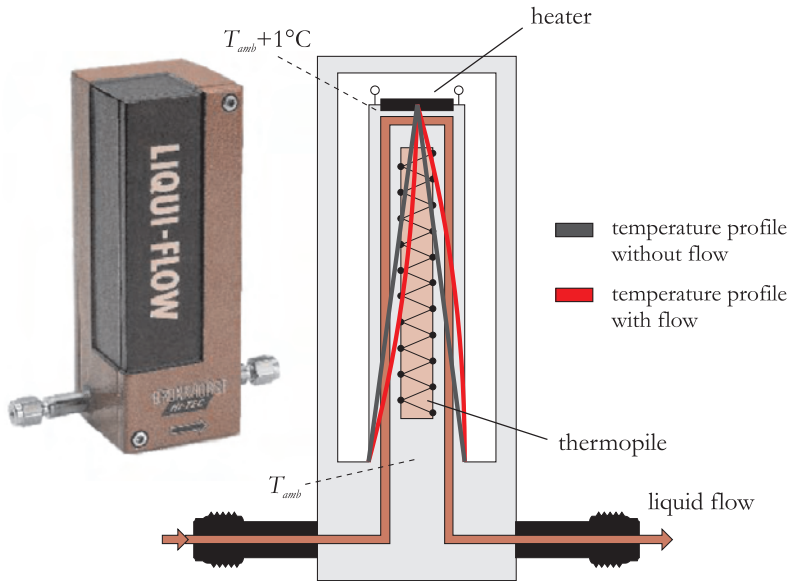


Fig. 6.1 The Liqui-Flow mass flow sensor from Bronkhorst Nederland B.V. [6.8].

proves to have an extremely stable zero with high sensitivity due to large number of thermocouple junctions that can be realised on the tube structure. In this chapter a similar sensor design is integrated on a silicon chip. The sensor design facilitates the integration of a large number of thermocouple junctions, while minimising the dependency on thermal gradients over the chip, because both ends of the U-shaped microchannel are fixed to the substrate over a small distance.

Figure 6.2 shows a representation of the freely-suspended U-shaped thermal flow sensor chip. Liquid flows through a microchannel embedded in the substrate. The freely-suspended U-shape microchannel is heated at the top by the gradient heater R_{H1} , creating a thermal gradient towards the base of the freely-suspended structure. Without flow both sides of the U-shape are heated symmetrically. When flow is applied, one side of the U-shape decreases in temperature, while the other side increases in temperature, creating temperature imbalance similar to the commercial flow sensor (Fig. 6.1). A thermopile with junctions on either side of the U-shape measures the temperature difference and gives an output voltage V_{TC} corresponding to the temperature imbalance. In this way, the sensor can be used as a calorimetric thermal flow sensor, with the voltage output of the thermopile giving a measure of the flow rate, while the power in the gradient heater can be controlled accurately using four-point measurement.

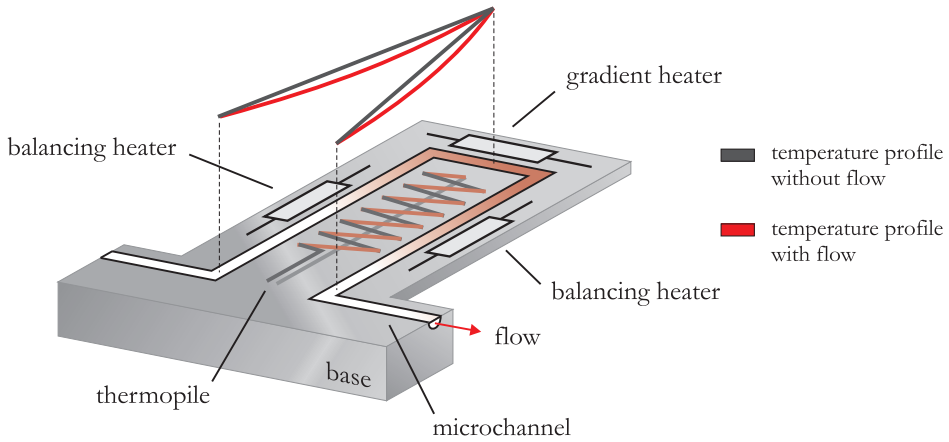


Fig. 6.2 Schematic U-shaped microchannel flow sensor structure, with integrated Al heaters and Al/poly-Si⁺⁺ thermopile.

Additionally a control system can be used (Fig. 6.3), in order to obtain a flow sensor that is insensitive to resistance drift and non-linear thermopile characteristics $m_{TC}(\Delta T_{TC})$. This requires the cancellation of the thermopile output voltage and thus the temperature imbalance between both sides of the U-shape, requiring two additional balancing heater resistors R_1, R_2 . The power dissipated in both balancing heaters is also accurately controlled using four-point measurement and power control. The output voltage V_{TC} of the thermopile is cancelled by controlling a power differ-

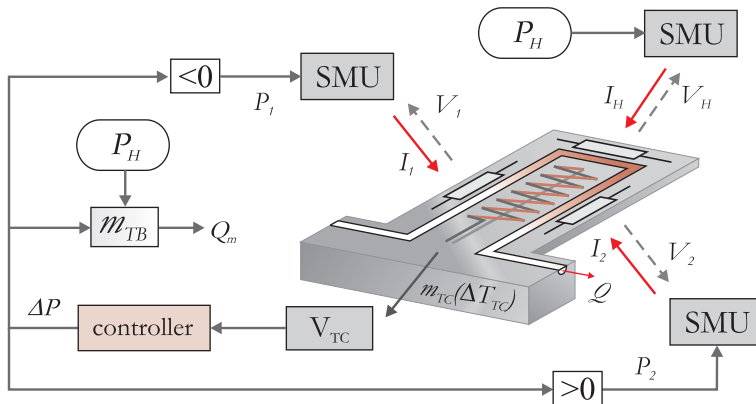


Fig. 6.3 Thermopile based power-feedback control system.

ence ΔP between the two heater resistors, while dissipating a constant power P_H in the gradient heater, according to the temperature-balancing anemometry principle [6.5]. The power difference ΔP is dissipated in one of the balancing heater resistors, with R_1 dissipating power when ΔP is negative and R_2 dissipating power when ΔP is positive, depending on the polarity of the thermopile.

As PI controller can be used to minimise the error V_{TC} , however as discussed in Chap. 5 a linear fit can be made, so that the correct ΔP is directly obtained using only two measurement points near ΔP . This means the controller does not have parameters to adjust, cannot become instable and does not suffer from settling time. The temperature distribution changes on applying a fluid flow due to forced convection. This results in the relation $\Delta P/P_H$ being dependent on the flow rate Q . A measure of the flow rate $m_{TB}(\Delta P/P_H)$ can be obtained, with $\Delta P/P_H$ being linear for small flow rates.

6.2 Sensor Fabrication

The fabricated thermopile flow sensor incorporates mechanically-strong freely suspended surface microchannels for on-chip transport of fluid. Surface microchannels are fabricated without the requirement for a sacrificial layer and allow for the integration of sensor elements in close proximity to the fluid [6.9] as explained in Chap. 3.

Figure 6.4 gives a schematic overview of the process scheme for the fabrication of the U-shaped thermopile flow sensor. Surface microchannels are created by isotropic

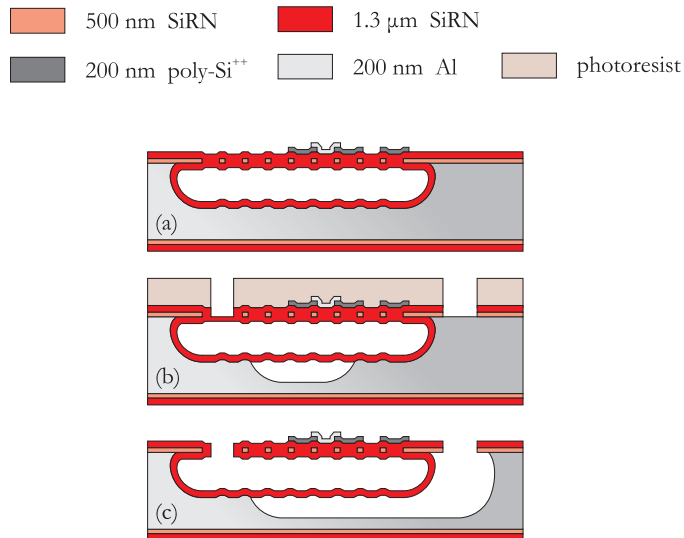


Fig. 6.4 Process scheme for the fabrication of the microchannel thermopile U-shaped microchannel flow sensor.

dry etching, using high-density SF_6 plasma with zero self-bias, through etch holes $2\ \mu\text{m}$ in width, in a low-stress $500\ \text{nm}$ silicon-rich silicon-nitride (SiRN) layer. The etch holes and inner surfaces of the microchannels are conformally coated by a second low-stress LPCVD deposited $1.3\ \mu\text{m}$ SiRN layer, resulting in completely sealed microchannels, while leaving a planar substrate surface for the integration of an Al/poly-Si⁺⁺ thermopile and Al heater resistors.

The thermopile is created by LPCVD deposition and boron doping by solid source diffusion of a $200\ \text{nm}$ poly-Si⁺⁺ layer and sputtering of a $200\ \text{nm}$ Al layer (Fig. 6.4a). The surface microchannels are released by SF_6 plasma etching, for thermal isolation from the heat-conducting substrate, with the photoresist mask protecting the sensor elements during the release (Fig. 6.4b). The photoresist is removed after fluidic entrance holes are etched through the SiRN layer, allowing for direct interfacing to the microchannels on the substrate surface.

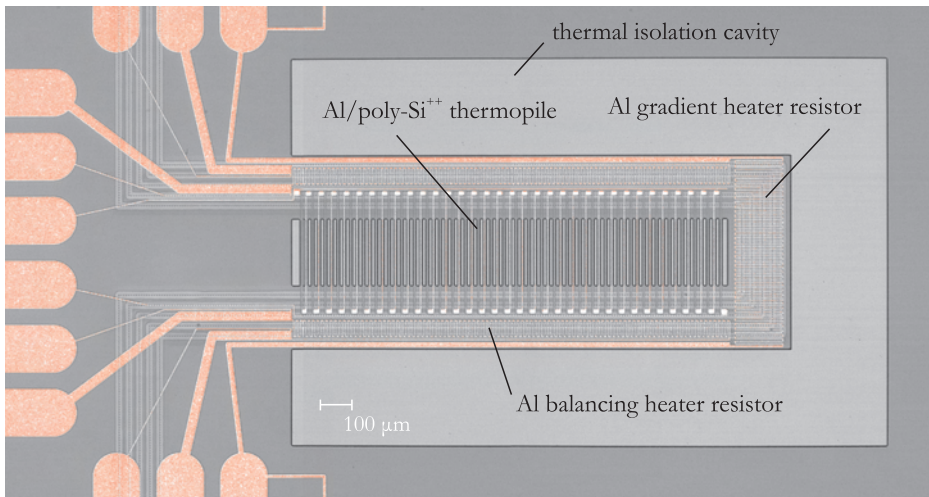


Fig. 6.5 U-shaped microchannel flow sensor just before release, showing the integrated Al heater resistors and Al/poly-Si⁺⁺ thermocouples.

Figure 6.5 shows micrographs of the fabricated thermopile flow sensor. The sensor contains five parallel $15\ \mu\text{m}$ diameter microchannels, forming a $1.5\ \text{mm}$ long U-shape freely suspended over a thermal isolation cavity etched in the silicon substrate. The thermal isolation cavity is etched to a depth of $230\ \mu\text{m}$, while the etch depth inside the U-shape is only $130\ \mu\text{m}$, due to the RIE lag around the thermocouple structures. The parallel microchannels have $200\ \mu\text{m}$ total width, which allows for the integration of the Al heater resistors and Al/poly-Si⁺⁺ thermocouple leads. The thermopile has 34 junctions on either side of the U-shape structure. The Al heater resistors have four-point measurement contacts for accurate resistance measurements.

6.3 Experimental Results

Fabricated 12.5×12.5 mm thermopile flow sensor chips were characterised using a chip holder for electrical and fluidic connections (Fig. 6.6). The chip-holder contains O-rings for fluidic interfacing and twelve pogo-pins for electrical connection to the sensor chip. Water flow was applied by an elevation head up to a few meters (see Fig. 5.8), giving a stable flow rate. The flow is restricted by the hydraulic resistance of the microchannels and was determined by microbalance weighing.

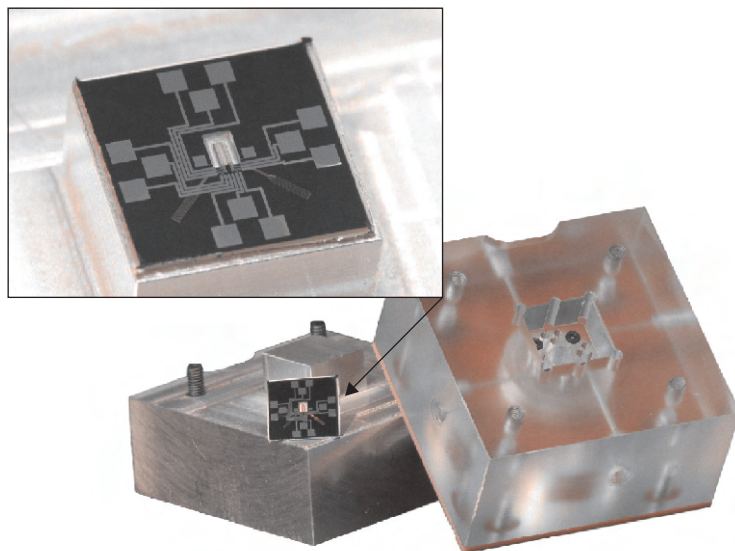


Fig. 6.6 Chip holder for fluidic and electrical interfacing with the sensor chip, measuring 12.5×12.5 mm. The inset shows an overview of the U-shaped microchannel flow sensor chip.

A source meter (Keithley 2420) was used to control the power in the gradient heater resistor. A two-channel source meter (Keithley 2602) was used to control the power on the balancing heater resistors. The thermopile output voltage was measured by a nano-volt meter (HP 34420A). The measurements were performed using MATLAB.

The flow sensor was first characterised as an open-loop calorimetric flow sensor applying different gradient heating powers P_H . The flow sensor shows a linear output response up to about $500 \text{ nl}\cdot\text{min}^{-1}$ water flow rate (Fig. 6.7), with maximum sensitivity ($40 \text{ mV}/\mu\text{l}\cdot\text{min}^{-1}$) at the largest (2 mW) heating power. At zero flow rate the thermal gradient is uniform along the U-shape structure, which gives no thermopile voltage and hence no output signal. As expected the sensor shows a well defined stable zero-level.

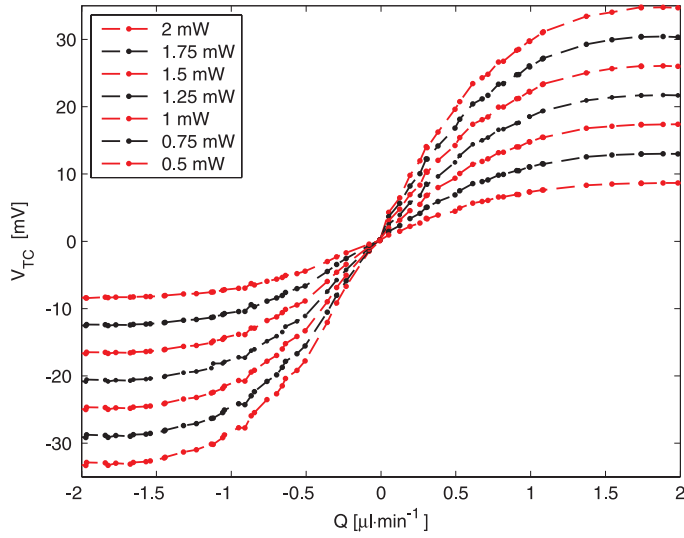


Fig. 6.7 Calorimetric sensor output response V_{TC} versus water flow rate Q with different powers P_H applied.

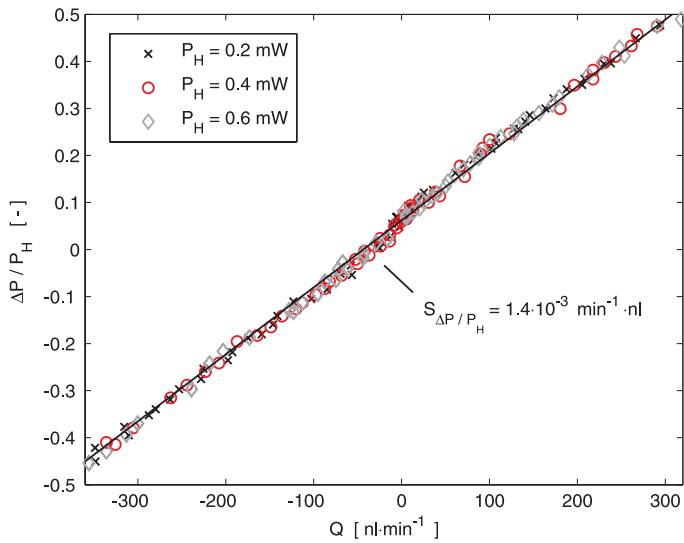


Fig. 6.8 Measured sensor output response $\Delta P/P_H$ as function of water flow rate Q at various gradient heating powers P_H .

The sensor response to externally applied temperature gradients was characterised, using the setup described in Chap. 7. It appears that a thermal gradient in the direction of the thermopile leads still generates a thermopile voltage. This can be the result of the large thermal conductivity of the air surrounding the U-shaped microchannel. A temperature gradient perpendicular to the thermopile leads does not generate an output voltage.

The flow sensor was additionally characterised applying temperature balancing, by dissipating power in the balancing heater resistors, thus cancelling the temperature imbalance due to water flow. Figure 6.8 shows the measured $\Delta P/P_H$ sensor output having a linear response independent of gradient heating power P_H with $1.4 \cdot 10^{-3} \text{ min} \cdot \text{nl}^{-1}$ sensitivity.

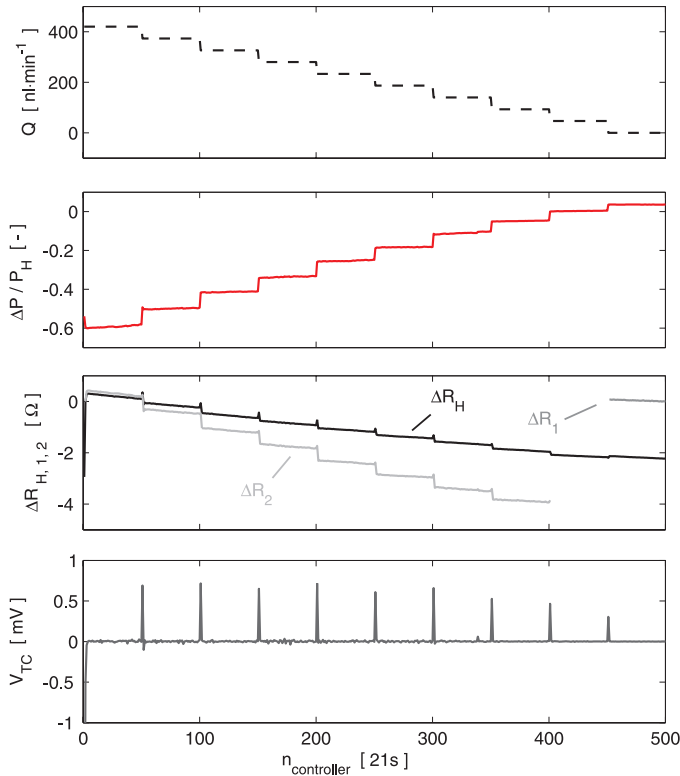


Fig. 6.9 Measured sensor control system response at changing water flow rate. The gradient heating power P_H was kept constant at 0.4 mW .

Figure 6.9 shows the measured control system response on applying a changing water flow rate, with 0.4 mW gradient heating power. The sensor output $\Delta P/P_H$ gives a correct measure of the flow rate, while being independent of heater resistance values. Figure 6.9 shows that the balancing power ΔP is being dissipated in R_1 instead of R_2 after ΔP changes sign, while the thermopile voltage is correctly regulated to zero.

Instead of the PI controller used in Fig. 5.10 a linear fit was made using two measured thermopile voltages at different ΔP . As a result the settling time of the sensor output in Fig. 6.9 is reduced to a few samples.

6.4 Conclusions

A thermal flow sensor consisting of a freely suspended U-shape microchannel with integrated Al heaters and an Al/poly-Si⁺⁺ thermopile has successfully been fabricated. A linear calorimetric sensor output was obtained for water flow rates up to about 500 nl·min⁻¹, with 40 mV/μl·min⁻¹ sensitivity at the largest (2 mW) heating power. The heating power dissipated has been made independent of the heater resistance value, by using power control applying four-point measurement on the gradient heater resistor. Influence of thermal gradients across the chip has been minimised by the U-shape microchannel design, although a gradient parallel to the thermopile leads can still generates a small output voltage. The dependency on the non-linear characteristics of the thermopile has been minimised by using a control system, cancelling the temperature imbalance across the thermopile. A linear sensor response for water flow rates up to about 300 nl·min⁻¹, with 1.4·10⁻³ min·nl⁻¹ sensitivity, was measured using temperature balancing. The temperature-balancing control system operates correctly, without the possibility of instability and without settling time.

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7

Ambient Temperature-Gradient Compensated Low-Drift Thermopile Flow Sensor

A highly-sensitive thermal flow sensor for liquid flow with $\text{nl}\cdot\text{min}^{-1}$ resolution has been realised. The sensor consists of freely suspended silicon-rich silicon-nitride microchannels with integrated Al heater resistors and Al/poly-Si⁺⁺ thermopiles. The influence of drift in the thin film metal resistors is effectively eliminated by using thermopiles combined with an adequate measurement method, where the power in the heater resistors is controlled, e.g. constant-power calorimetric method or temperature balancing method. The special meandering layout of the microchannels and the placement of thermopile junctions increases sensitivity by summing the thermopile voltages due to convection by fluid flow, whereas the influence of ambient temperature gradients is compensated for.^{7.1}

7.1 Introduction

Fabricated micromechanical thermal flow sensors [7.2-7.5] have shown the capability of measuring down liquid flows down to $\text{nl}\cdot\text{min}^{-1}$ resolution. Important problems limiting the accuracy of these thermal flow sensors are: the drift in the electrical resistance of thin-film layers used for heating and temperature sensing, and the influence of external temperature gradients across the sensor chip [7.6]. Resistors used for temperature sensing can be avoided, by using thermopiles to measure a temperature difference, with zero-offset at a uniform temperature across the thermopile [7.7, 7.8]. Using thermopiles, the flow sensor can be made independent of resistance drift by applying power control on heater resistors. Additionally, a temperature-balancing feedback loop can be used to compensate for thermopile sensitivity drift [7.8].

This chapter describes a low-drift calorimetric flow sensor for liquid flows with $\text{nl}\cdot\text{min}^{-1}$ resolution. The flow sensor consists of freely suspended microchannels with integrated thermopiles for temperature sensing, with power control applied on heater resistors used. Furthermore, a special meandering microchannel layout is utilised to compensate for the influence of external temperature gradients across the sensor chip. The meandering layout eliminates the remaining sensitivity to temperature gradients in the U-shaped microchannel sensor [7.6] described in Chap. 6.

^{7.1} This chapter is based on an article presented at the IEEE MEMS 2009 conference [7.1].

7.2 Compensation Concept

Ambient temperature-gradients are compensated using a meandering microchannel layout, in which a fluid flow Q passes in alternating direction through several freely suspended microchannels used for thermal flow sensing (Fig. 7.1). Thermopiles are connected in series and positioned on the freely-suspended microchannels in such a way that each thermopile measures a temperature difference ΔT between the upstream temperature T_{up} and a downstream temperature T_{down} . Effectively the flow passes the same sensor configuration several times, while flowing through the meandering microchannel, giving a summation of multiple thermopile voltages due to convection by fluid flow.

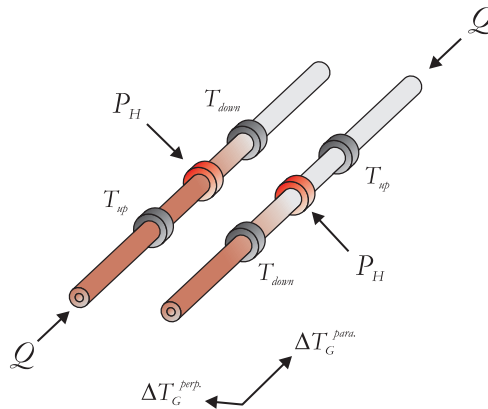


Fig. 7.1 Ambient temperature-gradient compensated flow sensor concept.

External temperature gradients ΔT_G of first order can be decomposed in two directions. A temperature gradient parallel to the microchannels $\Delta T_G^{para.}$ changes T_{up} and T_{down} in opposite ways on alternating microchannels, not influencing the total measured ΔT . A gradient perpendicular to the microchannels $\Delta T_G^{perp.}$ changes the offset temperature of each microchannel, not influencing the difference between T_{up} and T_{down} on the same microchannel. First order temperature gradients can therefore be fully compensated. However, the heater resistors are slightly temperature dependent and therefore require control of the heating power P_H , in order for the compensation concept to work properly.

7.3 Sensor Fabrication

The fabricated thermopile flow sensor uses freely suspended surface microchannels for on-chip transport of fluid. The microchannels are fabricated without the requirement for sacrificial layer etching and allow for the integration of sensor ele-

ments in close proximity to the fluid [7.9]. The technology has been applied in the fabrication of highly-sensitive flow sensors [7.5, 7.8, 7.10] and is explained in Chap 3.

Figure 7.2 gives a schematic overview of the process scheme for the fabrication of the thermopile flow sensor. Surface microchannels are created by isotropic dry etching, using high-density SF₆ plasma with zero self-bias, through etch holes 2 μm in width, in a low-stress 500 nm silicon-rich silicon-nitride (SiRN) layer. The etch holes and inner surfaces of the microchannels are conformally coated by a second low-stress LPCVD deposited 1.3 μm SiRN layer, resulting in completely sealed microchannels, while leaving a planar substrate surface for the integration of Al/poly-Si⁺⁺ thermopiles and Al heater resistors. The thermopiles are created by LPCVD deposition and boron doping, by solid source diffusion, of a 200 nm poly-Si⁺⁺ layer and sputtering of a 200 nm Al layer (Fig. 7.2a). The surface microchannels are released by SF₆ plasma etching for thermal isolation from the heat-conducting substrate, with the photoresist mask protecting the sensor elements during the release (Fig. 7.2b). The photoresist is removed after fluidic entrance holes are etched through the SiRN layer (Fig. 7.2c).

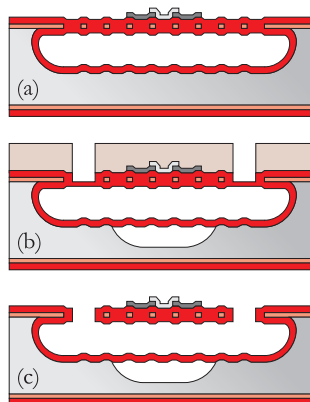
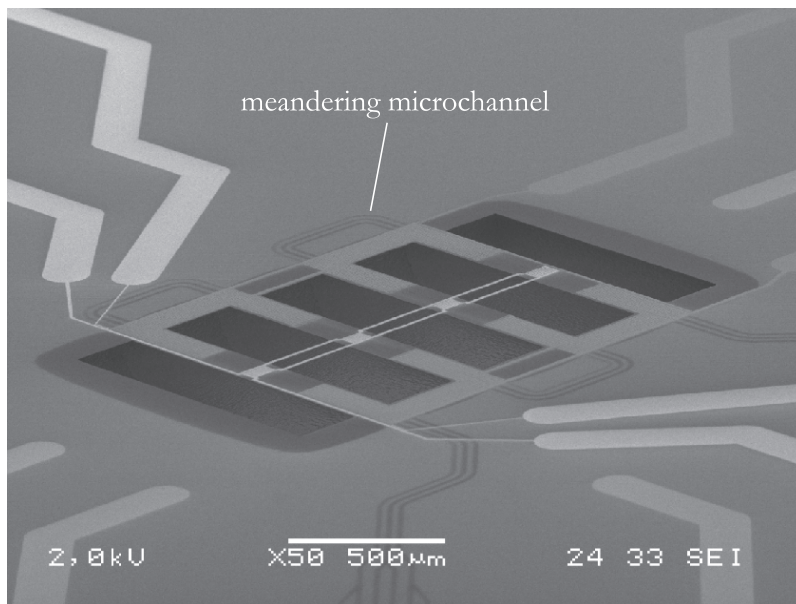
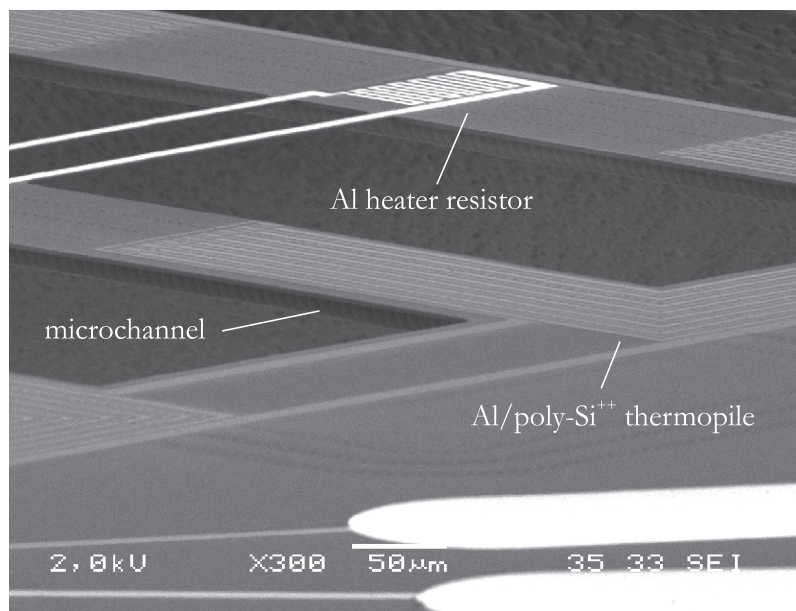


Fig. 7.2 Process scheme for the fabrication of the temperature-gradient compensated thermopile flow sensor.

Figure 7.3a shows a micrograph of the fabricated temperature-gradient compensated flow sensor. The sensor contains three parallel 20 μm diameter microchannels, which cross a thermal-isolation cavity (1.2 × 2.0 mm, 250 μm deep) several times in alternating flow directions. Heater resistors are centred on the freely-suspended microchannels and connected in series. The total resistance of the heater resistors is



a)



b)

Fig. 7.3 a) Microchannel temperature-gradient compensated thermopile flow sensor. b) Close-up of the microchannels with integrated Al heater resistors and Al/poly-Si⁺⁺ thermopiles, suspended over a thermal isolation cavity etched in the substrate.

determined accurately using four-point measurement. Of course for maximum accuracy all resistors should be measured and power controlled individually.

Figure 7.3a shows a close-up of the surface microchannels with integrated Al/poly-Si⁺⁺ thermopiles having eight junctions on either side, for measuring the down- and upstream temperature difference.

7.4 Sensor Modelling

A thermal model of the flow sensor was constructed in COMSOL Multiphysics [7.11]. The temperature field was solved including the thermal conduction in the flow sensor and surrounding air and forced convection by water flow through the microchannels. The flow inside the microchannels is approximated using a fixed velocity profile corresponding to a given flow rate, without affecting the accuracy of

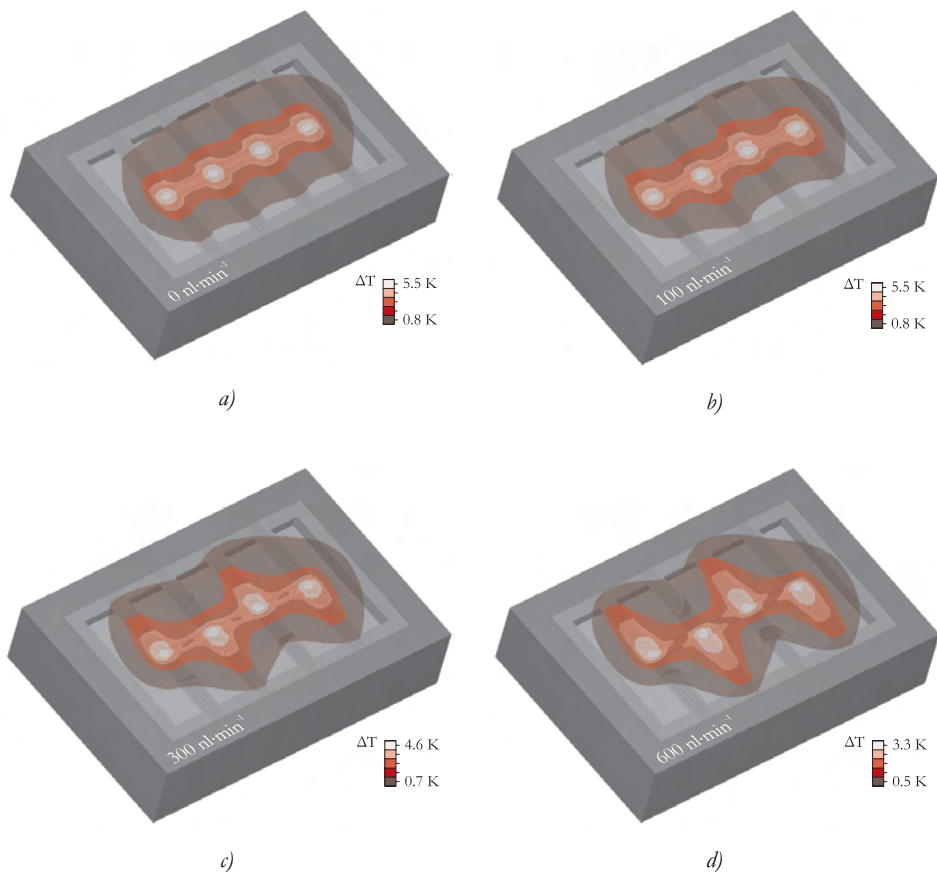


Fig. 7.4 Thermal FEM model of the flow sensor showing the influence of convection by the alternating direction of various water flow rates through the microchannels at 0.6 mW total heating power.

the thermal model. Additionally, deposited layers are modelled as highly conductive layers, including the thermopiles and heater resistors.

Figure 7.4 shows isothermal surfaces of the resulting temperature field at various water flow rates through the microchannels, with 0.6 mW total heating power applied. Clearly, the effect of forced convection by the alternating direction of the flow through the microchannels can be observed at this relatively high flow rate. The high thermal conduction of the Al leads connecting the heater resistors is also apparent, although there is no heat flow between adjacent microchannels for the temperature field by conduction alone.

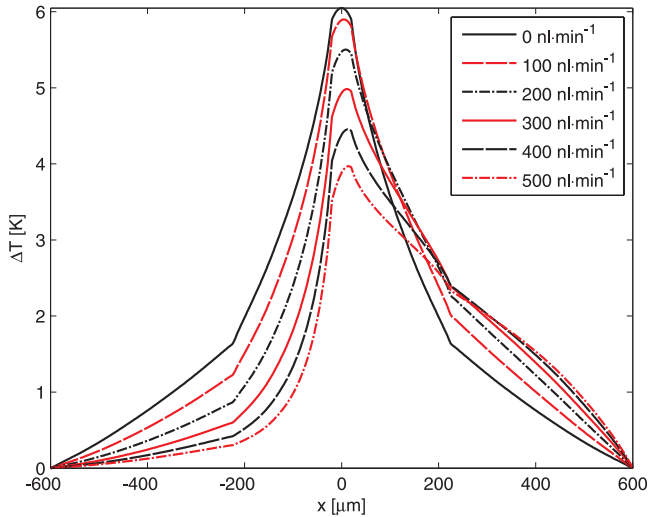


Fig. 7.5 Temperature profiles for various water flow rates in the positive x -direction along one of the microchannel segments suspended at $\pm 600 \mu\text{m}$, with 0.6 mW total heating power applied.

Figure 7.5 shows temperature profiles along the microchannel at various water flow rates. A difference in the slope of the temperature profiles can be observed at $\pm 225 \mu\text{m}$ from the centre, where the thermopile junctions are situated on the microchannel, with the thermopile leads causing much higher thermal conduction to the substrate.

7.5 Experimental Results

7.5.1 Flow-Rate Sensitivity

The fabricated thermopile flow sensors chips ($12.5 \times 12.5 \text{ mm}$) have bondpads and microchannel entrance holes on fixed positions for self alignment with a chip holder (Fig. 7.6). The chip holder contains O-rings for fluidic interfacing and pogo-pins for

electrical connections to the sensor chip. Water flow is applied by an elevation head Δh , giving a stable flow rate. The flow rate is calibrated by microbalance weighing, where evaporation from the balance, in the order of $100 \text{ nl}\cdot\text{min}^{-1}$, has to be taken into consideration. A source measurement unit (Keithley 2420) was used for controlling a fixed total heating power P_H . The thermopile voltage V_{TC} was obtained by a nano-volt meter (HP 34420A).

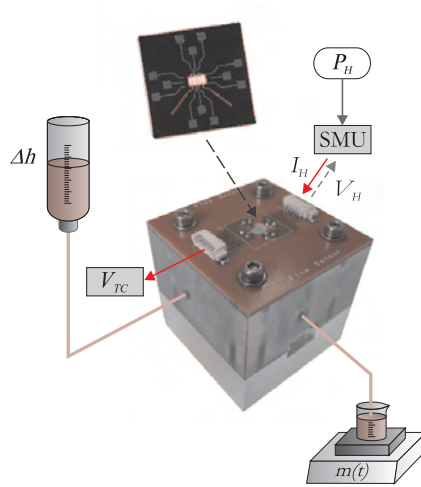


Fig. 7.6 Overview of the flow measurement setup, with the chip holder establishing fluidic and electrical connections.

Figure 7.7 shows the thermopile voltage V_{TC} dependency on water flow rate Q up to $550 \text{ nl}\cdot\text{min}^{-1}$ at various applied heating powers. The sensor shows linear sensitivity up to flow rates of about $100 \text{ nl}\cdot\text{min}^{-1}$. The heater temperature decreases at higher flow rates (Fig. 7.5) according to King's law, causing the sensor output to decrease. The dashed lines in Fig. 7.7 show fitted FEM results after scaling calculated temperature differences to measured voltages. Measurements show offsets in the thermopile voltage, which are dependent on the heating power and can be attributed to sensor and measurement asymmetries.

7.5.2 Ambient Temperature-Gradient Sensitivity

A temperature-gradient measurement setup was constructed in order to determine the influence of external temperature gradients on the thermopile voltage (Fig. 7.8). The setup consists of a 5 mm thick Cu plate, which is heated from one side by a resistor dissipating 25 W, while the other side is connected to a large aluminium plate acting as heat sink, thus creating a well defined temperature gradient. The temperature gradient was determined to be approximately $2.8 \text{ K}\cdot\text{cm}^{-1}$, measured using two Pt 100

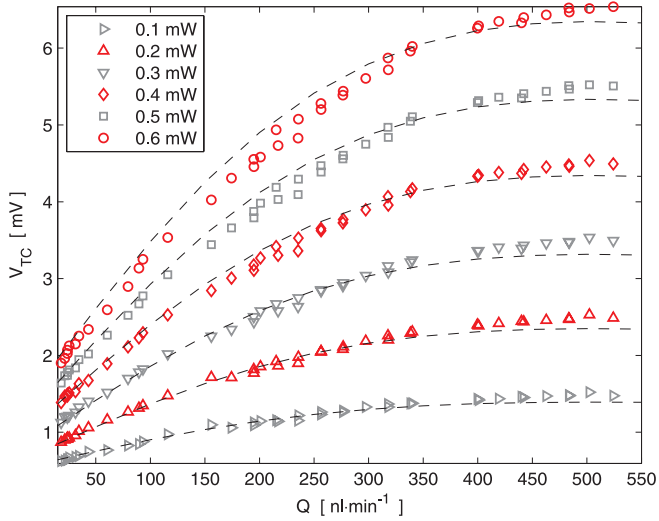


Fig. 7.7 Measured flow sensor thermopile-voltage V_{TC} dependency on water flow rate Q at various applied total heating powers. Dashed lines show calculated thermopile temperature-differences from the FEM model scaled to the measurement points.

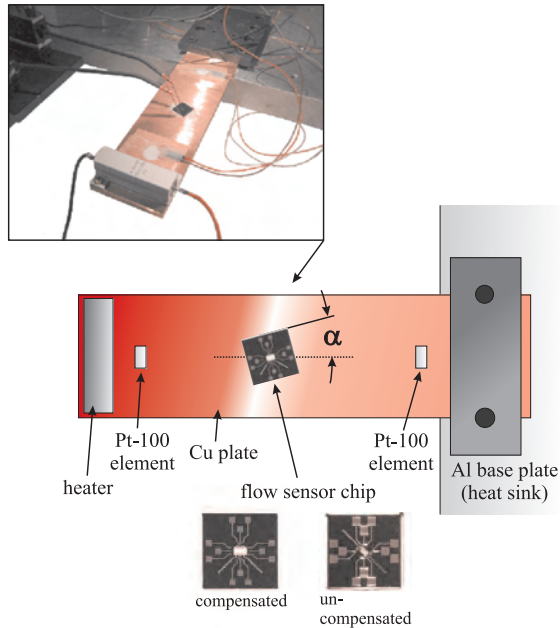


Fig. 7.8 Temperature-gradient measurement setup for inducing a well defined temperature gradient across the sensor chip.

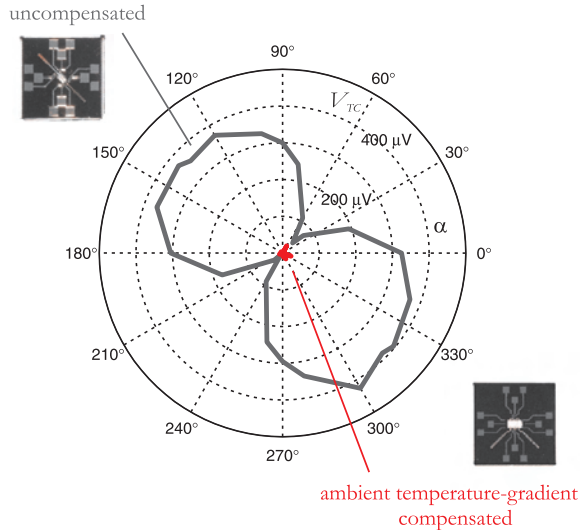


Fig. 7.9 External temperature-gradient sensitivity of the thermopile voltage V_{TC} measured for a compensated sensor and a single microchannel uncompensated flow sensor.

elements placed 10 cm apart. The sensor chip is placed in the centre of the Cu plate and probes are used to measure the thermopile voltage using a nano-volt meter (HP 34420A). This allows the sensor chip to be rotated and the influence of the external temperature gradient to be measured at any angle.

An ambient temperature-gradient compensated flow sensor and an uncompensated single-microchannel flow sensor, for comparison, were measured, with both sensors having nearly equal amounts of thermopile junctions. The ambient temperature-gradient compensated flow sensor shows to be completely compensated for external temperature gradients (Fig. 7.9). The uncompensated flow sensor shows a figure-of-eight sensitivity pattern, because the flow sensor is only sensitive to temperature gradients parallel to the microchannel ΔT_{para} .

7.6 Conclusions

A calorimetric flow sensor measuring liquid flow down to $\text{nl}\cdot\text{min}^{-1}$ resolution has been realised, using freely suspended microchannels for thermal isolation from the silicon substrate. The flow sensor shows a linear response for water flow up to about $100 \text{ nl}\cdot\text{min}^{-1}$, which is accurately described by a thermal FEM model of the sensor.

It was demonstrated in Chap. 5 that drift in thin-film metal resistors can be effectively eliminated by using power control, in conjunction with thermopiles for temperature sensing. This in combination with the special meandering microchannel layout can effectively minimise drift in material properties and drift due to ambient temperature-gradients across the sensor chip.

Compared to a single microchannel uncompensated flow sensor a significant improvement is obtained in sensor accuracy. The characteristic figure-of-eight to temperature gradients, which was observed even in the special U-shaped design (Chap. 6) has disappeared below the noise level.

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8

AC-Driven Temperature-Balancing Flow Sensor with Power Feedback

A microchannel thermal flow sensor with integrated Al resistors has successfully been fabricated. A control system has been implemented using a nearly offset-free power-feedback signal applying heat waves, giving a sensor output independent of resistance values and thermo-electric offsets on interface wires. Four-point measurement and power control is used on heater resistors, which can therefore drift without influencing sensor output. The heat waves are detected at the sensor resistor using a lock-in amplifier and are mutually cancelled using a temperature-balancing controller. Offsets due to thermal gradients across the chip and sensor resistor drift are eliminated by the lock-in amplifier and therefore do not influence the sensor output signal. The flow sensor is capable of measuring water flow rates up to about 500 nl·min⁻¹. Measurement results are in good agreement with a dynamic model of the flow sensor. Drift measurements show the compensation for resistance drift and thermal gradient in the sensor output signal. ^{8.1}

8.1 Introduction

Increased miniaturisation of flow sensors has resulted in micro thermal flow sensors capable of measuring liquid flows down to a few nl·min⁻¹ [8.2, 8.3]. These flow sensors require accurate measurement of very small flow induced temperature changes. Using a thermopile and advanced control techniques the influence of drift in resistance values and thermopile sensitivity can be largely eliminated. However, it is difficult to eliminate the influence of thermo-electric offset voltages, resistance drift and thermal-gradients completely.

In fact, non-homogeneous heating due to local variations in resistivity of the heater resistors cannot be prevented and will ultimately limit accuracy. Furthermore, irregularities in the thermopile material may result in thermopile voltages even when the average temperature difference between the junctions is zero. Therefore, a different approach [8.4] is investigated in this chapter, in which the heater resistors are heated by alternating currents, which may result in even greater accuracy using less complicated device structures. A disadvantage is a slower response time of the sensor,

^{8.1} This chapter is based on an article presented at the MME 2008 conference [8.1].

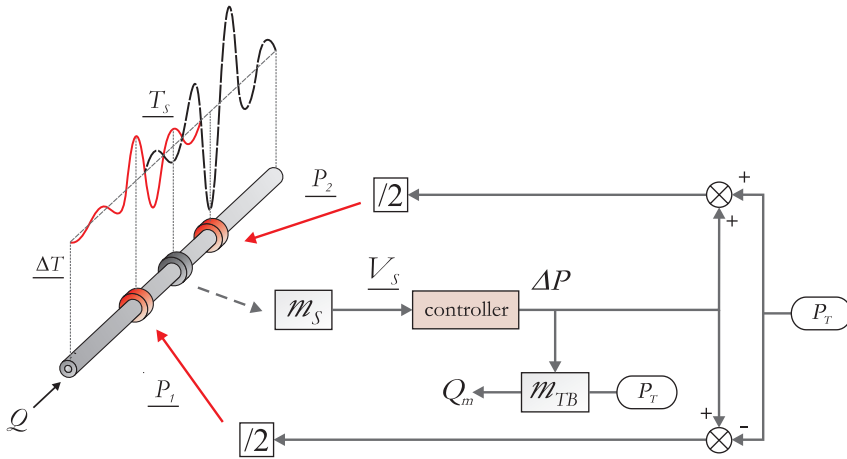


Fig. 8.1 Control system for the AC-driven temperature-balancing flow sensor.

since very low frequencies have to be used in order to obtain sufficiently large signal amplitudes.

Figure 8.1 shows a schematic drawing of a thermal flow sensor and control system that exploits sinusoidal heat waves, detected by a sensor resistor, to create a nearly offset-free power-feedback signal ΔP independent of sensor resistance value R_S . Two 180° out-of-phase sinusoidal heat waves are generated by dissipating alternating currents in two resistors, up- and downstream of a central sensing resistors. The heating powers P_1, P_2 are controlled, with constant measurement of the resistance values of the heater resistors R_1, R_2 , which can therefore drift without influencing sensor output. Thermo-electric offset voltages on interface wires are effectively eliminated by the lock-in amplifier. The lock-in amplifier also eliminates offset signals due to slowly varying thermal-gradients across the chip. The heat wave signals \underline{T}_S detected by the sensor resistor \underline{V}_S are mutually cancelled, minimising \underline{V}_S by controlling ΔP between the two heater resistors, while keeping the total heating power P_T dissipated constant [8.5-8.6]. The amplitude of \underline{V}_S is controlled to a minimum by changing the power difference ΔP . The exact amplitude of \underline{V}_S is therefore not important and the sensor resistance is allowed to drift.

On applying a fluid flow the temperature distribution is changed by forced convection. This results in the relation $\Delta P/P_T$ being dependent on the flow rate Q . A measure of the flow rate $m_{TB}(\Delta P/P_T)$ can be obtained, with $\Delta P/P_T$ being linear for small flow rates.

8.2 Sensor Fabrication

Thermal flow sensors have been fabricated using the surface channel technology [8.7], discussed in Chap. 3, for on-chip transport of fluid through mechanically strong freely suspended microchannels. The microchannels are fabricated below the wafer surface allowing for the integration of sensor elements in close proximity to the fluid.

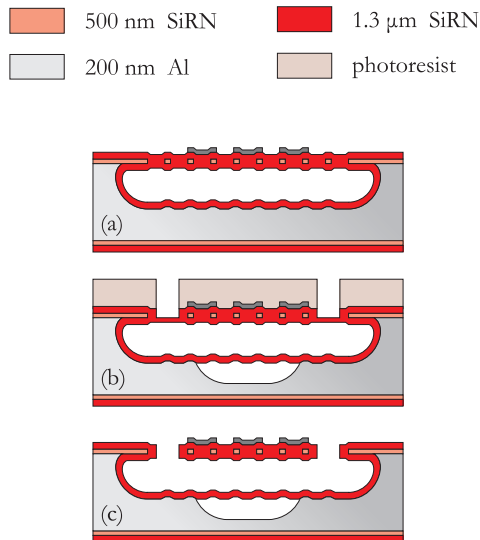
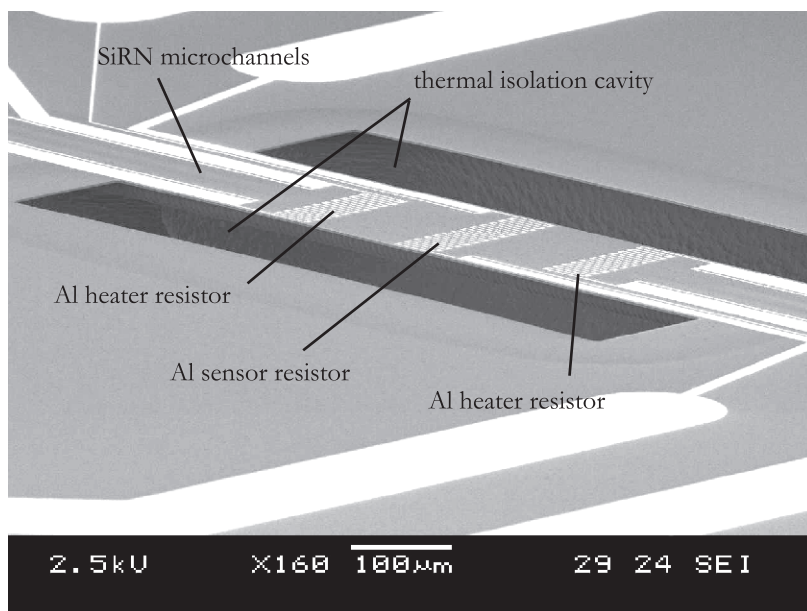
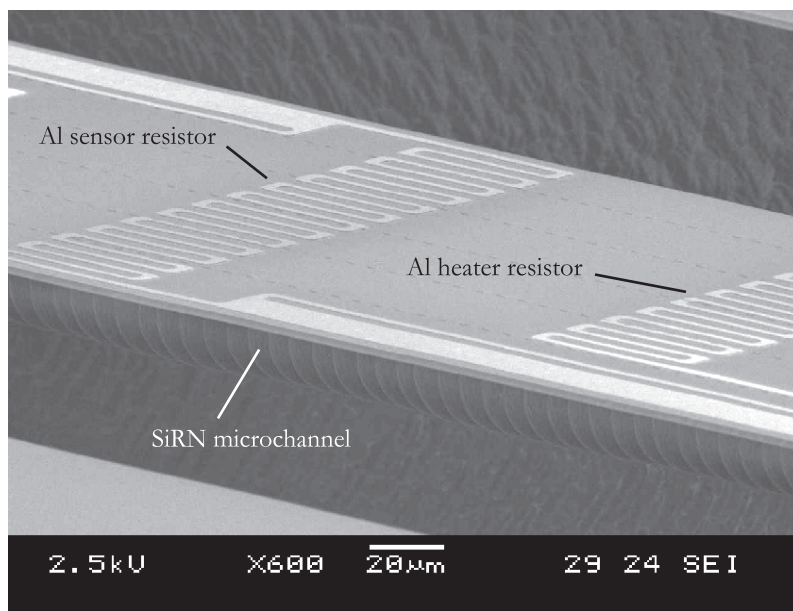


Fig. 8.2 Process scheme for the fabrication of the AC-driven temperature-balancing flow sensor.

Figure 8.2 shows the process scheme for the fabrication of the temperature-balancing flow sensor. Surface microchannels are created by isotropic dry etching, using high-density SF_6 plasma with zero self-bias, through etch holes $2\ \mu\text{m}$ in width, in a low-stress $500\ \text{nm}$ silicon-rich silicon-nitride (SiRN) layer. The etch holes and inner surfaces of the microchannels are conformally coated by a second low-stress LPCVD deposited $1.3\ \mu\text{m}$ SiRN layer, resulting in completely sealed microchannels, while leaving a planar substrate surface for the integration of Al resistors (Fig. 8.2a). The surface microchannels are released by SF_6 plasma etching, for thermal isolation from the heat-conducting substrate, with the photoresist mask protecting the sensor elements during the release (Fig. 8.2b). The photoresist is removed after fluidic entrance holes are etched through the SiRN layer, allowing for direct interfacing to the microchannels on the substrate surface. Figure 8.3 shows SEM micrographs of the fabricated microchannel thermal flow sensor. The sensor contains five parallel $20\ \mu\text{m}$ diameter microchannels, spanning a $1.2\ \text{mm}$ thermal isolation cavity. Up- an down-



a)



b)

Fig. 8.3 Micrographs of the temperature-balancing flow sensor, with a) sensor structure suspended over thermal isolation cavity and b) close-up of Al resistors.

stream Al heater resistors and an Al sensor resistor are integrated on top of the microchannels. The resistors have four-point contacts for accurate resistance measurements.

8.3 Sensor Modelling

A two-dimensional cylindrical finite volume method (FVM) model was constructed in MATLAB, solving the heat advection-diffusion equation for sinusoidal heat waves, with fluid flow along the x -direction. Using complex sparse-matrix calculations (8.1)

$$\left(j2\omega\rho c + \rho c v_x \frac{\partial}{\partial x} - \nabla(\kappa\nabla) \right) T = P''' \quad (8.1)$$

can be solved on a cylindrical mesh of the sensor, resulting in a heat capacity matrix \mathbf{B} , a convection matrix \mathbf{G}_Q and a conduction matrix \mathbf{G}_κ , which can be solved directly using the Gauss-Seidel method on (8.2).

$$\begin{aligned} (j2\omega \cdot \mathbf{B} + \mathbf{Q} \cdot \mathbf{G}_Q - \mathbf{G}_\kappa) \vec{T}_e &= \vec{P}_e \\ \mathbf{Y} \vec{T}_e &= \vec{P}_e \end{aligned} \quad (8.2)$$

The column vectors \vec{T}_e and \vec{P}_e contain respectively the resulting complex temperature and complex power input for each element in the FVM mesh. The 180° out-of-phase sinusoidal heat waves are given as two complex vector at the heater positions, with amplitude determined by temperature-balancing.

Figure 8.4 shows calculated temperature fields, represented by the amplitude and phase of the complex field. Fig. 8.4a shows the temperature field with no flow applied. Two heat waves are generated at the location of the heater resistors, having equal amplitude at zero flow, where the phase plot shows that both heat waves are actuated 180° out of phase. In Fig. 8.4b 500 nl·min⁻¹ water flow is applied, where the downstream heater dissipates more than the upstream heater maintaining a minimum in the amplitude of the temperature at the sensor resistor. A small asymmetry can be seen in the phase plot of Fig. 8.4b, due to forced convection in the microchannel.

The heat waves at 5 Hz extend only within one period between the heater resistor and sensor resistor. Figure 8.4c shows the temperature field with no flow applied at 1 kHz. The heat wave has more than one period inside the sensor structure, while the amplitude of the heat wave has decreased significantly. The heat capacity of the fluid inside the microchannel results in the fluid retaining most of the dissipated heat.

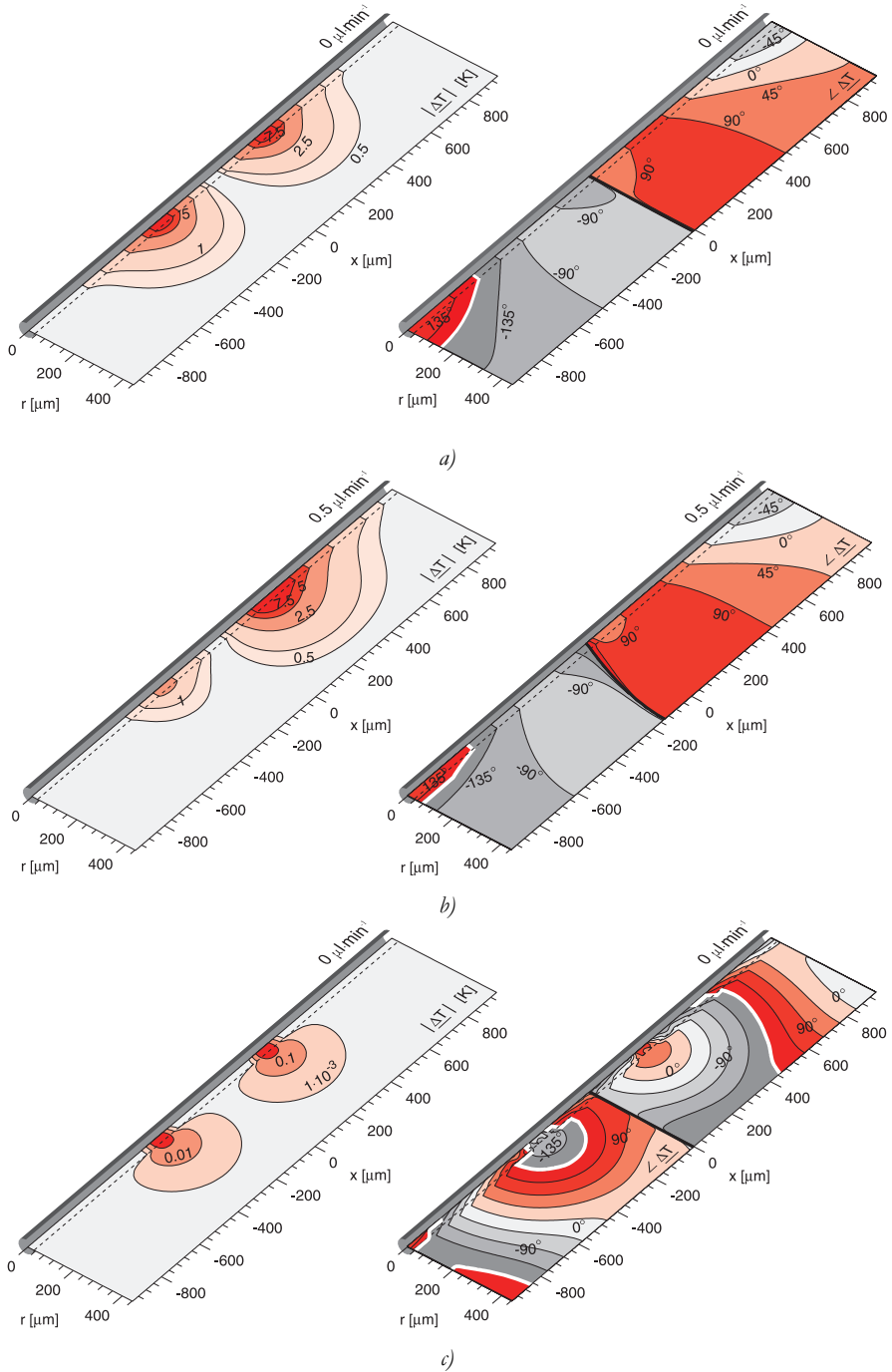


Fig. 8.4 Calculated temperature amplitude (left) and phase (right) fields with 3 mW total power P_T at a) 5 Hz without flow, b) 5 Hz with 500 nl·min⁻¹ water flow and c) 1 kHz without flow.

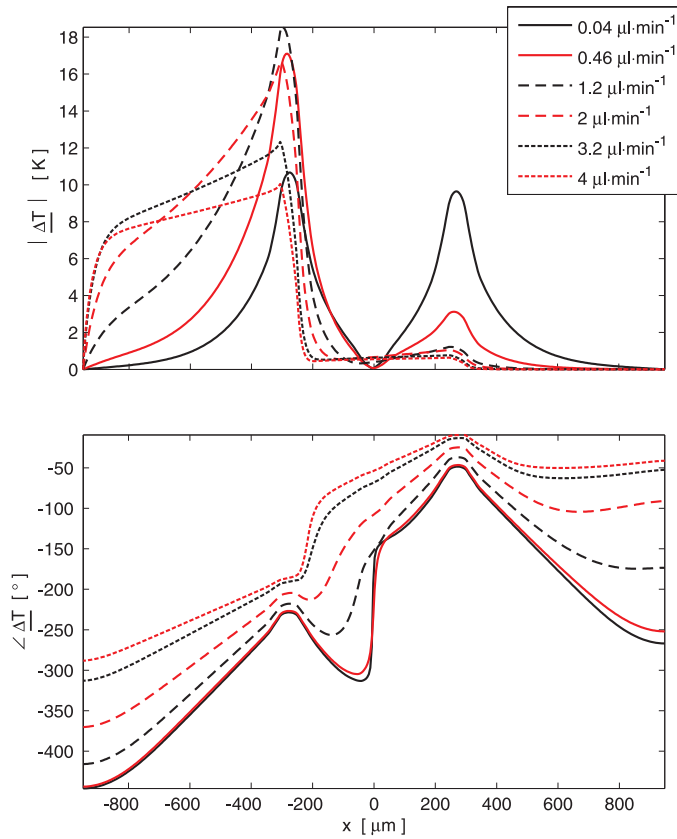


Fig. 8.5 Calculated temperature amplitude and phase profiles along the microchannel at 5 Hz with 3 mW total heating power P_T and various water flow-rates Q in the negative x -direction.

Figure 8.5 shows resulting temperature profiles along the microchannel from FVM model calculations. The temperature-balancing of the heat waves, minimising the amplitude at the sensor resistor, was simulated for different flow rates Q . It can be observed from Fig. 8.5 that the amplitude in temperature at the sensor resistor can be made zero for flow rates up to $0.46 \mu\text{l}\cdot\text{min}^{-1}$. At flow rates of $1.2 \mu\text{l}\cdot\text{min}^{-1}$ and higher a minimum can be obtained at the sensor resistor but the temperature amplitude cannot be made completely zero, due to the large forced convection from the upstream heater.

8.4 Experimental Results

Fabricated thermal flow sensor chips were characterised using a chip holder for electrical and fluidic connections (Fig. 8.6). Water flow was applied by adjusting an

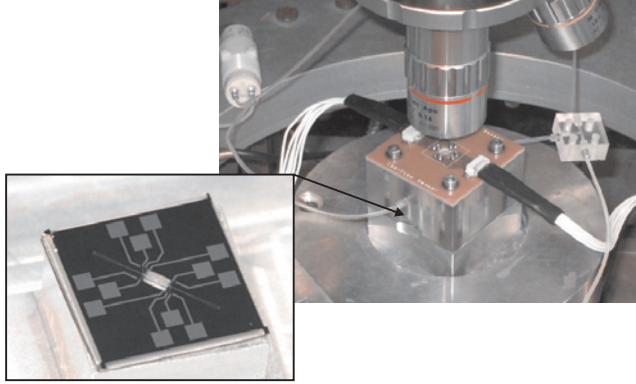


Fig. 8.6 Electrical and fluidic interconnection to the sensor chip. Inset shows a photograph of the temperature-balancing flow sensor chip.

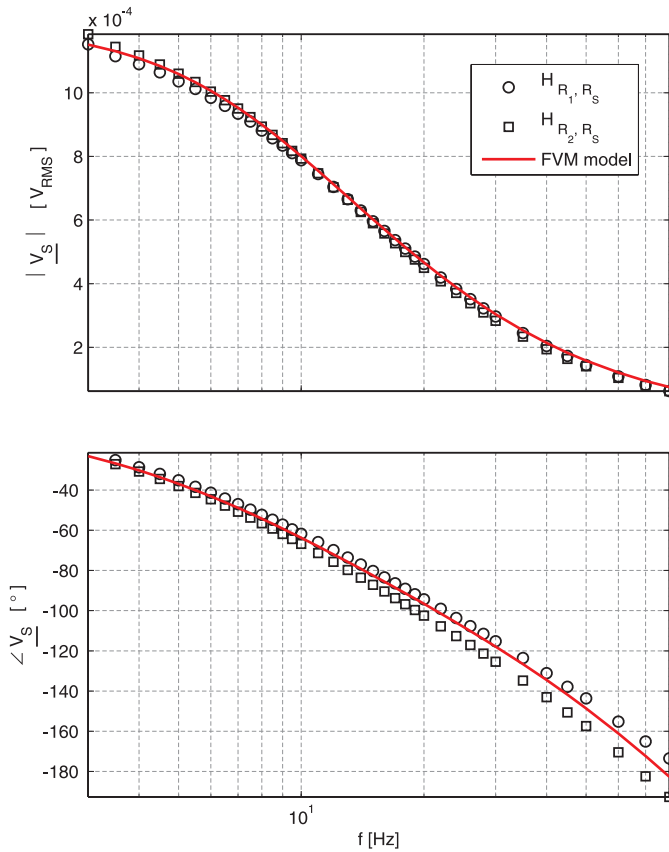


Fig. 8.7 Frequency response of each heater resistor R_1 , R_2 to the sensor resistor R_S at 0.4 mW power P_T compared to FVM model calculations.

elevation head, while the actual flow rate was calibrated by microbalance weighing. The measurements were controlled by MATLAB. The heater resistors were heated by applying an alternating current (HP 3245A). The heating power was controlled accurately by reading the voltage drop over the heater resistors using multimeters (HP 34401A) and by adjusting the AC amplitude accordingly. A small direct current was passed through the sensor resistor (Keithley 237), while the 2ω component in the voltage drop over the sensor resistor was measured by a DSP lock-in amplifier (SRS 830).

Figure 8.7 shows the open-loop frequency response, simulated, and measured for each heater resistor to the sensor resistor, without flow. The amplitude is proportional to $\sqrt{V_f}$, due to the frequency dependency on the thermal penetration depth of the sinusoidal heat wave. A small phase difference between both heaters can be observed

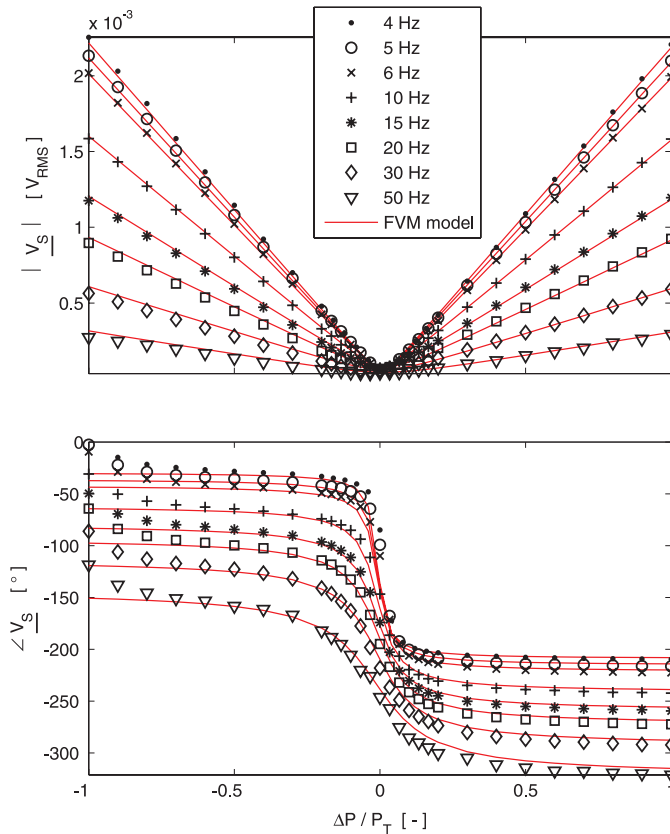


Fig. 8.8 Open-loop sensor resistor response on power difference ΔP without flow at various frequencies with 0.8 mW total heating power P_T applied compared to FVM model calculations.

in Fig. 8.7 for higher frequencies, indicating a small asymmetry in the flow sensor.

Figure 8.8 shows the sensor response on the power difference ΔP between both heaters. Minimum amplitude for \underline{V}_S , independent of frequency, is obtained when both heaters dissipate equal amount of power. The determination of the minimum in amplitude is independent of the sensor resistance value R_S and is theoretically offset and drift free. However, asymmetries in the sensor structure can still give rise to an offset in ΔP , which shows to be more apparent in close-loop operation. The decrease in slope of the phase of \underline{V}_S near zero $\Delta P/P_T$ at higher frequencies (Fig. 8.8) is caused by the sensor asymmetry measured in Fig. 8.8. The FVM model results only show this decrease in slope when the phase difference from Fig. 8.7 is taken in consideration.

The position of the minimum in the amplitude of \underline{V}_S changes with flow rate applied. Figure 8.9 shows measured and simulated open-loop response on ΔP to var-

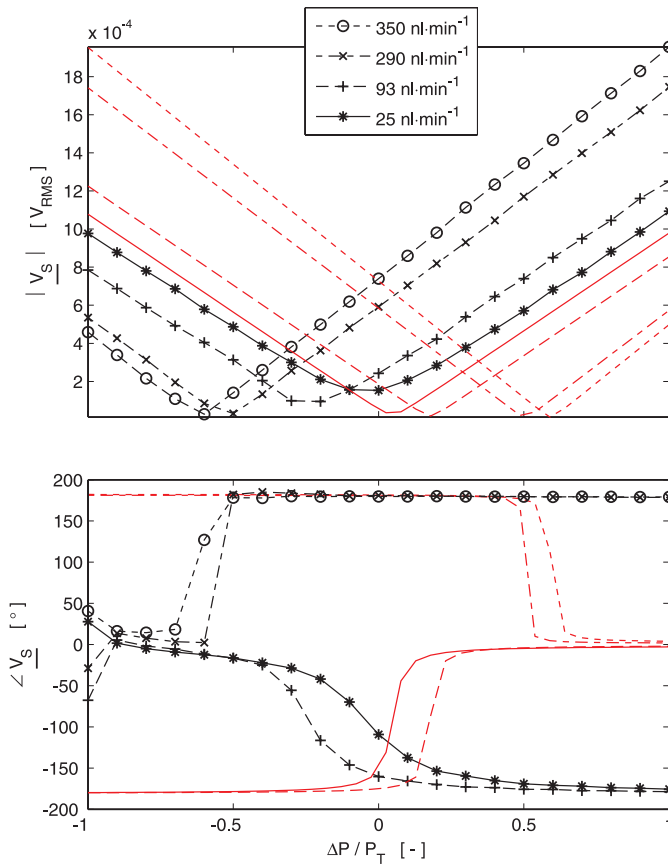


Fig. 8.9 Measured and calculated open-loop sensor response on power difference ΔP at various water flow rates.

ous water flow rates. These open-loop measurements show that it is difficult to use the phase of \underline{V}_S as a control signal. In Fig 8.8 the phase decreases, with increasing frequency. The FVM model predicts that this is also dependent on the type of fluid in the microchannel. This is because thermal conduction and heat capacity determine the phase of the temperature at the heater resistors, which does not need to be exactly the same as the phase in the power dissipated in the heaters. Figure 8.9 shows that the amplitude of \underline{V}_S at various flow rates has a clearly defined minimum, which can be obtained by using a control system.

Figure 8.10 shows the sensor response of the flow sensor obtained by determining the ΔP required for minimum amplitude of \underline{V}_S for various flow rates. The sensor output $\Delta P/P_T$ shows to be independent of operation frequency as well as total heating power. A nearly linear sensor output is obtained up to about $500 \text{ nl}\cdot\text{min}^{-1}$. The FVM model accurately indicates that at higher flow rates temperature gradients near the silicon substrate force the heater temperature to decrease (Fig. 8.5).

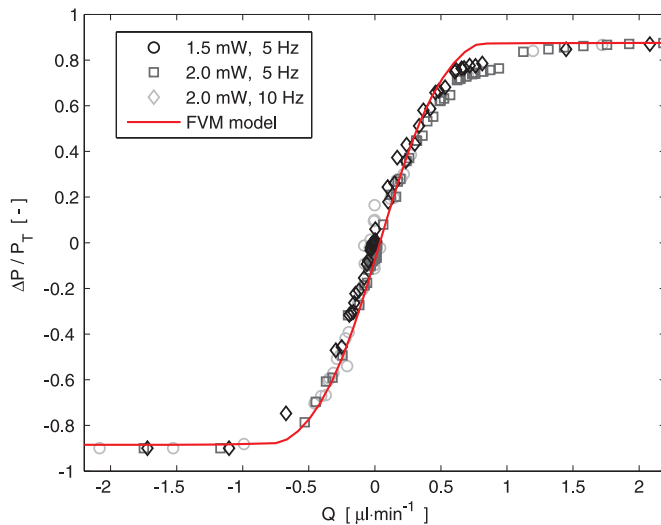


Fig. 8.10 Sensor output $\Delta P/P_T$ response on water flow rate Q at different frequencies and total heating power, compared to FVM model results.

A control-system was implemented based on gradient descent to obtain ΔP at minimum amplitude of \underline{V}_S (Fig. 8.11). The gradient is determined by two measurement points ($\Delta P_n \pm \delta P$) close to ΔP_n . Using small step-size for convergence, ΔP_{n+1} is placed in opposite direction to the gradient. Convergence can be slow when ΔP_n is far from minimum amplitude. A direct linear fit to find the minimum, when large changes in flow rate occur, can be made if during sensor operation an estimate

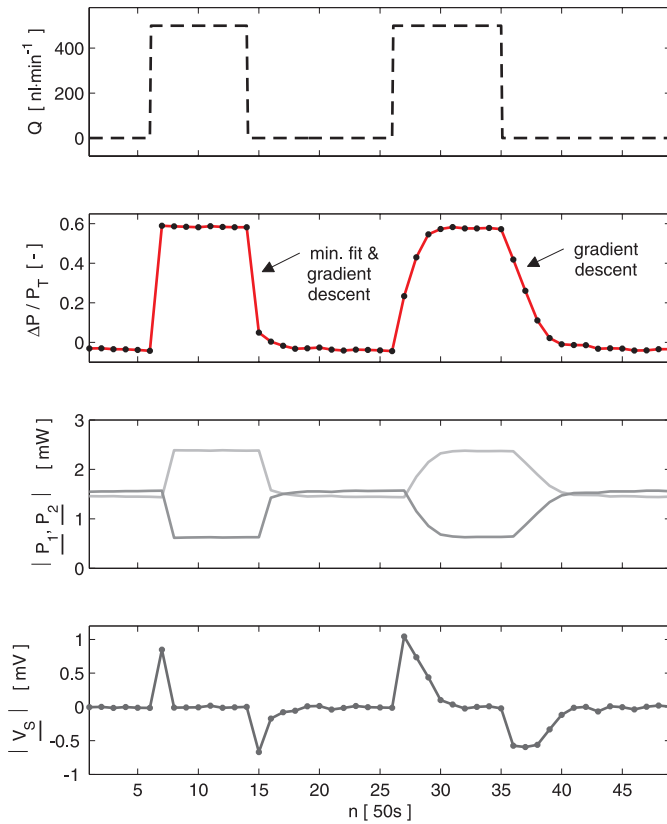


Fig. 8.11 Sensor control-system response on changing flow rate Q at 3 mW total heating power P_T . During the second change in flow only the gradient descent method is used, resulting in a significant slower response.

of the absolute value of the minimum amplitude of \underline{V}_S can be made. Accurate convergence has to be obtained by gradient descent, if the direct fit was not entirely accurate. The question remains how to optimally decide at what change in flow rate the linear fit should be applied. Fig. 8.11 shows that direct linear fit combined with gradient descent has a much faster response than when only using gradient descent. A sample time n of only 50 s could be obtained. This is mostly limited by the DSP lock-in amplifier (SRS 830). Much faster sampling can be obtained using dedicated electronics. Ultimately, the sample time is limited by the frequency of the heat waves measured with the lock-in amplifier, where an optimum has to be found between required sample time and the decrease in amplitude measured by the sensor resistor.

A drift measurement for 498 hrs was conducted, without water in the microchannel in an ambient environment. Figure 8.12 shows the power spectral density of the heater resistors and the drift in the sensor output ΔP . The heater resistors clearly show

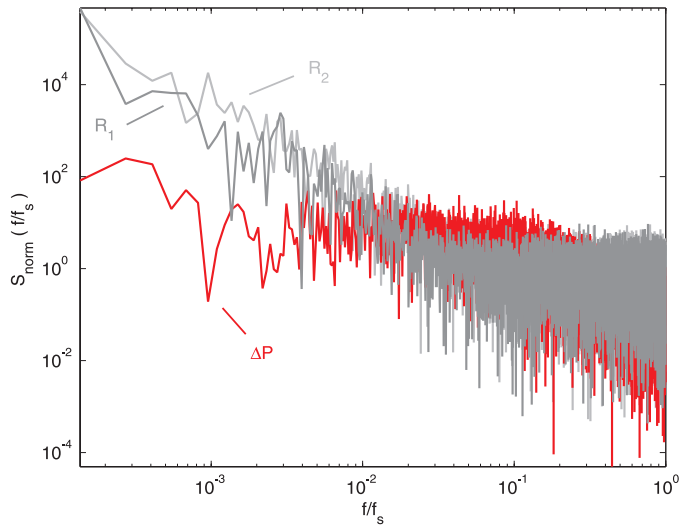


Fig. 8.12 Drift measurement for 498 hrs with $8.2 \cdot 10^{-3}$ Hz sampling frequency f_s , showing normalised power spectral density S_{norm} at 2 mW total power P_T .

$1/f^\alpha$ noise spectra, while the power spectral density for ΔP is nearly flat. The power spectral density for ΔP is even slightly flatter than in Fig. 5.11. Measurement points of ΔP indicate that thermal gradients over the chip are not measured. For instance, no day-night rhythm was observed in the time signal of ΔP , while it can be observed in a change in resistance values of the heater resistors.

8.5 Conclusions

A microchannel thermal flow sensor with integrated Al resistors has successfully been fabricated. The flow sensor is capable of measuring water flow rates up to about $500 \text{ nl} \cdot \text{min}^{-1}$, using the AC-driven temperature-balancing concept [8.5], providing a sensor output independent of operation frequency, resistance values and total heating power. Thermal operation of the flow sensor is well understood by the dynamic FVM model of the flow sensor, giving exact fits to measurement results. A control-system based on gradient descent shows stable operation independent of operation frequency and thermal properties. Optimisation of the control system is shown to be feasible. The obtained drift performance is similar, if not better, to the sensor design with thermopiles discussed in Chap. 5. A disadvantage is the slower response time, because relatively low frequencies have to be used in combination with long averaging times. Effects of thermal gradient are however eliminated with a relatively simple sensor structure.

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9

Nano-Nozzle Electrospray Emitters Fabricated by a Micro- to Nano-Fluidic Via Technology

This chapter describes nano-nozzle electrospray emitters fabricated by a micro- to nano-fluidic via technology. The main advantage of the technology is the ability to integrate freely-suspended nanochannels anywhere on a micro-fluidic chip. Nano-nozzle electrospray emitters were created on freely suspended microchannels. Leak-tight delivery of fluid from a fluidic reservoir was established through long microchannels. It was observed that other mechanisms aside from bulk capillary action contribute to filling of nanochannels coupled to microchannels. Stable electrospray IV-curves could be obtained from fabricated nano-electrospray emitters.^{9.1}

9.1 Introduction

Nano-fluidic tip based applications have emerged in the form of nano-fluidic fountain pens [9.1-9.3] and nano-electrospray emitters [9.4-9.6]. These applications require the delivery of fluid to functional nano-fluidic tips, possibly arranged in large arrays. Fluidic connection all the way up to the tip has mostly [9.7] been achieved by capillary slots or by extending nanochannels from fluid reservoirs [9.1-9.6]. In this chapter a fluidic via technology is described that allows for the coupling of microchannels to nanochannels. Microchannel fluidic networks are created beneath the wafer surface, facilitating the leak-tight chemical-inert delivery of fluid from macro-fluidic connections to nanochannels anywhere on the wafer surface. The mechanical strength of the microchannels can furthermore be utilised in making long freely suspended probes, with a nanochannel positioned right at the tip. The microchannels deliver fluid at low hydraulic resistance and large fluid volume ratios compared to connected nanochannels.

In this chapter nano-nozzle electrospray emitters are described, fabricated by the via technology. S. Koster and E. Verpoorte [9.8] presented a detailed review on microfluidics coupled to electrospray mass spectrometry (ESI-MS) [9.9-9.11]. A clear trend in on-chip integrated electrospray emitters was observed. In electrospray ionisation (ESI), reducing emitter aperture size is actively being pursued [9.5, 9.6], because of smaller droplets being generated. The enhanced ionisation efficiency obtained is of use to proteomics [9.12] and metabolomics. The via technology de-

^{9.1} This chapter is to be submitted to Lab on a Chip.

scribed in this chapter, provides a method to combine small aperture-size emitters, with microfluidics in a monolithically-integrated electro spray chip.

9.2 Experimental

9.2.1 Micro- to Nano-Fluidic Vias

Microchannel fluidic networks are created beneath the wafer surface for the on-chip delivery of bulk fluid, applying the surface channel concept [9.13]. Microfluidic devices (e.g. a thermal flow sensor [9.14]) can be integrated with nanochannels using the via technology described in this chapter. The microchannels are formed by isotropic dry-etched of the silicon wafer through $2\ \mu\text{m}$ wide etch holes in a $500\ \text{nm}$ silicon-rich silicon nitride (SiRN) layer. The etch holes are closed by a second $1.3\ \mu\text{m}$ SiRN low-pressure chemical vapour deposition (LPCVD), while the inside surface of the etched profile in the silicon wafer is simultaneously conformally coated [9.13].

Fluidic vias are created by selectively designed etch holes $3\ \mu\text{m}$ wide, preventing them from closing during SiRN deposition. Plugs are inserted in the open vias by $1\ \mu\text{m}$ poly-Si LPCVD deposition and patterning (Fig. 9.1a), where the deposition rate inside the microchannels was found to decrease with distance from the via.

The sealed microchannels leave a nearly planar wafer surface on which sacrificial nanochannels ($4\ \mu\text{m} \times 200\ \text{nm}$), with $200\ \text{nm}$ SiRN capping layer, are connected to the poly-Si plugs (Fig. 9.1b). Nano-lithographical methods could be applied to integrate nanochannels with even smaller dimensions.

Electro spray chips were released using $25\text{wt}\%$ KOH at $75\ ^\circ\text{C}$ [9.15], where the sacrificial poly-Si is etched after inlets and outlets have been created (Fig. 9.1c). Only

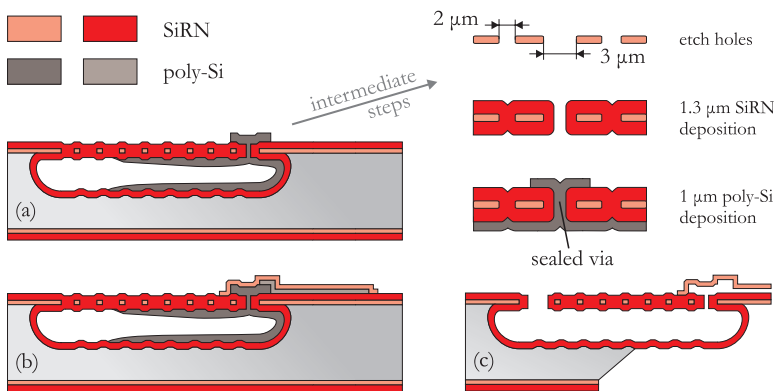


Fig. 9.1 Process schematic for the fabrication of via connections between surface microchannels [9.13] and nanochannels, with a) fabrication of via (see intermediate steps), b) fabrication of nanochannels and c) top- and bottom side SiRN etching of SiRN layers, where simultaneously openings are etched in the micro- and nanochannel, followed by KOH release of the electro spray chip.

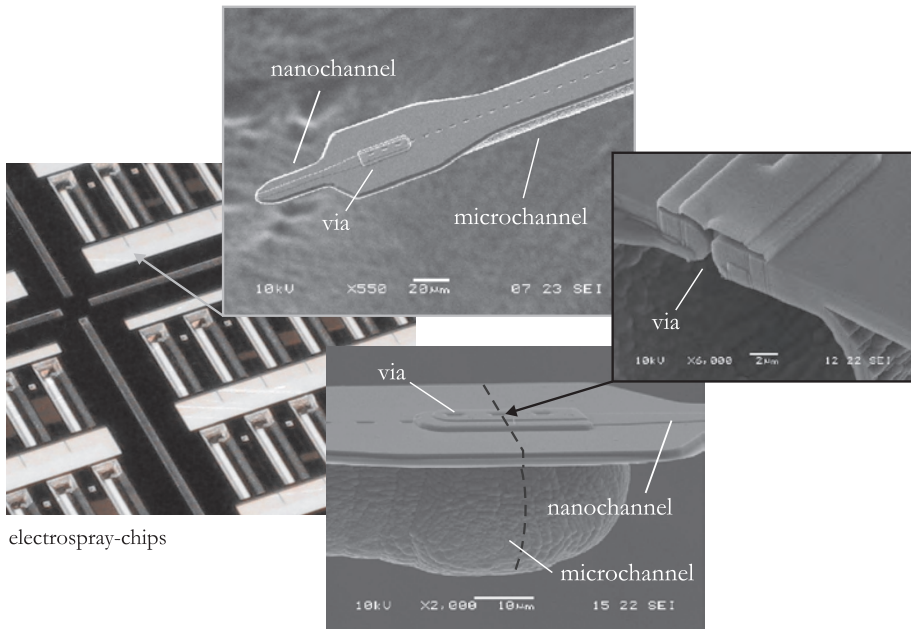


Fig. 9.2 Nanochannel electro spray emitters, with via connection between micro- and nanochannels. The rightmost inset shows a FIB cut through a fluidic via.

in cases where etching occurs from a nanochannel into a sealed microchannel without an outlet, it was observed that via connections break. This is caused by a sudden increase in H_2 gas produced due to a large surface array of poly-Si being exposed.

Figure 9.2 shows a fabricated electro spray probe with a $60\ \mu\text{m}$ long nanochannel, where the inset shows a focused ion-beam (FIB) cut through the fluidic via connection between micro- and nanochannel.

9.2.2 Filling Experiments

Capillary filling of the electro spray tip (Fig. 9.2) was attempted by applying deionised (DI) water at the inlet. Capillary filling of the microchannel was found to be restricted by the large volume of air in the microchannel, which needs to be pushed out through the nanochannel. This was confirmed by preparing several nanochannel lengths from the initial $60\ \mu\text{m}$ long nanochannel (Fig. 9.2), where the filling speed was found to be constant in time during restricted capillary filling and inversely proportional to the nanochannel length. Subsequent refilling of previously filled electro spray chips resulted in instant wetting of the nanochannel with the formation of bubbles along the microchannel, likely due to an already saturated atmosphere present in the microchannel.

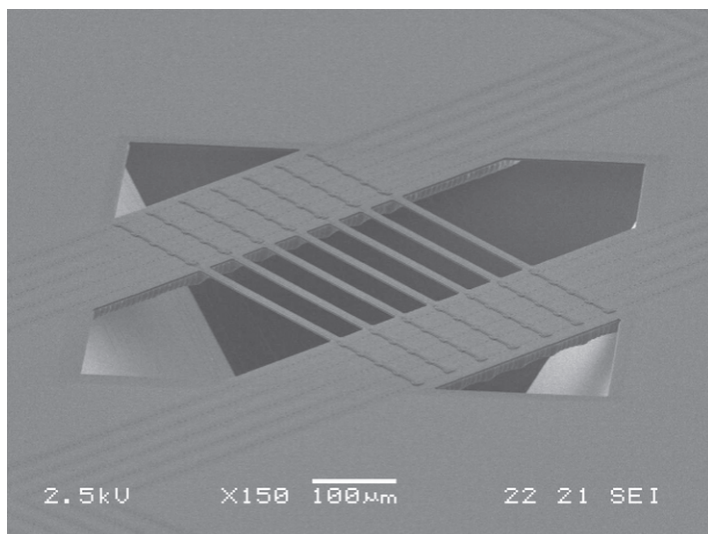


Fig. 9.3 Micrograph of freely suspended nanochannels connected to microchannels for on-chip transport of fluid.

A fluidic chip was fabricated to study filling mechanisms, containing freely suspended nanochannels, connected to separate microchannels on either side for the transport of fluid (Fig. 9.3). The on-chip freely-suspended nanochannels could be used for fluid-resonance applications. D. Westberg et al. [9.16] have created a fluid density sensor using sacrificial nanochannels and T.P. Burg et al. [9.17] have used nanochannel cantilevers for the weighing of biomolecules and single cells.

The filling of the chip was studied by applying DI water at the inlets of the microchannels connected to the lower-right part of the nanochannels in Figs. 9.3-9.4, while the microchannels on the upper-left part were not filled.

Figure 9.4 shows subsequent snapshots taken, during filling of the chip, where after 1.6 s the fluid reaches the via connections through a crossflow, where the air in the microchannels is pushed out through the microchannel outlets. The fluid then enters through the fluidic vias into the nanochannels. Liquid appears on the opposite ends of the nanochannels first, just before bulk fluid with a visible corner flow [9.18] enters the nanochannels from the wetted microchannels at 3.8 s. The unexpected filling observed, with liquid appearing on opposite ends first, is likely caused by invisible transport mechanisms, such as vapour diffusion, film flow and corner flow [9.19]. After 6.1 s air bubbles remain, which slowly dissolve until the nanochannels are completely filled at 17.3 s. The pinning of the fluid contact line [9.20] at the fluidic via connections on the opposite ends of the nanochannels prevents further capillary filling of the connected microchannels.

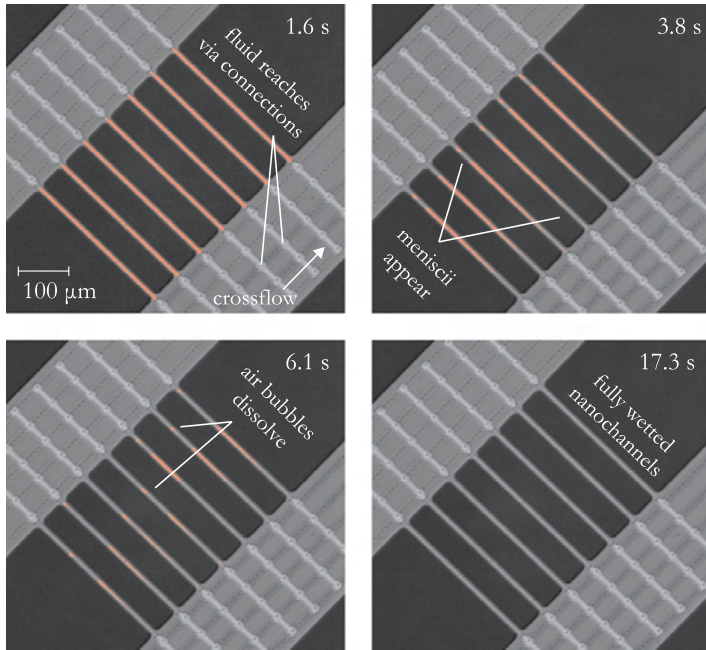


Fig. 9.4 Snapshots taken during the filling with DI water of freely suspended nanochannels. Each frame has been enhanced with the same image filter.

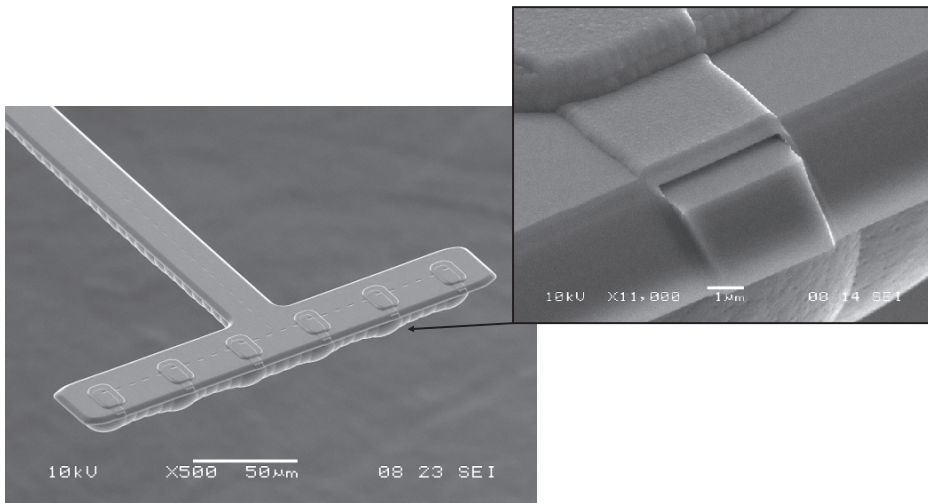


Fig. 9.5 Micrographs of nano-nozzle electro spray emitters on a microchannel probe. The inset shows a nano-nozzle with $4 \mu\text{m} \times 200 \text{nm}$ aperture size. The poly-Si acts as a delayed mask during SiRN etching, resulting in a protrusion near the nozzle opening.

9.2.3 Nano-Nozzle Electropray Emitters

A redesign of the nanochannel electropray emitter was made based on experimental filling results, taking vapour diffusion, corner flow, film flow [9.18-9.19] and gas-flow restriction into consideration. Electropray chips were batch fabricated containing microchannel probes with integrated nano-nozzle emitters, where the nanochannel length was designed to be $3\ \mu\text{m}$ (Fig. 9.5), facilitating the fast filling of the microchannel. With the redesign, electrospraying can be performed from multiple emitters for higher spray throughput, while the air inside the microchannel can easily escape during capillary filling of the fluidic chip.

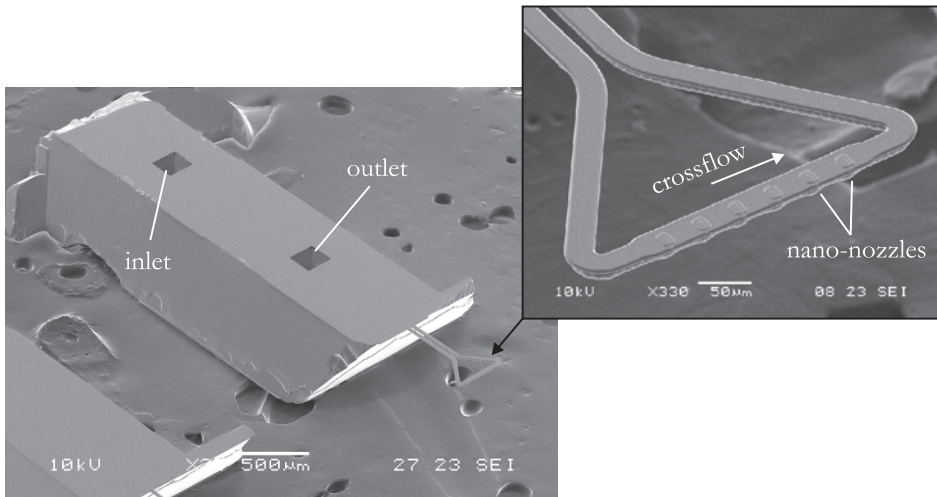


Fig. 9.6 Micrographs of nano-nozzle electropray emitters, where a crossflow can pass the nano-nozzles from an on-chip inlet to an outlet.

Additional designs were made, where the nano-nozzles are integrated on a continuous microchannel, such that a crossflow can pass the nozzles (Fig. 9.6). This eliminates problems with air bubbles remaining in the microchannel and even electrospraying from a continuous flow can be achieved.

9.2.4 Electropray-Ionisation Setup

Both the nanochannel electropray chip after FIB cutting (Fig. 9.2) and the nano-nozzle emitter chip (Fig. 9.5) were used for electrospraying. A Keithley 237 high-voltage source-measurement unit (SMU) was used to apply an electric field (Fig. 9.7). Triax-cable connections with active guarding were used for high spray-current resolution. The electropray chip was filled with spraying solution and a probe needle

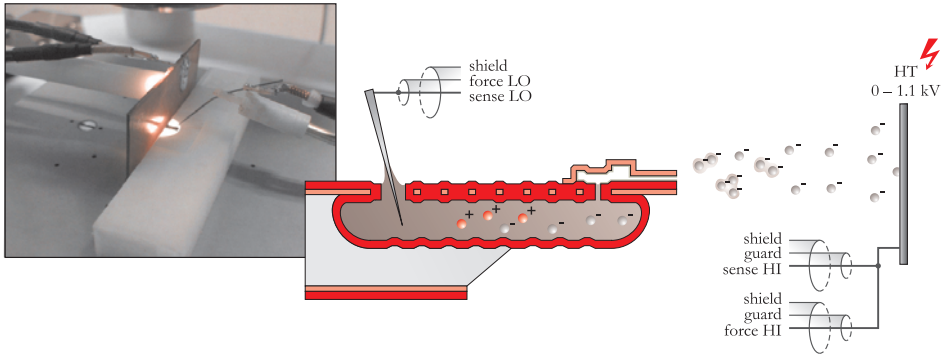


Fig. 9.7 Electro spraying from nano-nozzle emitter, with pA resolution using the Keithley 237 high-voltage SMU and triax-cable connections.

was inserted in the inlet. The counter electrode was isolated from the rest of the setup using a Teflon rod, with at least several TΩ resistance. The counter electrode distance to the emitter was determined using an adjustable micrometer. With the setup, pA resolution was obtained up to 1.1 kV counter electrode voltage applied. Without spraying, IV-curves show hysteresis due to capacitive effects in cabling and the solution inside the microchannel. Therefore, an 8 s delay was used between applying counter electrode voltage and acquisition of the spray current.

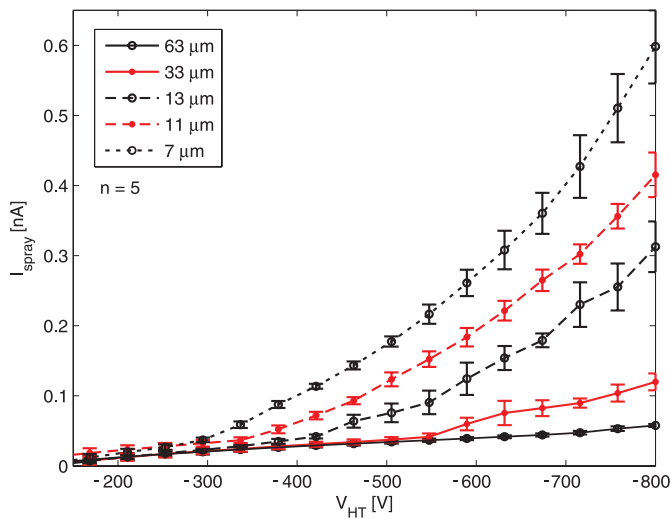


Fig. 9.8 Electro spray IV-curves of fully wetted FIB-cut 15 μ m nanochannel emitter at various distances to the cathode, using $H_2O:MeOH(50:50v\%)$ solution ($4.6 \mu S \cdot cm^{-1}$).

9.3 Results and Discussion

Electrospray IV-curves were obtained on a fully-wetted nanochannel emitter (Fig. 9.2) reduced in length to 15 μm , using $\text{H}_2\text{O}:\text{MeOH}$ (50:50v%) solution, without back pressure applied (Fig. 9.8). Reproducible IV-curves in the nA range were obtained at various distances of the emitter to the cathode. The spray current is generally very stable (small standard deviation) near the onset of spraying. The spray current shows a near exponential relation at higher spray voltages.

Several nano-nozzle emitter chips (Fig. 9.5) were filled with $\text{H}_2\text{O}:\text{MeOH}$ (50:50v%) solution for spraying. However, during capillary filling the solution did not fully wet the nano-nozzle emitters. The microchannels always filled completely, but the fluid contact line did not cross the fluidic vias. The fluidic chip with freely suspended nanochannels (Fig. 9.3), also fabricated on the same wafer, did however fill with the same $\text{H}_2\text{O}:\text{MeOH}$ (50:50v%) solution. It is likely, considering filling experiment results discussed earlier, that the fluid contact line in both cases is pinned on the fluidic vias, which can be overcome by the development of a saturated atmosphere in the nanochannel. This can explain the unsuccessful filling of the 3 μm short nano-nozzle emitters, due to a high evaporation rate, such that a saturated atmosphere cannot develop. Remarkably spraying does occur when using a high-conductive $\text{H}_2\text{O}:\text{MeOH}$ (50:50v%)+ $\text{CH}_3\text{-COOH}$ (1v%) solution, where electrospray IV-curves could be obtained from non fully-wetted nano-nozzle emitters (Fig. 9.5). The exact spraying

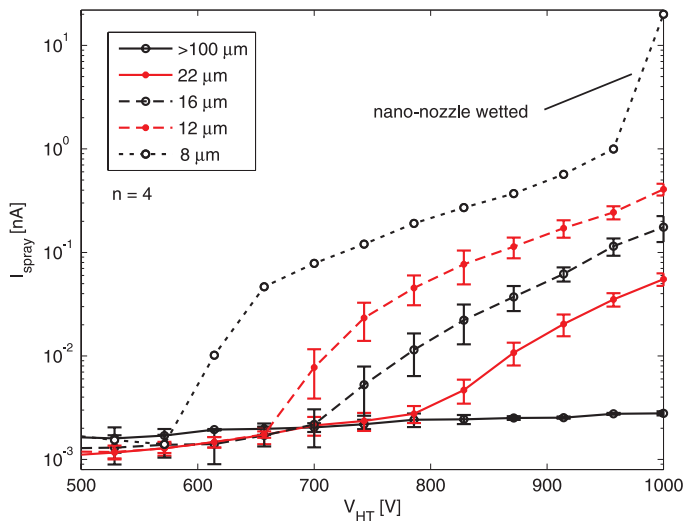


Fig. 9.9 Electro spray IV-curves of non fully-wetted nano-nozzle emitters at various distances to the anode, using $\text{H}_2\text{O}:\text{MeOH}$ (50:50v%)+ CH_3COOH (1v%) solution ($120 \mu\text{S}\cdot\text{cm}^{-1}$).

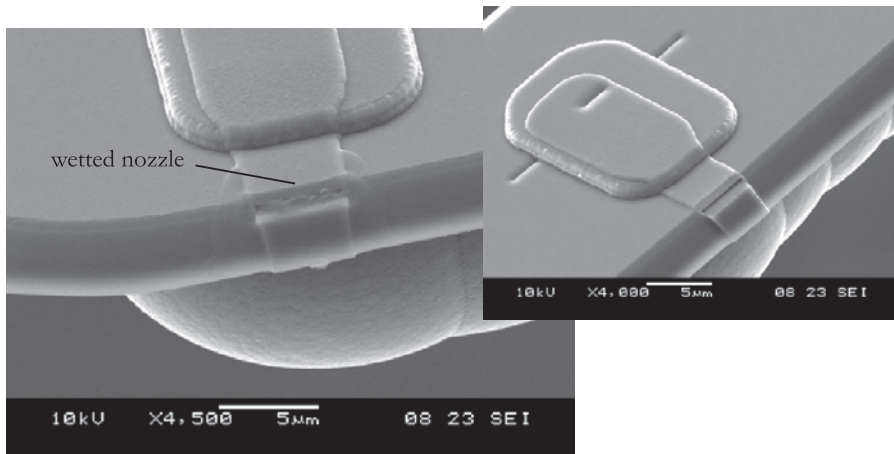


Fig. 9.10 Micrographs of the nano-nozzles, wetted after electro spraying at $8\ \mu\text{m}$ anode distance applying $1\ \text{kV}$ compared to non fully-wetted nozzle.

mechanism is currently not understood. It is possible that nano-sized droplets are directly sprayed from the nano-nozzles emitters [9.21].

IV-curves, at various distances to the anode, were obtained (Fig. 9.9). During the last IV-curve measured at $8\ \mu\text{m}$ distance it was observed that the nano-nozzle closest to the anode, due to a small angular misalignment, became fully wetted. The strong electric field apparently pulled the fluid contact line through the fluidic via. The spray current instantly became $20\ \text{nA}$, which was set as current limit. The electro spray chip was taken for SEM inspection directly after the electro spray measurement. Figure 9.10 shows fluid in- and outside of the fully wetted nano-nozzle, after the sample had been SEM coated. The other nozzles were no different from nozzles inspected just after fabrication, confirming that the IV-curves in Fig. 9.8 were obtained on non fully-wetted emitters.

9.4 Conclusions

A novel micro- to nano-fluidic via technology has been developed. It was demonstrated that bulk fluid can be transported from macro-fluidic connections to on-chip nanochannels. The technology has shown potential in applications where on-chip freely suspended nanochannels are required. The capillary filling of fluidic chips was studied and it was concluded that vapour diffusion, corner flow, film flow, gas-flow restriction and pinning of fluid contact lines have to be considered when designing coupled micro- to nano-fluidic devices. Better understanding of these transport mechanisms during filling is required for the via technology to be applicable to other fluidic designs. Proper capillary filling of microchannels can however be readily

achieved by applying crossflow filling of connected nanochannels, such that the air in the microchannel can escape through a microchannel outlet.

Stable electrospray IV-curves were obtained from nano-nozzle emitters, despite the fact that the nozzles did not fully wet. This provides for an interesting spraying mode, which requires further investigation. Especially, the size distribution of possibly emitted nano-droplets, compared to fully wetted emitters, is of interest.

The prospect of the nano-nozzle electrospray chip is to increase emitter density, while reducing aperture size. Nano-droplet emission at higher densities can then be achieved. The nano-droplets emitted could be used for thin-film deposition. Thin-film deposition with large aperture-size emitters [9.22, 9.23] is limited by the need for the evaporation of solvents. R. Saf et al. [9.24] have used drying gas to dry of solvents and used electrostatic lenses for focussing, creating structured organic thin-films under atmospheric pressure.

The combination of the nano-nozzle emitters with surface microchannels allows for the integration of a high-pressure liquid chromatography (HPLC) column, where HPLC packing material can be packed up to the nano-nozzle emitters. Alternatively, capillary electrophoresis (CE) can be performed with on-chip.

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10

Conclusions and Future Research

In this final chapter conclusions are drawn on a number of important research results described in previous chapters. These results relate to: the fabrication technology used to fabricate the micro flow sensors presented in this thesis, improvements in accuracy by reduction of material drift influences and reduction of influences by external temperature gradients. The chapter ends with an outlook on possible future research in low-drift micro flow sensors.

10.1 Surface Channel Technology

The existing buried channel technology [10.1] has been modified to make it suitable for application in flow sensors for the measurement of extremely small flow rates in the $\text{nl}\cdot\text{min}^{-1}$ range. The reason for developing the microchannel fabrication process was that previously used sacrificial layer based technology proved not to be suitable for these applications due to the large pressure drops associated with surface micromachined channels (see e.g. [10.2-10.4]). The developed surface channel technology (Chap. 3) allows for the fabrication of microchannels with semicircular cross sections, with low hydraulic resistance.

After fabrication of the microchannels a completely planar wafer surface remains, so that transducer structures can easily be integrated in close proximity to the fluid. The planar wafer surface allows for standard lithographic processing. Furthermore, high-temperature fabrication steps are possible, because the microchannels are made of low-stress silicon-rich silicon-nitride (SiRN). For example the Al/poly-Si⁺⁺ thermopiles used in micro flow sensors presented in this thesis were made by solid-source diffusion (SSD) of boron at 1050 °C, but also other high-temperature steps like wafer bonding have been demonstrated. Wafer level packaging of surface microchannel devices using wafer bonding is therefore possible and should be investigated further.

After integration of transducer structures the microchannels can easily be released by selectively etching the silicon substrate surrounding the microchannels. Wet etching in KOH can be used if this does not affect transducer material. Alternatively dry-etching can be used, where transducers are protected by photoresist during the release. In this thesis thermally-isolated freely suspended microchannels have been used in the fabrication of micro thermal flow sensors, but the microchannel technology can also be used in other applications that require freely suspended structures (e.g. micro heat exchangers, micro reactors or fluidic resonators [10.5, 10.6]).

In Chap. 9 the surface channel technology is extended to include nanochannels that are fabricated on the wafer surface and connected to the semicircular microchannels

using fluidic vias. The via technology was demonstrated with the realisation of nano-nozzle electrospray emitters. The technology can be used in applications where micro fluidic components need to be integrated with nanofluidic channels (e.g. an HPLC column or a CE capillary with electrospray emitters). The combination of nano-channels with surface microchannels enables the fabrication of nanofluidic structures positioned at the tip of freely suspended microchannel probes.

10.2 Material-Drift Compensation

Drift in material properties of thin-film materials used in micro flow sensors is inevitable. Resistors are heated by Joule heating and the heating and current density cause an increase in $1/f^\alpha$ noise, by the diffusion of material due to electromigration [10.7, 10.8]. In this thesis it was shown that power control on heater resistors is very efficient in reducing influences of material drift in heater resistors (Chap. 5, 8). Power control requires the measurement of both the voltage across and the current through the resistor by four-point measurement, so that the resistance value can be calculated and the current adjusted accordingly. To avoid influence of thermocouple offset voltages on interface wires to the chip a heating current with alternating polarity has to be used. In this way resistance drift is largely cancelled. Drift by non-uniform changes in resistance material remains, because this may result in redistribution of the dissipated power on the heater resistor. This secondary effect has been observed at high-temperature (Fig. 5.12). It shows to be the main cause of residual sensor output fluctuation.

Although power control can effectively be used to eliminate $1/f^\alpha$ noise in heating elements, a low-drift sensing element is required in order to obtain a low-drift flow sensor. Therefore, a thermopile was used as sensor element, which has inherently zero-offset if the thermopile is at a uniform temperature. Sensitivity drift of the thermopile remains, which can be eliminated by using a temperature-balancing control system as demonstrated in Chap. 5. The control system adjusts the heating powers such that the thermopile voltage is cancelled. The resulting heating powers are a measure for the flow, independent of the thermopile sensitivity.

Accurate read-out of the thermopile voltage requires that interface wires to the chip are close to each other in order to retain an almost equal temperature, avoiding offset voltages. Further improvement in reducing influence by interface offset voltages can be obtained by integrating more junctions in the thermopile (Chap. 6) and by using a low-drift or chopper amplifier close to the chip.

With interface offset voltages eliminated, temperature gradients along the thermopile leads in combination with material inhomogeneities (giving a non-uniform Seebeck coefficient along the leads) [10.9, 10.10] may still result in a thermopile offset voltage even when thermopile junctions on opposite sides are at the same

temperature. As a result, temperature-balancing control can give zero thermopile voltage, but this does not necessarily correspond to zero temperature difference between the thermopile junctions. Therefore, a thermopile temperature-balancing flow sensor may still give drift in offset voltage in case of inhomogeneities being introduced in the thermopile leads. The long-term drift properties of the thermopiles in combination with temperature-balancing control has to be investigated further by extremely long-term drift measurements, much longer than required for long-term material drift to be apparent in heater resistors.

Another method to obtain a drift-free measure for temperature difference is to use a sensor resistor in combination with heat-waves. This possibility was explored in Chap. 8 and it was shown that similar and possibly better long-term accuracy can be achieved at the expense of a slower response time of the sensor.

10.3 Temperature-Gradient Compensation

In Chap. 5 it was shown that drift in sensor materials can be effectively eliminated. However, the sensor output appeared to be dependent on temperature gradients induced by changes in ambient temperature (e.g. day-night rhythm was visible in the sensor output). Additional measurements in an oven also showed that the oven temperature can modulate thermal gradients over the chip, due to differences in heat leakage to ambient through interface wires going outside the oven. With material drift largely compensated, the drift performance of the micro flow sensor was now limited by these external temperature-gradient effects.

To solve this problem a U-shaped flow sensor was designed (Chap. 6). This resulted in a significant improvement, although conduction through the surrounding air still resulted in an output from the thermopile in the presence of a thermal gradient across the chip. To fully eliminate the influence of externally induced gradients a completely symmetrical flow sensor was designed (Chap. 7).

Also the sensor based on heat-waves (Chap. 8) is independent of thermal gradients over the chip, because only heat-waves at the lock-in frequency are detected.

10.4 Outlook on Low-Drift Micro Flow Sensing

Long-term flow sensor accuracy has been greatly enhanced by near complete elimination of drift factors (temperature-gradient and material-drift compensation). What remains to be optimised is sensor precision at a given acquisition rate for measuring liquid flows in the $\text{nl}\cdot\text{min}^{-1}$ range. Relatively low acquisition rates were obtained with measurement equipment used during measurements presented in this thesis. The acquisition rates used are much slower than the time needed to obtain

thermal equilibrium at a given power setting, which is in the order of milliseconds, allowing for enormous improvements in acquisition rate.

At the time of writing, a temperature-balancing control system is under investigation, built using zero-drift opamps (AD8628) and a data-acquisition board (NI-DAQ). Initial measurements show that temperature-balancing, using a linear least square method, and power control on heater resistors, with currents having alternating polarity, can be performed on flow sensors presented in Chap. 5 with $\text{nl}\cdot\text{min}^{-1}$ resolution at 0.5 Hz acquisition rate. The largest limiting factor in increasing precision and acquisition rate are external disturbances superimposed on the thermopile voltage, due to the relatively high thermopile impedance and the high-impedance opamp used.

The most important topic for further research is the improvement of the signal to noise ratio. Improvements are possible by using adequate electrical shielding and optimisation of thermopile materials, obtaining high sensitivity at low impedance. The total heating power can also be increased and should be optimised for life-time and reliability of the micro flow sensor.

Further investigation is needed to find an optimal flow sensor design. In Chap. 7 it was shown that a symmetrical design with a meandering microchannel can be used to fully compensate for external temperature gradients. However, for optimal low-drift performance every heater resistor should have individual power control, which requires the total number of heater resistors to be minimised. The minimal symmetric design, consisting of two parallel microchannels with opposite flow direction and independent power control on four heater resistors is therefore close to an optimal low-drift flow sensor design. An important aspect that should be investigated is how to obtain highest sensitivity. The two parallel microchannels allow thermopile leads to be suspended between the microchannels (as with the U-shaped sensor in Chap. 6) increasing the amount of thermocouple junctions, however at the expense of increased thermal conduction between the microchannels.

Finally, further research is needed to improve calibration of the micro flow sensors. The flow sensors presented in this thesis were calibrated using an elevation head, where a water container was put at a specific height compared to a container on a balance. This provided for a stable flow through the microchannel, however the exact flow rate, measured by measuring the weight on the balance, was still difficult to obtain at high precision. Typically in the order of $100 \text{ nl}\cdot\text{min}^{-1}$ evaporation from the container on the balance was measured. Evaporation was largely compensated for by measuring the evaporation rate between each measurement point and long averaging time was used for flow-rate calibration. Improvements can be made by using a smaller container on the balance and by using a thin layer of oil on top of the fluid to reduce evaporation. The obtained maximum resolution of the flow sensors has so far been limited by the calibration setup. A calibration setup for accurately determining flow

rates down to $1 \text{ nl}\cdot\text{min}^{-1}$ and below will be a challenge, but is needed to accurately determine the resolution of the fabricated low-drift micro flow sensors.

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A.1 Surface Microchannel Fabrication

Nr.	Process	Process Parameters	Design related parameters and remarks
1.	Substrate selection - Silicon <100> OSP (#subs001)	CR112B / Wafer Storage Cupboard Orientation: <100> Diameter: 100 mm Thickness: 525 μm +/- 25 μm Polished: Single side Resistivity: 5-10 $\Omega\cdot\text{cm}$ Type: p	nr. wafers: ? nr. dummies: ?
2.	Cleaning Standard (#clean003)	CR112B / Wet-Bench 131 HNO ₃ (100 %) Selectipur: MERCK HNO ₃ (69 %) VLSI: MERCK • Beaker 1: fuming HNO ₃ (100 %), 5 min • Beaker 2: fuming HNO ₃ (100 %), 5 min • Quick Dump Rinse <0.1 μS • Beaker 3: boiling (95 °C) HNO ₃ (69 %), 10 min • Quick Dump Rinse <0.1 μS • Spin drying	
3.	Etching HF (1%) Native Oxide (#etch027)	CR112B / Wet-Bench 3-3 HF (1 %) VLSI: MERCK 112629.500 • Etch time: >1 min • Quick Dump Rinse <0.1 μS • Spin drying	
4.	LPCVD SiRN - low-deposition rate (#depo002)	CR125C / Tempress LPCVD new system 2007 program: SiRN01/N2 Tube: G3 • “Use 5-8 boat fillers in front and back of the boat to achieve specifications.” • SiH ₂ Cl ₂ flow: 77.5 sccm • NH ₃ flow: 20 sccm • temperature: 820/850/870 °C • pressure: 150 mTorr • deposition rate: $\pm 4 \text{ nm}\cdot\text{min}^{-1}$ • Nr: ± 2.18 • Stress (range): 200-280 Mpa • Boat position 12: 200 MPa (centre of the boat) • Boat position 1: 280 MPa (front of the boat) • Uniformity/wafer: <2 % • Uniformity over the boat: (20 wafers): < 8 %	layer thickness: 500 nm program: SiRN01 nom. deposition rate: 3.62 $\text{nm}\cdot\text{min}^{-1}$ nom. deposition time: 2 hr 20 min
5.	Ellipsometer Measurement (#char007)	CR118B / Plasmos Ellipsometer	
6.	Sputtering of Cr (Sputterke) (#depo017)	Eq.Nr. 37 / Sputterke Cr Target (gun #: “see mis logbook?”) • Electrode temp.: water cooled electrode • Ar flow: app. 80 sccm, pressure depending! • Base pressure: < 1.0·10 ⁻⁶ mbar • Sputter pressure: 6.6·10 ⁻³ mbar • power: 200 W • Depositionrate: 15 $\text{nm}\cdot\text{min}^{-1}$	layer thickness: 50 nm nom. deposition time: 4 min 10 s “A Cr mask is used for proper definition of the 2 μm wide etch holes.”

7.	Lithography - Vapor prime HMDS (gas phase) (#lith002)	CR112B / Furnace Vapor HMDS HexaMethylDiSilazane (HMDS) <ul style="list-style-type: none"> Furnace: Lab-line Duo-vac-oven “See manual for operating the system.” Dehydration bake (150 °C, low pressure): 30 min Vapor prime HMDS at 150 °C: 5 min “Before coating let cooldown the wafers to room temperature.” 	
8.	Lithography - Coating Olin907-17 (#lith005)	CR112B / Suss Micro Tech Spinner (Delta 20) Hotplate 95 °C Olin 907-17 <ul style="list-style-type: none"> Spin Program: 4 (4000 rpm, 20 s) Prebake (95 °C): 90 s 	
9.	Lithography - Alignment & Exposure Olin 907-17 (EV620) (#lith021)	CR117B / EV620 Electronic Vision Group 620 Mask Aligner <ul style="list-style-type: none"> Hg-lamp: 12 mW·cm⁻² Exposure Time: 4 s 	mask: CHNL , recipe: first mask exposure time: 3.5 s mask: 5" 2.3 mm sub: 4" 0.53 mm contact: vacuum flat rotation angle: 0° “Use EV with robot arm to achieve good <100>-alignment.”; “Check the vacuum ring.”
10.	Lithography - Development Olin Resist (#lith011)	CR112B / Wet-Bench 11 Developer: OPD4262 Hotplate 120 °C (CR112B or CR117B) <ul style="list-style-type: none"> After Exposure Bake (120 °C): 60 s Development: <ul style="list-style-type: none"> Time: 30 s in Beaker 1 Time: 15-30 s in Beaker 2 Quick Dump Rinse <0.1µs Spin drying 	bkr1 develop time: 30 s bkr2 develop time: 30 s
11.	Optical microscopic inspection - Lithography (#char001)	CR112B / Nikon Microscope	“The width of the etch holes must be close to 2 µm.”
12.	Ozone anneal of Olin 907 (to improve wetting) (#lith038)	CR116B-1 / UV PRS-100 “To improve wetting during etching of Chromium layers.” <ul style="list-style-type: none"> time: 300 s 	“Improves wetting of the Olin907-17 resist.”
13.	Etching of Cr Wet (#etch034)	CR116B / Wet-Bench 2 Chromium etch LSI Selectipur: MERCK 111547.2500 <ul style="list-style-type: none"> Quick Dump Rinse <0.1 µs Spin drying Etchrates: 100 nm·min⁻¹ 	etch time: 30 s “Pretet wafers in DI water before Cr etching to further improve wetting of the Olin907-17 resist.”; “Minimise underetch after 30 s by QDR.”
14.	Optical microscopic inspection - Lithography (#char001)	CR112B / Nikon Microscope	“No Cr underetch should be visible.”
15.	Plasma etching - chamber cleaning (Etske) (#etch003)	CR102A / Elektrotech PF310/340 Select chamber for desired etch process Select electrode for desired etch process <ul style="list-style-type: none"> Electrode temp.: 10 °C or 25 °C O₂ flow: 20 sccm pressure: 50 mTorr power: 150 W DC-Bias: 780 V “Chamber is clean when plasma color is white.”	“Use quartz electrode.”
16.	Plasma etching SiN (Etske) (#etch004)	CR102A / Elektrotech PF310/340 Dirty chamber Styros electrode <ul style="list-style-type: none"> Electrode temp.: 10 °C CHF₃ flow: 25 sccm O₂ flow: 5 sccm pressure: 10 mTorr 	layer thickness: 500 nm min. etch time: 8 min 30 s nom. etch time: 10 min “Use quartz electrode.”

17.	Argon Sputter etched (#Etch074)	<ul style="list-style-type: none"> power: 75 W Etchrate SiN : 50 nm·min⁻¹ (for V_{DC}=-460 V) Etchrate SiN : 75 nm·min⁻¹ (for V_{DC} = -580 V) Etchrate Olin resist : 95 nm·min⁻¹ “If DC-Bias < 375 V apply chamber clean (#etch003).” CR 125c/Adixen SE Application: removal of native oxide and/or residue <table border="1" data-bbox="452 356 738 624"> <thead> <tr> <th>Parameters</th> <th>Value</th> </tr> </thead> <tbody> <tr> <td>Gas</td> <td>Ar</td> </tr> <tr> <td>Flow [sccm]</td> <td>20</td> </tr> <tr> <td>Time [s]</td> <td>?</td> </tr> <tr> <td>APC [%]</td> <td>100</td> </tr> <tr> <td>ICP [W]</td> <td>500</td> </tr> <tr> <td>CCP [W] RF</td> <td>10</td> </tr> <tr> <td>SH [mm]</td> <td>200</td> </tr> <tr> <td>Electrode temp. [°C]</td> <td>+20</td> </tr> <tr> <td>He [bar]</td> <td>10</td> </tr> </tbody> </table>	Parameters	Value	Gas	Ar	Flow [sccm]	20	Time [s]	?	APC [%]	100	ICP [W]	500	CCP [W] RF	10	SH [mm]	200	Electrode temp. [°C]	+20	He [bar]	10							
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Electrode temp. [°C]	+20																												
He [bar]	10																												
18.	Plasma etching Silicon for semi-isotropic channels (SCT) (#etch063)	CR 125c/Adixen AMS 100 SE Applications: semi-isotropic channels (SCT) Pre-etch step: same values, incl. CCP of 30 W, time 30 s. <table border="1" data-bbox="452 726 847 1119"> <thead> <tr> <th>Parameters</th> <th>Value</th> </tr> </thead> <tbody> <tr> <td>SF₆ [sccm]</td> <td>400</td> </tr> <tr> <td>He-backside cooling [mbar]</td> <td>10</td> </tr> <tr> <td>ICP [W]</td> <td>2500</td> </tr> <tr> <td>CCP On/Off [ms]</td> <td>0</td> </tr> <tr> <td>p [Pa]</td> <td>10</td> </tr> <tr> <td>APC position [%]</td> <td>100</td> </tr> <tr> <td>Electrode temperature [°C]</td> <td>20</td> </tr> <tr> <td>Substrate height [mm]</td> <td>110</td> </tr> <tr> <td>Etch rate SiO₂ [μm·min⁻¹]</td> <td>?</td> </tr> <tr> <td>Etch rate Olin 907 [μm·min⁻¹]</td> <td>0.250</td> </tr> <tr> <td>Etch rate -Silicon (μm·min⁻¹)</td> <td>1.2 μm·min⁻¹ (for 2 μm slits-SCT)</td> </tr> <tr> <td>Etch rate SiRN [nm·min⁻¹]</td> <td>25</td> </tr> </tbody> </table>	Parameters	Value	SF ₆ [sccm]	400	He-backside cooling [mbar]	10	ICP [W]	2500	CCP On/Off [ms]	0	p [Pa]	10	APC position [%]	100	Electrode temperature [°C]	20	Substrate height [mm]	110	Etch rate SiO ₂ [μm·min ⁻¹]	?	Etch rate Olin 907 [μm·min ⁻¹]	0.250	Etch rate -Silicon (μm·min ⁻¹)	1.2 μm·min ⁻¹ (for 2 μm slits-SCT)	Etch rate SiRN [nm·min ⁻¹]	25	etch rate: 1.2 μm·min ⁻¹ radius: ? etch time: ?
Parameters	Value																												
SF ₆ [sccm]	400																												
He-backside cooling [mbar]	10																												
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Etch rate SiRN [nm·min ⁻¹]	25																												
19.	Stripping of polymers in NO₃ multi-purpose (#lith014)	CR116B / Wet-Bench 2 HNO ₃ (100 %) Selectipur: MERCK 100453 <ul style="list-style-type: none"> Beaker 1: HNO₃ (100 %) 20 min Quick Dump Rinse <0.1 μS Spin drying 	cleaning time: 20 min																										
20.	Etching of Cr Wet (#etch034)	CR116B / Wet-Bench 2 Chromium etch LSI Selectipur: MERCK 111547.2500 <ul style="list-style-type: none"> Quick Dump Rinse <0.1 μS Spin drying Etchrates: 100 nm·min ⁻¹	etch time: >30 min “Periodically rinse with DI to remove masking Cr-oxide.”																										
21.	Optical microscopic inspection (#char002)	CR112B / Nikon Microscope CR117B / Olympus Microscope CR102B / Olympus Microscope	“Determine the LPCVD SiRN layer thickness (half of the etch hole width+0.2 μm) required to seal the surface microchannels.”																										
22.	Cleaning Standard (#clean003)	CR112B / Wet-Bench 131 HNO ₃ (100 %) Selectipur: MERCK HNO ₃ (69 %) VLSI: MERCK <ul style="list-style-type: none"> Beaker 1: fuming HNO₃ (100 %), 5 min Beaker 2: fuming HNO₃ (100 %), 5 min Quick Dump Rinse <0.1 μS 	nr. dummies: ? “Dummies for ellipsometer measurement.”																										

23.	Etching HF (1%) Native Oxide (#etch027)	<ul style="list-style-type: none"> • Beaker 3: boiling (95 °C) HNO₃ (69 %), 10 min • Quick Dump Rinse <0.1 μS • Spin drying CR112B / Wet-Bench 3-3 HF (1 %) VLSI: MERCK 112629,500 <ul style="list-style-type: none"> • Etch time: >1 min • Quick Dump Rinse <0.1 μS • Spin drying 	
24.	LPCVD SiRN - low-deposition rate (#depo002)	CR125C / Tempress LPCVD new system 2007 program: SiRN01/N2 Tube: G3 <ul style="list-style-type: none"> • “Use 5-8 boat fillers in front and back of the boat to achieve specifications.” • SiH₂Cl₂ flow: 77.5 sccm • NH₃ flow: 20 sccm • temperature: 820/850/870 °C • pressure: 150 mTorr • deposition rate: ± 4 nm·min⁻¹ • Nr: ± 2.18 • Stress (range): 200-280 Mpa • Boat position 12: 200 MPa (centre of the boat) • Boat position 1: 280 MPa (front of the boat) • Uniformity/wafer: <2 % • Uniformity over the boat: (20 wafers): < 8 % 	nom. layer thickness: 1.3 μm program: SIRN01 nom. deposition rate: 3.62 nm·min⁻¹ nom. deposition time: 6 hrs <i>“Slowly(manually) move through the front heatzone.”, “Microchannels should be dry to avoid rapid evaporation of water.”</i>
25.	Ellipsometer Measurement (#char007)	CR118B / Plasmos Ellipsometer	<i>“Ellipsometer measurement on dummies.”</i>
26.	Optical microscopic inspection (#char002)	CR112B / Nikon Microscope CR117B / Olympus Microscope CR102B / Olympus Microscope	<i>“Rinse wafers in DI water and spin-dry before inspection. Look for the lack of water in the surface microchannels, confirming proper sealing of the microchannels.”</i>

A.2 Al/poly-Si⁺⁺ Thermopile Fabrication

Nr.	Process	Process Parameters	Design related parameters and remarks
1.	Cleaning Standard (#clean003)	CR112B / Wet-Bench 131 HNO ₃ (100 %) Selectipur: MERCK HNO ₃ (69 %) VLSI: MERCK <ul style="list-style-type: none"> • Beaker 1: fuming HNO₃ (100 %), 5 min • Beaker 2: fuming HNO₃ (100 %), 5 min • Quick Dump Rinse <0.1 μS • Beaker 3: boiling (95 °C) HNO₃ (69 %), 10 min • Quick Dump Rinse <0.1 μS • Spin drying 	
2.	Micro Balance Measurement (#char006)	CR125C / Satotius Micro Balance	<i>“Measure weight of wafers before LPCVD of poly-Si.”</i>
3.	LPCVD Poly Si - 590°C (#depo005)	CR125C / Tempress LPCVD Tube: F2 Program: senspoly <ul style="list-style-type: none"> • SiH₄ flow: 50 sccm • temperature: 590 °C • pressure: 250 mTorr • deposition rate: xx nm·min⁻¹ • Stress: 30 MPa 	layer thickness: 200 μm program: SENSPOLY nom. deposition rate: 3.33 nm·min⁻¹ nom. deposition time: 1 hr

4.	Micro Balance Measurement (#char006)	CR125C / Satotius Micro Balance	<i>“thickness [nm] = 26.5 · Δm [mg]”, “26.5 is the conversion factor for 4" wafers with LPCVD poly-Si.”</i>
5.	Solid Source Diffusion (SSD) of Boron at 1050°C (#dopi004)	CR112B / Furnace B1 Standby temperature: 700 °C • Program: SSD-1050 • Temp.: 1050 °C • Gas N ₂ : 3.80 SLM • Gas O ₂ : 0.20 SLM	doping time: 45 min <i>“Only use metal tweezers and fast unload/ load procedure.”, “Etch SSD dummies 5 min in BHF.”, “Old dummies require 15 min 800 °C DOX and 15 min BHF.”, “Old dummies can result in wafer curvature.”</i>
6.	Etching BHF (1:7) B₂O₅ (#etch026)	CR112B / Wet-Bench 3-3 NH ₄ F/HF (1:7) VLSI: MERCK 101171.2500 • Etch time: 5 min • Quick Dump Rinse <0.1 μS • Spin drying	etch time: 15 min
7.	Dry Oxidation at 800°C of Silicon (#depo026)	CR112B / Furnace B3 Standby temperature: 800 °C • Program: DOX-800 • Temp.: 800 °C • Gas: O₂ • Flow: 4 l·min⁻¹	oxidation time: 30 min
8.	Etching BHF (1:7) B₂O₅ (#etch026)	CR112B / Wet-Bench 3-3 NH ₄ F/HF (1:7) VLSI: MERCK 101171.2500 • Etch time: 5 min • Quick Dump Rinse <0.1 μS • Spin drying	etch time: 15 min
9.	Sheet Resistance measurement (#char009)	CR118B / Resistance Measurement Equipment	sheet resistance ±200 nm poly-Si ⁺⁺ : 453 Ω·□⁻¹ (1hr SSD), 815 Ω·□⁻¹ (45min SSD) <i>“R_{sheet} [Ω·□⁻¹] = R_{mas} · π / ln(2) = R_{mas} · 4.532”</i>
10.	Lithography - Priming (liquid) (#lith001)	CR112B / Suss Micro Tech Spinner (Delta 20) Hotplate 120 °C HexaMethylDiSilazane (HMDS) • Dehydration bake (120 °C): 5 min • Spin program: 4 (4000 rpm, 20 s)	
11.	Lithography - Coating Olin907-17 (#lith005)	CR112B / Suss Micro Tech Spinner (Delta 20) Hotplate 95 °C Olin 907-17 • Spin Program: 4 (4000 rpm, 20 s) • Prebake (95 °C): 90 s	
12.	Lithography - Alignment & Exposure Olin 907-17 (EV620) (#lith021)	CR117B / EV620 Electronic Vision Group 620 Mask Aligner • Hg-lamp: 12 mW·cm⁻² • Exposure Time: 4 s	mask: TCPY exposure time: 3.5 s mask: 5" 2.3 mm sub: 4" 0.53 mm contact: vacuum
13.	Lithography - Development Olin Resist (#lith011)	CR112B / Wet-Bench 11 Developer: OPD4262 Hotplate 120 °C (CR112B or CR117B) • After Exposure Bake (120 °C): 60 s Development: • Time: 30 s in Beaker 1 • Time: 15-30 s in Beaker 2 • Quick Dump Rinse <0.1 μS • Spin drying	
14.	Optical microscopic inspection - Lithography (#char001)	CR112B / Nikon Microscope	

15.	Plasma etching - chamber cleaning (Etske) (#etch003)	CR102A / Elektrotech PF310/340 Select chamber for desired etch process Select electrode for desired etch process • Electrode temp.: 10 °C or 25 °C • O ₂ flow: 20 sccm • pressure: 50 mTorr • power: 150 W • DC-Bias: 780 V <i>“Chamber is clean when plasma color is white.”</i>	<i>“Use quartz electrode.”</i>
16.	Plasma etching Silicon - Low etch-rate (Etske) (#etch002)	CR102A / Elektrotech PF310/340 Clean chamber Styros electrode Showerhead • Electrode temp.: 10 °C • SF ₆ flow: 30 sccm • CHF ₃ flow: 25 sccm • O ₂ flow: 20 sccm • pressure: 75 mTorr • DC-Bias: 0-30 V • 50 W : 300 nm·min ⁻¹ • 60 W : 450 nm·min ⁻¹ • 75 W : 530 nm·min ⁻¹ <i>“valid for a 3" wafer at 50 % loading”</i>	power: 50 W etch time: 1 min 10 s <i>“Topside and bottomside etching”, “Protect topside during bottomside etching with Oling907-17 resist.”, “Use quartz electrode.”</i>
17.	Stripping of Olin PR - standard (#lith016)	CR112B / Wet-Bench 3-2 HNO ₃ (100 %) Selectipur: MERCK 100453 • Time: 20 min • Quick Dump Rinse <0.1 μS • Spin drying • Visual microscopic inspection	
18.	Optical microscopic inspection (#char002)	CR112B / Nikon Microscope CR117B / Olympus Microscope CR102B / Olympus Microscope	
19.	Surface profile measurement (#char005)	CR118B / Veeco Dektak 8	<i>“Measure layer thickness after BHF¹ etching”, “Measure wafer curvature after SSD.”</i>
20.	Cleaning Short (#clean002)	CR112B / Wet-Bench 131 HNO ₃ (100 %) Selectipur: MERCK • Beaker 1: HNO ₃ (100 %) 5 min • Beaker 2: HNO ₃ (100 %) 5 min • Quick Dump Rinse <0.1 μS • Spin drying	
21.	Sputtering of Al (Oxford) (#depo022)	CR106A / Oxford PL 400 • Program: 3000 nm Al pos 1 • Pressure: 10 mTorr Depositionrate (100 mm wafer) : 820 nm·min ⁻¹	layer thickness: 200 nm power: 7 kW deposition time: 15 s
22.	Lithography - Priming (liquid) (#lith001)	CR112B / Suss Micro Tech Spinner (Delta 20) Hotplate 120 °C HexaMethylDiSilazane (HMDS) • Dehydration bake (120 °C): 5 min • Spin program: 4 (4000 rpm, 20 s)	
23.	Lithography - Coating Olin907-17 (#lith005)	CR112B / Suss Micro Tech Spinner (Delta 20) Hotplate 95 °C Olin 907-17 • Spin Program: 4 (4000 rpm, 20 s) • Prebake (95 °C): 90 s	
24.	Lithography - Alignment & Exposure Olin 907-17 (EV620) (#lith021)	CR117B / EV620 Electronic Vision Group 620 Mask Aligner • Hg-lamp: 12 mW·cm ⁻² • Exposure Time: 4 s	mask: TCAL exposure time: 3.5 s mask: 5" 2.3 mm sub: 4" 0.53 mm contact: vacuum
25.	Lithography - Development Olin Resist	CR112B / Wet-Bench 11 Developer: OPD4262 Hotplate 120 °C (CR112B or CR117B)	

	(#lith011)	<ul style="list-style-type: none"> • After Exposure Bake (120 °C): 60 s Development: <ul style="list-style-type: none"> • Time: 30 s in Beaker 1 • Time: 15-30 s in Beaker 2 • Quick Dump Rinse <0.1 μS • Spin drying 	
26.	Optical microscopic inspection - Lithography (#char001)	CR112B / Nikon Microscope	
27.	Etching of Al Wet (#etch035)	CR116B / Wet-Bench 2 Aluminium etch: MERCK 115435.2500 <ul style="list-style-type: none"> • Temp.: 55 °C • Quick Dump Rinse <0.1 μS • Spin drying Etchrates: 1 μm·min ⁻¹	temp: 40 °C etch time: 40 s <i>“Prepare to quickly stop the etching process in a beaker filled with DI water.”</i>
28.	Stripping of polymers in HNO ₃ multi-purpose (#lith014)	CR116B / Wet-Bench 2 HNO ₃ (100 %) Selectipur: MERCK 100453 <ul style="list-style-type: none"> • Beaker 1: HNO₃ (100 %) 20 min • Quick Dump Rinse <0.1 μS • Spin drying 	
29.	Optical microscopic inspection (#char002)	CR112B / Nikon Microscope CR117B / Olympus Microscope CR102B / Olympus Microscope	

A.3 Surface Microchannel SF₆ Plasma Release

Nr.	Process	Process Parameters	Design related parameters and remarks
1.	Lithography - Priming (liquid) (#lith001)	CR112B / Suss Micro Tech Spinner (Delta 20) Hotplate 120 °C HexaMethylDiSilazane (HMDS) <ul style="list-style-type: none"> • Dehydration bake (120 °C): 5 min • Spin program: 4 (4000 rpm, 20 s) 	
2.	Lithography - Coating Olin908-35 (#lith006)	CR112B / Suss Micro Tech Spinner (Delta 20) Hotplate 95 °C Olin 908-35 <ul style="list-style-type: none"> • Spin Program: 4 (4000 rpm, 20 s) • Prebake (95 °C): 120 s 	spin speed: 2000 rpm layer thickness: 5 μm <i>“Dry wafer edge beads for 1hr in yellow room.”</i>
3.	Lithography - Alignment & Exposure Olin 907-17 (EV620) (#lith021)	CR117B / EV620 Electronic Vision Group 620 Mask Aligner <ul style="list-style-type: none"> • Hg-lamp: 12 mW·cm⁻² • Exposure Time: 4 s 	mask: RLSE exposure time: 22 s contact: soft
4.	Lithography - Development Olin Resist (#lith011)	CR112B / Wet-Bench 11 Developer: OPD4262 Hotplate 120 °C (CR112B or CR117B) <ul style="list-style-type: none"> • After Exposure Bake (120 °C): 60 s Development: <ul style="list-style-type: none"> • Time: 30 s in Beaker 1 • Time: 15-30 s in Beaker 2 • Quick Dump Rinse <0.1 μS • Spin drying 	bkr1 develop time: 1 min bkr2 develop time: 30 s <i>“No after exposure bake, avoiding cracking of the resist layer.”</i>
5.	Optical microscopic inspection - Lithography (#char001)	CR112B / Nikon Microscope	
6.	Lithography - Postbake standard (#lith009)	CR112B / Hotplate 120°C <ul style="list-style-type: none"> • Time: 30 min 	

7.	Plasma etching - chamber cleaning (Etske) (#etch003)	CR102A / Elektrotech PF310/340 Select chamber for desired etch process Select electrode for desired etch process <ul style="list-style-type: none"> • Electrode temp.: 10 °C or 25 °C • O₂ flow: 20 sccm • pressure: 50 mTorr • power: 150 W • DC-Bias: 780 V “Chamber is clean when plasma color is white.”	“Use quartz electrode.”																		
8.	Plasma etching SiN (Etske) (#etch004)	CR102A / Elektrotech PF310/340 Dirty chamber Styros electrode <ul style="list-style-type: none"> • Electrode temp.: 10 °C • CHF₃ flow: 25 sccm • O₂ flow: 5 sccm • pressure: 10 mTorr • power: 75 W Etchrate SiN : 50 nm·min ⁻¹ (for V _{DC} =-460 V) Etchrate SiN : 75 nm·min ⁻¹ (for V _{DC} = -580 V) Etchrate Olin resist : 95 nm·min ⁻¹ “If DC-Bias < 375 V apply chamber clean (#etch003).”	nom. etching rate: 60 nm·min ⁻¹ layer thickness: 1.8 μm nom. etch time: 33 min “Use quartz electrode.”																		
9.	Plasma etching Silicon SCT release (Etske) (#etch076)	CR 125c/Adixen AMS 100 SE Applications: release channels (SCT) <table border="1" data-bbox="452 697 821 939"> <thead> <tr> <th>Parameters</th> <th>Value</th> </tr> </thead> <tbody> <tr> <td>SF₆ [sccm]</td> <td>350</td> </tr> <tr> <td>He-backside cooling [mbar]</td> <td>10</td> </tr> <tr> <td>ICP [W]</td> <td>1800</td> </tr> <tr> <td>CCP On/Off [ms]</td> <td>0</td> </tr> <tr> <td>p [Pa]</td> <td>10</td> </tr> <tr> <td>APC position %</td> <td>15</td> </tr> <tr> <td>Electrode temperature [°C]</td> <td>0</td> </tr> <tr> <td>Substrate height [mm]</td> <td>200</td> </tr> </tbody> </table>	Parameters	Value	SF ₆ [sccm]	350	He-backside cooling [mbar]	10	ICP [W]	1800	CCP On/Off [ms]	0	p [Pa]	10	APC position %	15	Electrode temperature [°C]	0	Substrate height [mm]	200	nom. etch rate: 11 μm·min ⁻¹ etch time: 20 min nom. etch depth: 220 μm
Parameters	Value																				
SF ₆ [sccm]	350																				
He-backside cooling [mbar]	10																				
ICP [W]	1800																				
CCP On/Off [ms]	0																				
p [Pa]	10																				
APC position %	15																				
Electrode temperature [°C]	0																				
Substrate height [mm]	200																				
10.	Plasma etching SiN (Etske) (#etch004)	CR102A / Elektrotech PF310/340 Dirty chamber Styros electrode <ul style="list-style-type: none"> • Electrode temp.: 10 °C • CHF₃ flow: 25 sccm • O₂ flow: 5 sccm • pressure: 10 mTorr • power: 75 W Etchrate SiN: 50 nm·min ⁻¹ (for V _{DC} =-460 V) Etchrate SiN: 75 nm·min ⁻¹ (for V _{DC} = -580 V) Etchrate Olin resist: 95 nm·min ⁻¹ “If DC-Bias < 375 V apply chamber clean (#etch003).”	nom. etching rate: 60 nm·min ⁻¹ nom. etch time: 7 min “Use quartz electrode.”, “Open inlet holes to the surface microchannels.”																		
11.	Stripping of polymers in HNO₃ multi-purpose (Etske) (#lith014)	CR116B / Wet-Bench 2 HNO ₃ (100 %) Selectipur: MERCK 100453 <ul style="list-style-type: none"> • Beaker 1: HNO₃ (100 %) 20 min • Quick Dump Rinse <0.1 μS • Spin drying 																			
12.	Optical microscopic inspection (Etske) (#char002)	CR112B / Nikon Microscope CR117B / Olympus Microscope CR102B / Olympus Microscope																			

Summary

The emerging fields of micro total-analysis systems (micro-TAS), micro-reactors and bio-MEMS drives the need for further miniaturisation of sensors measuring quantities such as pressure, temperature and flow. The research described in this thesis concerns the development of low-drift micro flow sensors for accurate measurement of minute amount of liquid flow in the $\text{nl}\cdot\text{min}^{-1}$ range.

Miniaturisation means that flow channel dimensions and flow rates become smaller. This requires thermally-isolated flow channels, where the complete fluid can be heated in order to obtain maximum sensitivity. A microchannel fabrication concept (Chap. 3) was developed, based on buried channel technology (BCT), allowing for easy fluidic interfacing and integration of transducer materials in close proximity to the fluid. This is achieved by the reliable fabrication of completely sealed micro-channels directly below the substrate surface. The channel technology has found application in the fabrication of resonant flow sensors and low-drift micro flow sensors (Chap. 4-8) in the $\text{nl}\cdot\text{min}^{-1}$ range. Additionally, the technology has been extended by the possibility to integrate nanochannels using fluidic vias. This has been used in the fabrication of nano-nozzle electrospray emitters (Chap. 9).

In current micromachined thermal flow sensors the elements for temperature sensing are made by thin films. The problem is that thin films reproduce poorly and that practically all materials properties are subject to drift. This translates directly into the accuracy of thermal micro flow sensors. In this thesis low-drift micro flow sensors were investigated, using two heaters and a thermopile in order to eliminate material drift (Chap. 5). The low offset drift of thermopiles has been exploited in a feedback loop controlling the dissipated powers in the heater resistors, minimising inevitable influences of resistance drift, mismatch of thin-film metal resistors and thermopile material drift. The control system cancels the flow-induced temperature difference across the thermopile by controlling a power difference between both heater resistors, thereby giving a measure of the flow rate. Alternatively, a sensor resistor and heat waves can be used to provide for a low offset-drift error signal (Chap. 8). It was demonstrated that material drift can largely be compensated.

Sensitivity can be increased by designing flow sensors with a large number of integrated thermocouple junctions (Chap. 6), however it was observed that externally applied temperature gradients over the chip can still lead to drift of the sensor output signal. A special meandering microchannel layout was used to create a fully symmetrical flow sensor, where the arrangement of thermopile junctions has resulted in low-drift micro thermal flow sensors for liquids in the $\text{nl}\cdot\text{min}^{-1}$ range, with compensation for external temperature gradients (Chap. 7).

De opkomst van micro-total-analysis-systems (micro-TAS), micro-reactoren en bio-MEMS genereert de behoefte tot verdere miniaturisering van sensoren voor grootheden als druk, temperatuur en stromingssnelheid. Het onderzoek dat in dit proefschrift wordt beschreven, betreft de ontwikkeling van lage-drift micro-stromingssnelheidssensoren voor het nauwkeurig meten van minuscule vloeistofstroomsnelheden in het $\text{nl}\cdot\text{min}^{-1}$ bereik.

Miniaturisering betekent dat de afmetingen van de vloeistofkanalen en de daarin optredende stroomsnelheden afnemen. Hierdoor wordt het noodzakelijk om de kanalen thermisch te isoleren, om zo de volledige vloeistof in het kanaal te kunnen verwarmen, waarbij een maximale gevoeligheid kan worden verkregen. Hiertoe is een fabricageconcept ontwikkeld voor microkanalen (Hfst. 3), gebaseerd op buried-channel-technologie (BCI). Deze kanalen zijn eenvoudig fluïdisch aan te sluiten, waarbij transducenten geïntegreerd kunnen worden dicht bij de vloeistof. Dit wordt bereikt door fabricage van volledig gesloten microkanalen direct onder het oppervlak van het substraat. Deze technologie vindt toepassing in resonerende stromingssnelheidssensoren en lage-drift, thermische stromingssnelheidssensoren. (Hfst. 4-8). Daarbij is de technologie verder uitgebreid met de mogelijkheid om nanokanalen via fluïdische verbindingen te integreren met microkanalen. Dit is gebruikt bij de fabricage van nano-nozzle-electrospray-emitters (Hfst. 9).

In huidige micro-stromingssnelheidssensoren bestaan de elementen voor het meten van de temperatuur uit dunne films. Het probleem is dat bijna alle dunne films moeilijk reproduceerbaar zijn en dat de materiaaleigenschappen onderhevig zijn aan drift, hetgeen de nauwkeurigheid van de sensoren beïnvloedt. In dit proefschrift worden lage-driftsensoren, bestaande uit twee verwarmingselementen en een thermozuil, onderzocht om de invloed van drift in materiaaleigenschappen te elimineren (Hfst. 5). De lage offsetdrift van thermozuilen wordt gebruikt om de invloed van weerstandsdrift en ongelijkheid van weerstandswaarden en drift in de thermozuilmaterialen te minimaliseren d.m.v. een terugkoppellus, waarbij het gedissipeerde vermogen in de verwarmingselementen wordt bijgesteld. Het regelsysteem regelt het temperatuurverschil t.g.v. de vloeistofstroming naar nul, door het verschil in gedissipeerde vermogen van de verwarmingsweerstand aan te passen. Het benodigde vermogensverschil is daarbij een maat voor de stromingssnelheid. Een andere methode maakt gebruik van warmtegolven om een lage-drift foutsignaal te verkrijgen (Hfst. 8). Het is aangetoond dat materiaaldrift hiermee voor een groot deel wordt gecompenseerd.

Een grotere gevoeligheid kan verkregen worden door een groot aantal thermokoppeljuncties te gebruiken (Hfst. 6). Echter, extern aangelegde temperatuurverschillen over de chip kunnen nog steeds leiden tot drift van het uitgangssignaal. Een speciale, meanderende kanaalstructuur kan gebruikt worden om een volledig symmetrische sensor te verkrijgen, resulterende in lage-drift micro-thermische-stromingsnelheidssensoren voor vloeistofstroomsnelheden in het $\text{nl}\cdot\text{min}^{-1}$ bereik, met compensatie voor extern aangelegde temperatuurverschillen (Hfst. 7).

List of Publications

Journal Articles

In preparation and submitted:

- M. Dijkstra, T.S.J. Lammerink, M.J. de Boer, J.W. Berenschot, R.J. Wiegink, M.C. Elwenspoek, “Low-drift flow sensor with thermopile-based power feedback”, *J. Micromech. S.*, (accepted for publication).
- M. Dijkstra, N.R. Tas, J.W. Berenschot, T. Hankemeier, M.J. de Boer, T.S.J. Lammerink, R.J. Wiegink, M. Elwenspoek, “Nano-nozzle electrospray emitters fabricated by a micro- to nano-fluidic via technology”, *Lab Chip*, (to be submitted).
- M. Dijkstra, T.S.J. Lammerink, M.J. de Boer, R.J. Wiegink, M. Elwenspoek, “Ambient temperature-gradient compensated low-drift thermopile flow sensor”, *Sensor. Actuat. A-Phys.*, (in preparation).
- M. Dijkstra, T.S.J. Lammerink, O. Pjetri, M.J. de Boer, J.W. Berenschot, R.J. Wiegink, M. Elwenspoek, “AC-driven temperature-balancing flow sensor with power feedback”, *J. Micromech. Microeng.*, (in preparation).

Published:

- M. Dijkstra, M.J. de Boer, J.W. Berenschot, T.S.J. Lammerink, R.J. Wiegink, M. Elwenspoek, “Miniaturized thermal flow sensor with planar integrated sensor structures on semicircular surface channels”, *Sensor. Actuat. A-Phys.*, **143** (2008) 1-6.
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Conference Contributions

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- M. Dijkstra, T.S.J. Lammerink, O. Pjetri, M.J. de Boer, J.W. Berenschot, R.J. Wiegink, M. Elwenspoek, “AC-driven temperature-balancing flow sensor with zero-offset power feedback”, *Proc. MME*, (2008).

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Na vier jaar onderzoek is het dan gelukt dit proefschrift uit, grotendeels, publicaties samen te stellen. Vaak was de ene sensor net af of er waren voldoende nieuwe ideeën om een nog betere sensor te maken. Theo, bedankt dat wij het, soms met de nodige stevige discussies, zo ver hebben weten te brengen. Jammer, dat resultaten soms belangrijker leken, dan het bieden van ondersteuning bij het verrichten van mijn onderzoek. Tijdens het schrijven van mijn proefschrift heeft het willen doorgronden van driftinvloeden mij daarom nog de nodige hoofdbreken gekost. Nu kan ik dit voldoende begrijpen. Remco W., bedankt voor je bijdrage aan mijn onderzoek en dat je mij de laatste maanden hebt ondersteund in het voltooiën van dit proefschrift.

Ik herinner mij de colleges electrostatica en -dynamica van Miko. De sturende manier van onderwijs en tegelijk ongedwongen stimulans om de materie jezelf eigen te maken, spraken mij zeer aan. Iets wat later als onderwijsassistent, samen met Leon en Miko, binnen probleemgestuurd onderwijs niet zo makkelijk is om zelf toe te passen. De studenten de collegestof bij te brengen, door met opgaven te helpen, heb ik, onder andere met Johan, Wabe en Iris, met toenemend plezier gedaan.

Gijs, bedankt dat ik na mijn afstudeeropdracht een tijdje als medewerker-onderzoek door heb kunnen werken aan de uitlezing van kunstmatige krekelharen. Dit was natuurlijk niet mogelijk geweest zonder de haren van Rik, die deze met toewijding uit de cleanroom heeft getrokken. Bedankt John voor de samenwerking en dankzij jou kan ik nu ook een beetje jongleren. Daarnaast was er altijd wel een leuk gesprek met Toon en Saravanan als we in de mensa aten. Voor de nodige intellectuele ontspanning werden er af en toe met Joost algemene relativiteitspuzzels opgelost. “De covariante afgeleide van een contravariant tensorveld gegeven een affiene connectie” te ontrafelen is toch iets voor de meer gevorderde puzzelaar.

Meint en Erwin zijn wat mij betreft de beste werkpaarden van stal. Het was snel duidelijk dat Meint de kar eerder had getrokken en binnen afzienbare tijd hebben we een microkanaal-technologie neergezet, die mij de mogelijkheid gaf om de nodige sensoren te realiseren. Gaandeweg heb ik veel kunnen leren van Erwin z'n inzicht in cleanroom-technologie, al bleef het af en toe flink zweten in de cleanroom. In het algemeen verliep de fabricage van de sensoren echter zonder problemen, mede dankzij de MESA+ cleanroom-medewerkers Eddy, Hans, Huib, Ite-Jan, Marion, Rene en Samantha.

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der te komen. Olti, je inzet en goede samenwerking in het verrichten van de metingen voor Hfst. 8 worden zeer gewaardeerd. De posterprijs op de MME 2008-conferentie in Aachen was en mooie beloning voor je werk.

Ook andere conferenties waren voor mij de moeite waard. Daarbij wil ik Edin graag bedanken als onze gids in Tokyo. Ik heb nooit geweten dat Japanners zo goed kunnen dansen. Johan en zijn vrienden van IBM Zürich wil ik daarnaast bedanken voor het adhoc regelen van een leuke rondreis in Griekenland na de MNE 2009-conferentie in Athene. Alle overige mensen van de TST-afdeling, waarmee ik samenwerkt heb, Ahmad, Arjan, Bas, Berker, Boudewijn, Chris, Dannis, Deladi, Dick, Doekle, Duy, Hadi, Hammad, Hanh, Harmen, Henk, Henry, Hien, Imran, Ingrid, Jeroen, Judith, Kees, Laura, Luis, Marcel K., Marcus, Maryana, Mink, Mubassira, Narges, Nataliya, Paul, Ram, Roald, Robert, Sandeep, Satya, Shahina, Susan, Vitaly, Wilco, Willem, Winfred, Yiping et al., wil ik bedanken voor de prettige werksfeer, de vele slappe bakken tijdens de koffie, de leuke borrels en uitjes en voor het feit dat uiteindelijk iedereen z'n eigen gang kan gaan.

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Van stekkerdozen en schakelborden werd ik op jonge leeftijd voorzien door opa Dijkstra, die een bruin/witgoed- en installatiezaak in Loppersum had. Daarnaast waren er de inspirerende verhalen over mijn grootvader Geert de Vries, die op jonge leeftijd een constructiebedrijf begon. Op de rand van een oude krant werden ideeën uitgewerkt en indien nodig laste hij mee als een grote klus af moest. Ik bedank oma de Vries dat altijd alles kon, ook als sommige projecten van mij enigzins uit de hand dreigden te lopen. Tenslotte wil ik mijn paps & moes bedanken voor hun liefdevolle steun in de afgelopen jaren.

Marcel Dijkstra
 Enschede, 23 maart 2009



Biography

Marcel Dijkstra was born in Groningen, The Netherlands on the 28th of april 1978. He went through primary school and MAVO in Loppersum, The Netherlands till 1994. Subsequently, the HAVO/MBO diploma was obtained in 1996 at the Eemsmondcollege in Delfzijl, The Netherlands. In the same year he started his study to become an electrical engineer at the Hanzehogeschool in Groningen, The Netherlands. The first internship was followed at Philips Semiconductors in Nijmegen, The Netherlands at the Quality & Reliability department. Electromigration measurements on aluminium interconnects and degradation measurements on transistors were conducted. Additionally, work was continued on statistical analysis software for the processing of reliability data. A second internship was conducted with Albert Martena at the University Medical Centre Groningen, The Netherlands. Work was done on measurement software for the determination of motor abilities in patients with Parkinson's disease. Activities included the implementation of fast-fourier-transform algorithms in assembler and the implementation of wavelet-transform algorithms. The final bachelor assignment dealt with the development of a radio-controlled robot car using a prototype Bluetooth chip and was conducted together with Albert Martena at the university of Karlskrona/Ronneby, Sweden. Their examination work was awarded the Telecomcity - Sigma Exallon Systems prize in the province of Blekinge, Sweden in 2000. Work was continued for a few months on a prototype Bluetooth extension microphone for a conference telephone from Konftel AB in Umeå, Sweden. Study was continued at the University of Twente, where he obtained the M.Sc. degree in electrical engineering in 2004. The masters project was conducted at the Transducers Science & Technology group and dealt with the development of artificial sensory hairs based on crickets hairs. He worked on that project as temporary research associate until 2006. He then started as a Ph.D. student on the research described in this thesis.

Biografie

Marcel Dijkstra werd geboren op 28 april 1978 te Groningen. Hij doorliep de lagere school en MAVO in Loppersum tot 1994. Daarna werd het HAVO/MBO diploma gehaald in 1996 aan het Eemsmondcollege in Delfzijl. In hetzelfde jaar begon hij zijn studie tot electrotechnisch ingenieur aan de Hanzehogeschool te Groningen. De eerste stage werd gelopen bij Philips Semiconductors in Nijmegen op de Quality & Reliability afdeling. Daarbij werden electromigratiemetingen aan aluminium interconnects en degradatiemetingen aan transistoren uitgevoerd. Daarnaast werd er verder gewerkt aan statistische analyse software voor het verwerken van betrouwbaarheidsgegevens. Een tweede stage werd samen met Albert Martena aan het Universitair Medisch Centrum Groningen uitgevoerd. Er werd gewerkt aan meetsoftware voor het bepalen van motorische vaardigheden van patiënten met de ziekte van Parkinson. De implementatie van fast-fourier-transformatie algoritmes in assembler en de implementatie van wavelet-transformatie algoritmes behoorden tot de bezigheden. De afstudeeropdracht betrof de ontwikkeling van een radiografisch bestuurbare robotauto gebruikmakende van een prototype Bluetooth chip en werd samen met Albert Martena uitgevoerd aan de universiteit van Karlskrona/Ronneby, Zweden. Hun afstudeerwerk werd beloond in 2000 met de Telecomcity – Sigma Exallon Systems prijs in de provincie Blekinge, Zweden. Enkele maanden werd het werk voortgezet in de realisatie van een prototype Bluetooth extensie microfoon voor een conferentie telefoon van Konftel AB in Umeå, Zweden. De studie werd opgepakt aan de Universiteit Twente, waar hij de M.Sc. titel in de electrotechniek behaalde in 2004. De afstudeeropdracht werd uitgevoerd binnen de Transducers Science & Technology groep en betrof de ontwikkeling van kunstmatige haarsensoren gebaseerd op krekels haren. Hij werkte binnen dit project als medewerker-onderzoek tot 2006. Daarna begon hij als beoogd promovendus aan het onderzoek beschreven in dit proefschrift.

